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**NUMERICAL MODELING OF THE INTERNAL BALLISTICS OF A GAS
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**NUMERICAL MODELING OF THE
INTERNAL BALLISTICS
OF A GAS GUN**

A Dissertation

**Submitted to the Graduate Faculty of the
Louisiana State University and
Agricultural and Mechanical College
in partial fulfillment of the
requirements for the degree of
Doctor of Philosophy**

in

The Department of Mechanical Engineering

by

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August, 1982**

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TABLE OF CONTENTS

	Page
ACKNOWLEDGEMENT - - - - -	ii
LIST OF FIGURES - - - - -	v
NOMENCLATURE - - - - -	vii
ABSTRACT - - - - -	ix
 CHAPTER	
I INTRODUCTION - - - - -	1
II MATHEMATICAL MODEL OF INTERNAL CHAMBER FLOWS - - - - -	4
A. Discharge Chamber Flow - - - - -	4
1. Governing Equations - - - - -	4
2. Physical and Computational Flow Spaces - - - - -	7
3. Transformed Governing Equations - - -	8
4. Numerical Method - - - - -	10
4.1 Interior Grid Points - - - - -	11
4.2 Boundary Grid Points - - - - -	15
4.2.1. Center Line Grid Points - - -	15
4.2.2. Wall Grid Points - - - - -	16
4.2.3. Moving Wall Grid Points - - -	18
4.2.4. Exit Grid Points - - - - -	19
5. Results - - - - -	20
B. Barrel Flow Chamber - - - - -	40
1. Governing Equations - - - - -	40
2. Numerical Method - - - - -	41
2.1 Characteristic and Compatibility Equations - - - - -	42

2.2 Numerical Integration Scheme - -	42
2.2.1. Interior Grid Point - -	47
2.2.2. Outflow Boundary Point - -	49
2.2.3. Inflow Boundary Point -	51
2.2.4. Projectile Grid Point - -	52
2.2.5. Shock Wave Grid Point - -	54
3. Results - - - - -	57
C. Main Chamber Flow - - - - -	62
1. Physical and Computational Flow Spaces - - - - -	62
2. Transformed Governing Equations - -	65
3. Numerical Method - - - - -	65
4. Results - - - - -	69
REFERENCES - - - - -	76
BIBLIOGRAPHY - - - - -	78
APPENDICES - - - - -	
A. APPLICATION OF THE METHOD OF CHARACTER- ISTICS FOR UNSTEADY ONE-DIMENSIONAL FLOW -	79
B. CONSTANT ETA REFERENCE PLANE CHARACTER- ISTIC RELATIONS - - - - -	86
C. CONSTANT ZETA REFERENCE PLANE CHARAC- TERISTIC RELATIONS - - - - -	92
D. FORTRAN IV LISTING OF THE PROGRAM FOR THE DISCHARGE CHAMBER FLOW - - - - -	95
E. FORTRAN IV LISTING OF THE PROGRAM FOR THE MAIN AND BARREL CHAMBER FLOW - - - -	133
VITA - - - - -	187

LIST OF FIGURES

Figure	Page
1 Schematic illustration of a gas gun - - - - -	6
2 Schematic illustration of the discharge chamber -	7
3 Physical and computational domains - - - - -	12
4 Grid generated in the discharge chamber - - - -	22
5 Grid generated in the discharge chamber use in Pakarat's experiment - - - - -	24
6 Discharge chamber in the case where the wall has n different curvatures - - - - -	25
7 Velocity vector plots of the flow in the discharge chamber - - - - -	28
8 Velocity vector plots of the flow when smoothing devices are used - - - - -	30
9 Velocity vector plots of the flow when the method of characteristics is not used at the boundaries - - - - -	31
10 Discharge pressure vs time, charge pressure = 986.KPA - - - - -	32
11 Discharge chamber geometry, and midplane pressure and Mach number for inviscid flow, charge pressure = 860.KPA - - - - -	33
12 Discharge chamber geometry, and midplane pressure and Mach number for inviscid flow, charge pressure = 171.KPA - - - - -	34
13 Pressure (top) and Mach number contours for inviscid flow - - - - -	35
14 Velocity vector plots of the flow in the discharge chamber used in Pakarat's experiment - - - -	36
15 Discharge pressure vs time, charge pressure = 2449.KPA - - - - -	37
16 Discharge chamber geometry, and midplane pressure and Mach number for inviscid flow, charge pressure = 2449.KPA - - - - -	38

17	Pressure (top) and Mach number contours for inviscid flow - - - - -	39
18	Physical and computational flow planes - - -	44
19	Physical interpretation of the CFL stability criterion - - - - -	46
20	Finite difference grid for an outflow boundary	50
21	Finite difference grid for a subsonic inflow -	51
22	Finite difference grid for a moving projectile	53
23	Finite difference grid for a shock wave point -	56
24	Projectile base pressure, velocity, and displacement as a function of time, charge pressure = $8 \cdot 10^3$ KPA - - - - -	58
25	Projectile base pressure, Velocity, and displacement as a function of time, charge pressure = $5 \cdot 10^3$ KPA - - - - -	59
26	Projectile base pressure, velocity, and displacement as a function of time, charge pressure = $50 \cdot 10^3$ KPA - - - - -	60
27	Shock position and velocity as a function of time, charge pressure = $50 \cdot 10^3$ KPA - - - - -	61
28	Schematic illustration of the main chamber - -	63
29	Velocity distribution in the main chamber, time = $0.02(\text{MS})$ - - - - -	70
30	Velocity distribution in the main chamber, time = $0.44(\text{MS})$ - - - - -	71
31	Velocity distribution in the main chamber, time = $0.60(\text{MS})$ - - - - -	72
32	Velocity distribution in the main chamber, time = $0.72(\text{MS})$ - - - - -	73
33	Velocity distribution in the main chamber, time = $0.83(\text{MS})$ - - - - -	74
34	Velocity distribution in the main chamber, time = $0.93(\text{MS})$ - - - - -	75
A-1	Relationship between the vectors W_i and N , and the characteristic curve - - - - -	81

NOMENCLATURE

u	axial velocity (M/S)
v	radial velocity (M/S)
P	Pressure (PA)
p	density (KG/M ³)
T	Temperature (°K)
x	axial coordinate (M)
y	radial coordinate (M)
t	time (S)
a	speed of sound (M/S)
γ	specific heat ratio
R	gas constant (J/MOLE.°K)
μ	first coefficient of viscosity (PA · S)
λ	second coefficient of viscosity (PA · S)
(ξ, η, τ)	coordinates in the computational space
M _p	piston mass (KG)
X _p	piston displacement (M)
A _p	piston area (M ²)
D	friction force (N)
Y _w (x)	wall shape
P _c	charge pressure (PA)
R _i , R _e	inlet radius, outlet radius (in the discharge chamber)
σ	parameter defined in the grid generation of the discharge chamber
W	vector defined in the method of characteristic (Appendix A)

Λ , F and Ψ

scalars defined in the barrel
chamber flow

QROT, QUT, QVT, and QPT

scalars defined in the main
chamber flow

ABSTRACT

A numerical procedure is applied to three principal flow models of a gas gun, the discharge chamber, the main chamber, and the barrel chamber. In the first two models, an explicit method, the second order MacCormack scheme, is used to solve the Navier-Stokes equations for two-dimensional (axisymmetric), time dependent, compressible flow. The fluid is assumed to be a perfect gas. The flow boundaries may be arbitrary curved and time dependent. Transformations have been used to map the physical space into a computational space with uniform grid spacing. A grid generation surface has been achieved for various wall shapes. The third model investigates a detailed one-dimensional time dependent flow. In this case, the method of characteristics is used to solve the quasi-linear non-homogeneous partial differential equations of the first order. The numerical results obtained from the present solution are compared with the one-dimensional solution. The present technique shows better correlation with the experimental data.

CHAPTER I

INTRODUCTION

The two fields of compressible flow theory and numerical solution of partial differential equations were combined more than thirty years ago when Richtmyer reported a study of one dimensional waves (Ref. [1]). Since that time, the hybrid field of computational gas dynamics has undergone much growth. At the present time methods are available for the analysis of three-dimensional, unsteady, compressible, viscous flows involving complicated geometries, both internal and external. Still, the application of the numerical methods to various problems is somewhat of an art, and each solution achieved is evidence of ingenuity and perception on the part of the researcher. In other words, we are not yet to the point where an engineer can just push a button on an all-encompassing fluid dynamics computer program and expect a unique and correct result.

The field of ballistics is particularly amenable to the methods of computational gasdynamics. Several years ago the author performed a one dimensional analysis of the internal ballistics of a dual piston gas gun (Ref. [2]). That work involved reducing the gun equations to a system of ordinary differential equations which could

be solved simultaneously for the gun parameters. The results of the analysis were very encouraging, but they provided no details of the complicated gasdynamic processes at work within the gun chambers. In the present study a two-dimensional (axisymmetric) study of the transient behavior of the compressible viscous gases within the gun chamber is performed. However, the main purpose of the study is not to determine the flow characteristics for a specific gun, but rather to develop and demonstrate methods for dealing with internal chamber flows which involve time dependent boundaries, variable inlet streams and various wall shapes.

The studies in this work have been divided into three principal flow models. In the first model we have used a finite difference method to solve the governing partial differential equations for a discharge chamber flow. The method of characteristics is used to solve for the flow properties at the boundaries. Transformations have been used to map the physical space into a computational space with uniform grid spacing. Some of the walls are vertical. In order to eliminate the singularity special functions have been used to fit the shape of the wall. A complete surface generation has been achieved. The second flow model, representative of a main piston chamber flow, incorporates a two-dimensional, time-dependent flow for which the boundaries are also time dependent. In this model a

surface generation technique with a grid which is both spatially variable and time-dependent is required to handle the computations in the variable-inflow regions and at the moving boundaries. The third model investigates a detailed one-dimensional time dependent flow representative of a barrel flow. The method of characteristics, which is a very accurate method for solving hyperbolic partial differential equations, is used. This numerical technique presents another advantage in the treatment of the shocks. The shock wave is treated as a moving boundary separating two regions of continuous flow. The method of characteristics combined with the Rankine-Hugoniot equations are combined to solve for the flow properties.

Throughout the analysis it has been necessary to utilize certain devices which help to ensure the accuracy and stability of the numerical methods. An explicit artificial viscosity has been used to stabilize regions of large pressure rise such as shock waves. In regions of rapid expansion resulting in a change of sign of velocity components a velocity averaging technique has been employed. Finally, grid generation procedures are employed to improve the accuracy of the calculations.

CHAPTER II

MATHEMATICAL MODEL OF INTERNAL CHAMBER FLOWS

The analysis of gas guns encompasses a variety of internal flow problems. A typical arrangement consists of three parts; namely, the discharge chamber, the main chamber, and the barrel chamber. A schematic representation is shown in Figure 1. The gas flows through the discharge chamber and the main chamber are treated as two-dimensional, time dependent flows. The flow within the barrel chamber is treated as one dimensional and time dependent.

A. DISCHARGE CHAMBER FLOW

1. Governing Equations

In the absence of body force, the two-dimensional equations for a viscous, non heat conducting flow in cylindrical coordinates with axial symmetry and zero azimuthal velocity may be written as follows:

$$\frac{\partial \rho}{\partial t} + u \frac{\partial \rho}{\partial x} + v \frac{\partial \rho}{\partial y} + \rho \left(\frac{\partial u}{\partial x} + \frac{\partial v}{\partial y} \right) + \rho \frac{v}{y} = 0 \quad (1)$$

$$\begin{aligned} \frac{\partial u}{\partial t} + u \frac{\partial u}{\partial x} + v \frac{\partial u}{\partial y} + \frac{1}{\rho} \frac{\partial p}{\partial x} &= \mu \left[\frac{\partial^2 u}{\partial y^2} + \frac{4}{3} \frac{\partial^2 u}{\partial x^2} + \frac{1}{3} \frac{\partial^2 v}{\partial y \partial x} \right] \\ + \frac{\mu}{\rho y} \left[\frac{1}{3} \frac{\partial v}{\partial x} + \frac{\partial u}{\partial y} \right] \end{aligned} \quad (2)$$

$$\frac{\partial v}{\partial t} + u \frac{\partial v}{\partial x} + v \frac{\partial v}{\partial y} + \frac{1}{\rho} \frac{\partial P}{\partial y} = \mu \left[\frac{\partial^2 v}{\partial x^2} + \frac{4}{3} \frac{\partial^2 v}{\partial y^2} + \frac{1}{3} \frac{\partial^2 u}{\partial y \partial x} \right] \\ + \frac{4}{3} \frac{\mu}{\rho} \left[\frac{\partial v}{\partial y} - \frac{v}{y} \right] \quad (3)$$

$$\frac{\partial P}{\partial t} + u \frac{\partial P}{\partial x} + v \frac{\partial P}{\partial y} - a^2 \left(\frac{\partial \rho}{\partial t} + u \frac{\partial \rho}{\partial x} + v \frac{\partial \rho}{\partial y} \right) \\ = (\gamma - 1) \mu \left\{ \frac{4}{3} \left[\left(\frac{\partial u}{\partial x} \right)^2 + \left(\frac{\partial v}{\partial y} \right)^2 \right] + \left[\left(\frac{\partial u}{\partial y} \right)^2 + \left(\frac{\partial v}{\partial x} \right)^2 \right] \right. \\ \left. + \left[2 \frac{\partial u}{\partial y} \frac{\partial v}{\partial x} - \frac{4}{3} \frac{\partial u}{\partial x} \frac{\partial v}{\partial y} \right] + \left[\frac{4}{3} \frac{v^2}{y^2} - \frac{4}{3} \frac{v}{y} \left(\frac{\partial v}{\partial y} + \frac{\partial u}{\partial x} \right) \right] \right\} \quad (4)$$

$$P = \rho R T, \quad a^2 = \gamma P / \rho \quad (5)$$

where ρ is the density, u is the velocity in the axial direction, v is the velocity in the radial direction, P is the pressure, t is the time, a is the speed of sound, R is the engineering gas constant, γ is the ratio of the specific heats, y is the radial distance from the axis of symmetry, and μ is the first coefficient of viscosity. The bulk viscosity in the viscous terms is taken to be zero, that is, $\lambda = -\frac{2}{3}\mu$, where λ is the second coefficient of viscosity. The system of equations (1) - (5) is closed, in the sense that there are as many equations as unknowns. The basic unknowns are u , v , ρ , P , and T .

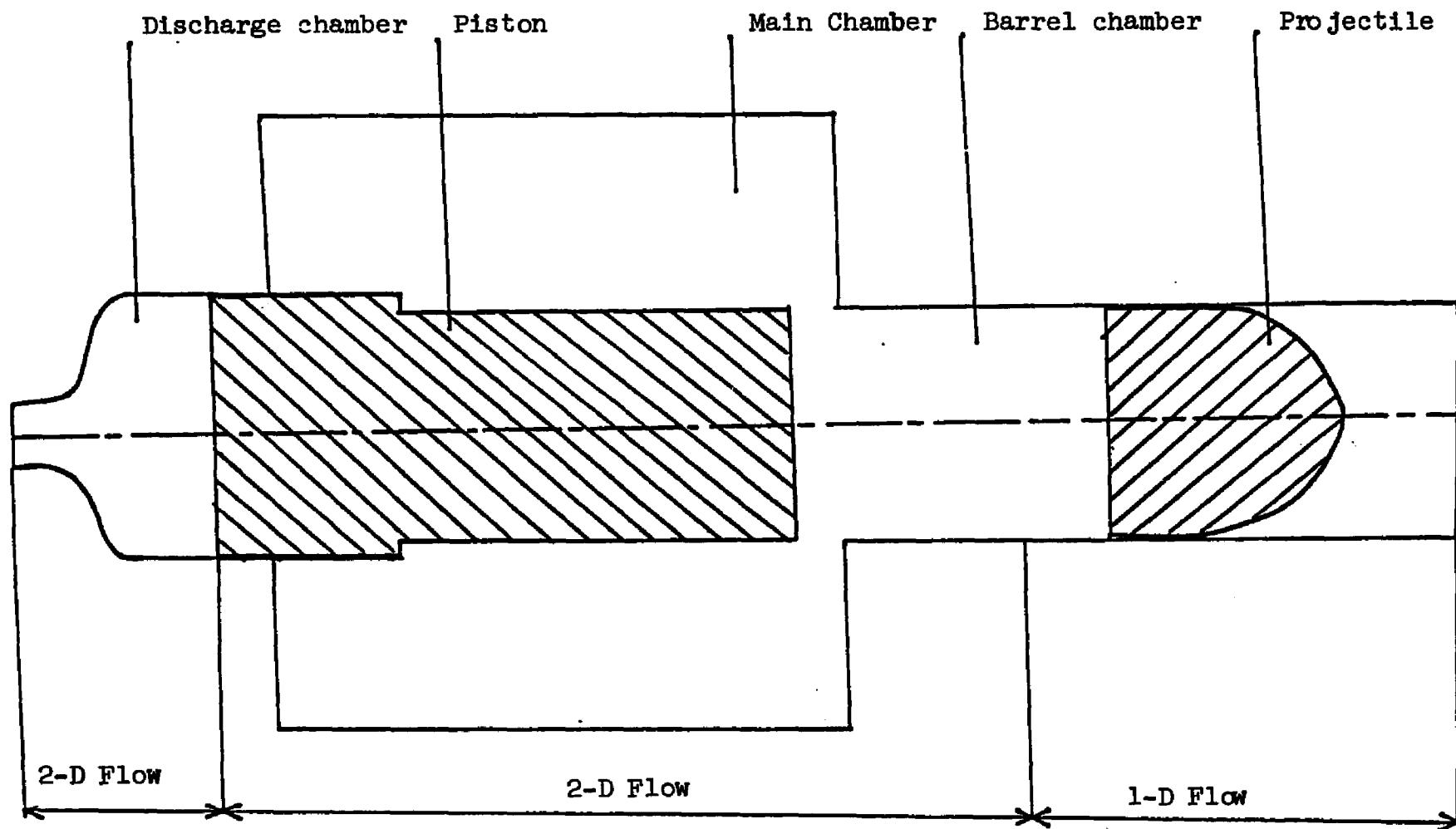


Figure 1. Schematic illustration of a gas gun.

2. Physical and Computational Flow Spaces.

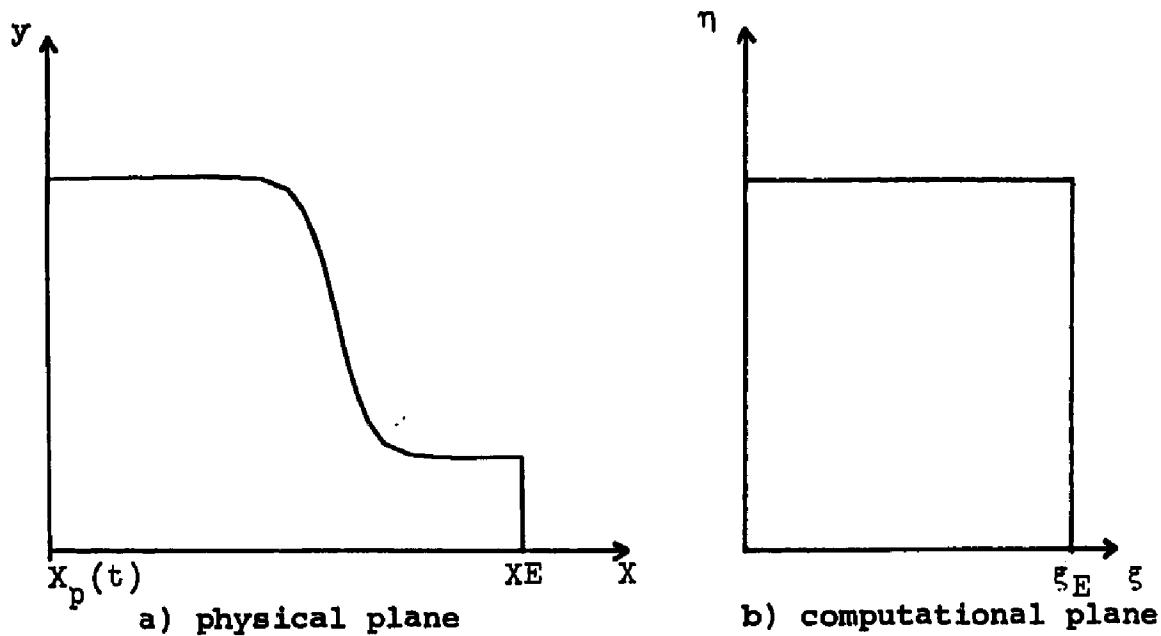


Figure 2. Schematic illustration of the discharge chamber.

The flow region enclosed by curved boundaries may be mapped into a rectangle by means of the following transformations, as shown in Figure 2.

$$g(x, t) = \frac{1}{\xi_E} \int_{x_p(t)}^x \sqrt{1 + y_w'^2(s)} \, ds \quad (6)$$

$$\eta(x, y) = \frac{y}{y_w(x)} \quad (7)$$

$$T = \frac{1}{2} \int d^3x \partial_\mu \phi \partial^\mu \phi - \frac{1}{2} m^2 \phi^2 + \frac{1}{4!} \lambda \phi^4 \quad (8)$$

where $y_w(x)$ is the wall shape as a function of x , $x_p(t)$ is the piston displacement as a function of t , and

$$\xi_E = \int_{x_p(t)}^{x_E} \sqrt{1 + y_w'^2(x)} dx$$

The transformation ξ stretches the axial coordinate x , such that a finite difference grid with uniform $\Delta\xi$ will concentrate the grid in regions of large curvatures. The functions ξ and η vary from 0 to 1.

The spatial and the time partial derivatives are then,

$$\frac{\partial}{\partial x} = w \frac{\partial}{\partial \xi} + \alpha \frac{\partial}{\partial \eta} \quad (9)$$

$$\frac{\partial}{\partial y} = \beta \frac{\partial}{\partial \eta} \quad (10)$$

$$\frac{\partial}{\partial t} = \delta \frac{\partial}{\partial \xi} + \frac{\partial}{\partial \tau} \quad (11)$$

Where $w = \frac{\sqrt{1 + y_w'^2(x)}}{\xi_E}$, $\alpha = -\eta \beta \frac{dy_w}{dx}$

$$\beta = \frac{1}{y_w(x)} , \text{ and } \delta = -\sqrt{1 + y_w'^2(x_p)} \frac{dx_p}{dt}$$

3. Transformed Governing Equations.

The governing Equations (1) - (4) can be written in the transformed plane as follows

$$\frac{\partial \rho}{\partial \tau} + u * \frac{\partial \rho}{\partial \xi} + v * \frac{\partial \rho}{\partial \eta} + \rho (w \frac{\partial u}{\partial \xi} + \alpha \frac{\partial u}{\partial \eta}) = - [\rho \beta \frac{\partial v}{\partial \eta} + \rho \frac{v}{\eta}] \quad (12)$$

$$\begin{aligned}
 \frac{\partial u}{\partial \tau} + u * \frac{\partial u}{\partial \xi} + v * \frac{\partial u}{\partial \eta} + \frac{1}{\rho} (w \frac{\partial p}{\partial \xi} + \alpha \frac{\partial p}{\partial \eta}) &= \frac{\mu}{\rho} [\beta \frac{\partial}{\partial \eta} (\beta \frac{\partial v}{\partial \eta}) \\
 + \frac{4}{3} (w \frac{\partial}{\partial \xi} + \alpha \frac{\partial}{\partial \eta}) (w \frac{\partial u}{\partial \xi} + \alpha \frac{\partial u}{\partial \eta}) + \frac{1}{3} (w \frac{\partial}{\partial \xi} + \alpha \frac{\partial}{\partial \eta}) (\beta \frac{\partial v}{\partial \eta})] \\
 + \frac{\mu}{\rho \bar{\eta}} [\frac{1}{3} (w \frac{\partial v}{\partial \xi} + \alpha \frac{\partial v}{\partial \eta}) + \beta \frac{\partial u}{\partial \eta}] \quad (13)
 \end{aligned}$$

$$\begin{aligned}
 \frac{\partial v}{\partial \tau} + u * \frac{\partial v}{\partial \xi} + v * \frac{\partial v}{\partial \eta} + \frac{\beta}{\rho} \frac{\partial p}{\partial \eta} &= \frac{\mu}{\rho} [(w \frac{\partial}{\partial \xi} + \alpha \frac{\partial}{\partial \eta}) (\frac{\partial v}{\partial \xi} + \alpha \frac{\partial v}{\partial \eta}) \\
 + \frac{4}{3} \beta \frac{\partial}{\partial \eta} (\beta \frac{\partial v}{\partial \eta}) + \frac{1}{3} (w \frac{\partial}{\partial \xi} + \alpha \frac{\partial}{\partial \eta}) (\beta \frac{\partial u}{\partial \eta})] \\
 + \frac{4}{3} \frac{\mu}{\rho} (\beta \frac{\partial v}{\partial \eta} - \frac{v}{\bar{\eta}}) \quad (14)
 \end{aligned}$$

$$\begin{aligned}
 \frac{\partial p}{\partial \tau} + u * \frac{\partial p}{\partial \xi} + v * \frac{\partial p}{\partial \eta} - a^2 (\frac{\partial p}{\partial t} + u * \frac{\partial p}{\partial \xi} + v * \frac{\partial p}{\partial \eta}) \\
 = (\gamma - 1) \mu \left\{ \frac{4}{3} \left[(w \frac{\partial u}{\partial \xi} + \alpha \frac{\partial u}{\partial \eta})^2 + (\beta \frac{\partial v}{\partial \eta})^2 \right] + [2\beta \frac{\partial u}{\partial \eta} (w \frac{\partial v}{\partial \xi} + \alpha \frac{\partial v}{\partial \eta}) \right. \\
 \left. - \frac{4}{3} (w \frac{\partial u}{\partial \xi} + \alpha \frac{\partial u}{\partial \eta}) (\beta \frac{\partial v}{\partial \eta}) \right] + [(w \frac{\partial v}{\partial \xi} + \alpha \frac{\partial v}{\partial \eta})^2 + (\beta \frac{\partial u}{\partial \eta})^2] \\
 + \left[\frac{4}{3} \frac{v^2}{\bar{\eta}^2} - \frac{4}{3} \frac{v}{\bar{\eta}} (\beta \frac{\partial v}{\partial \eta} + (w \frac{\partial u}{\partial \xi} + \alpha \frac{\partial u}{\partial \eta})) \right] \right\} \quad (15)
 \end{aligned}$$

where,

$$u^* = \delta + w u,$$

$$v^* = \beta v + \alpha u,$$

$$\bar{\eta} = \frac{\eta}{\beta}$$

The transformed Navier-Stokes equations are somewhat more complicated than the original ones, because new

coefficients are introduced, but these equations offer several advantages. The main advantage is that the physical boundary surface can be mapped into a rectangular domain surface in the transformed plane. Another significant aspect of the transformation is that the grid can be concentrated in regions that experience rapid changes in flow field gradients or variables.

4. Numerical Method.

At the present time, four numerical methods are applicable to the solution of Navier-Stokes equations. Namely, they are the method of finite differences, the method of characteristics, the method of integral relations, and the fluid-in-cell method. The first two methods are used in this problem. The method of finite differences consists of replacing derivatives by finite differences, such that the discretized forms are consistent with the physics of the Navier Stokes equations. If the physical parameters in the neighborhood of any point in the flow field are Taylor-expandable and their Taylor series are rapidly convergent, then the discretization errors can be evaluated and kept within tolerable limits by considering a sufficiently small neighborhood. However, in real problems, the flow parameters may not be Taylor-expandable. In this case a special treatment is needed, one such is the artificial viscosity. Sometimes even if the flow parameters are Taylor-expandable, their Taylor series start diverging and begin

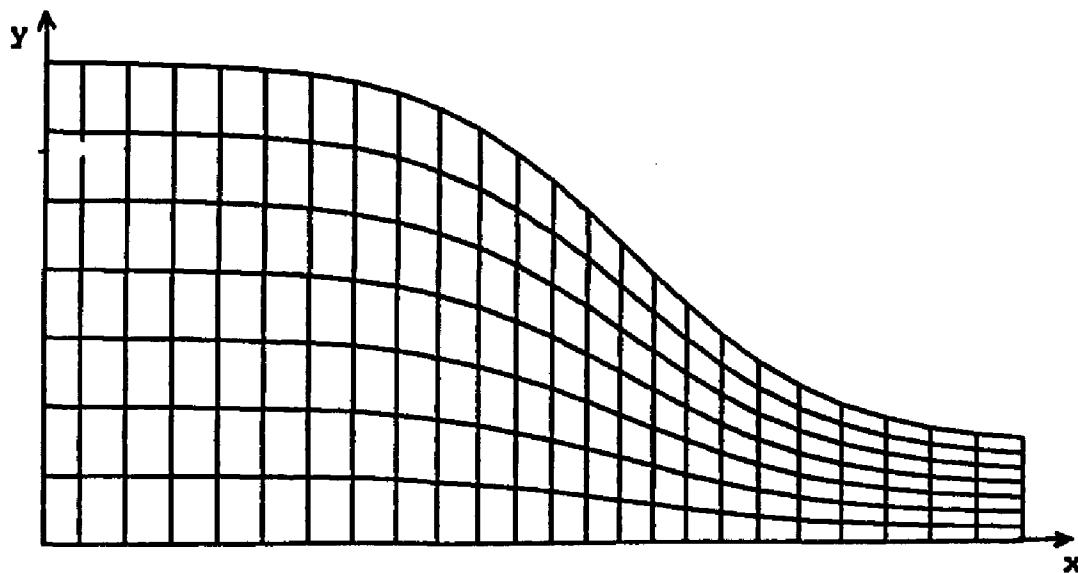
converging only when certain terms are taken into account. Such terms are the fourth order dissipation, their order being higher than the second order of the most finite difference schemes. These fourth order damping terms have been found helpful in the case of problems with non-smooth initial data.

The method of characteristics is to determine the characteristic curves and the corresponding compatibility equations which can be solved simultaneously at grid points for flow properties. A characteristic curve is a curve in the solution plane in which the governing partial differential equations may be combined to form a set of ordinary differential equations.

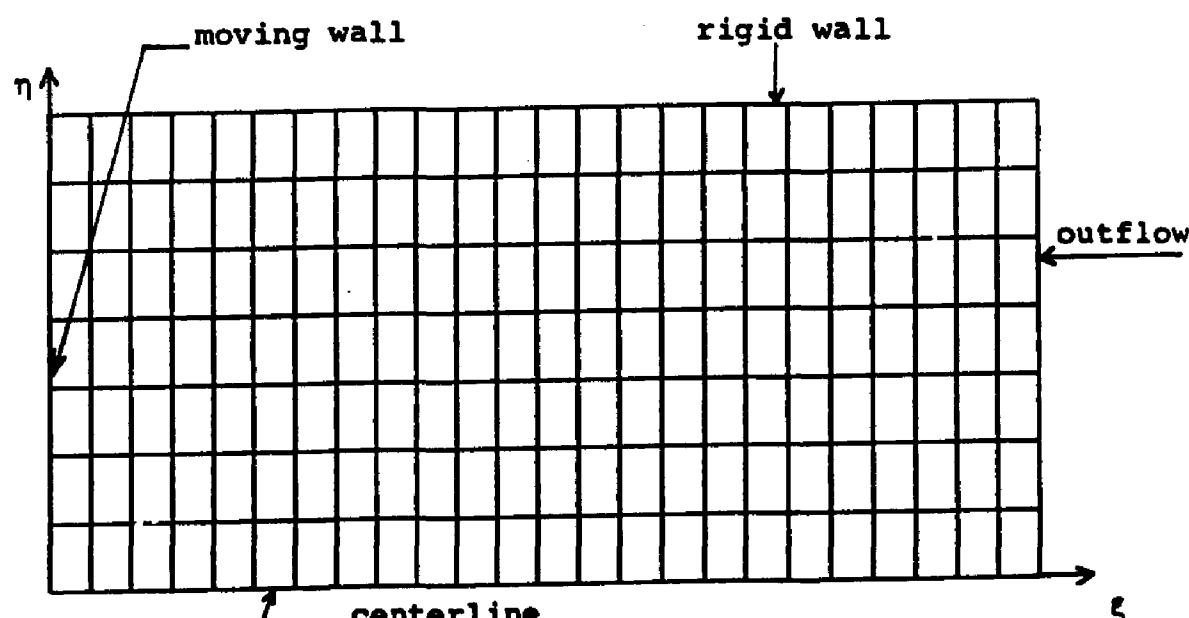
A numerical algorithm for determining the solution at a given point in the transformed plane can now be presented. The computational plane, shown in Figure 3, is divided into two sets of mesh points, the interior grid points and the boundary grid points.

4.1 Interior Grid Points

The interior grid points are computed using the explicit predictor-corrector finite difference method presented by MacCormack³ in 1969. MacCormack's method has been used successfully by a number of investigators. The finite difference approximates the governing equations to second-order accuracy in space and time, are simple to program, but are conditionally stable. In this case



a) Physical computational mesh.



b) Rectangular computational mesh

Figure 3. Physical and computational domains

it seemed advantageous to use the explicit scheme. The governing equations are left in non conservative form in order to apply the boundary conditions more easily. As an example, the finite difference equations approximating Equation (13) are,

$$\begin{aligned}
 \Delta U_{L,M} &= -\Delta t [(\delta_{L,M}^N + w_{L,M} U_{L,M}^N) \frac{U_{L,M}^N - U_{L-1,M}^N}{\Delta x} + (\beta_{L,M} V_{L,M}^N \\
 &+ \alpha_{L,M} U_{L,M}^N) \frac{U_{L,M}^N - U_{L,M-1}^N}{\Delta y} - \frac{1}{\rho_{L,M}^N} (w_{L,M} \frac{P_{L,M}^N - P_{L-1,M}^N}{\Delta x} \\
 &+ \alpha_{L,M} \frac{P_{L,M}^N - P_{L,M-1}^N}{\Delta y})] + \mu \frac{\beta_{L,M}}{\rho_{L,M}^N} \frac{\Delta t}{\Delta y} [\beta_{L,M} \frac{V_{L,M+1}^N - V_{L,M}^N}{\Delta y} \\
 &- \beta_{L,M} \frac{V_{L,M}^N - V_{L,M-1}^N}{\Delta y} + \frac{4}{3} \frac{\mu \cdot w_{L,M}}{\rho_{L,M}} \frac{\Delta t}{\Delta x} [(w_{L,M} \frac{U_{L+1,M}^N - U_{L,M}^N}{\Delta x} \\
 &+ \frac{\alpha_{L,M}}{2} \frac{U_{L+1,M+1}^N - U_{L+1,M-1}^N}{2\Delta y} - w_{L,M} \cdot \frac{U_{L,M}^N - U_{L-1,M}^N}{\Delta x} \\
 &- \frac{\alpha_{L,M}}{2} \cdot \frac{U_{L-1,M+1}^N - U_{L-1,M-1}^N}{2\Delta y}] + \frac{4}{3} \frac{\mu \cdot \alpha_{L,M}}{\rho_{L,M}^N} \frac{\Delta t}{\Delta y} \\
 &[\frac{w_{L,M}}{2} \frac{U_{L+1,M+1}^N - U_{L-1,M+1}^N}{2\Delta x} + \alpha_{L,M} \frac{U_{L,M+1}^N - U_{L,M}^N}{\Delta y} \\
 &- \frac{w_{L,M}}{2} \cdot \frac{U_{L+1,M-1}^N - U_{L-1,M-1}^N}{2\Delta x} - \alpha_{L,M} \cdot \frac{U_{L,M}^N - U_{L,M-1}^N}{\Delta y}] \\
 &+ \frac{1}{3} \frac{\mu \cdot w_{L,M}}{\rho_{L,M}} \frac{\Delta t}{\Delta x} [\frac{\beta_{L,M}}{2} \frac{V_{L+1,M+1}^N - V_{L+1,M-1}^N}{2\Delta y}
 \end{aligned}$$

$$\begin{aligned}
 & - \frac{\beta_{L,M}}{2} \frac{v_{L-1,M+1}^N - v_{L-1,M-1}^N}{2\Delta y} + \frac{1}{3} \frac{\mu \cdot \alpha_{L,M}}{\rho_{L,M}} \frac{\Delta t}{\Delta y} [\beta_{L,M} \cdot \frac{v_{L,M+1}^N - v_{L,M}^N}{\Delta y}] \\
 & - \beta_{L,M} \cdot \frac{v_{L,M}^N - v_{L,M-1}^N}{\Delta y}] + \frac{\mu}{N\rho_{L,M}} \frac{\Delta t}{\Delta y} [\frac{1}{3} (w_{L,M} \cdot \frac{v_{L+1,M}^N - v_{L-1,M}^N}{2\Delta x}) \\
 & + \alpha_{L,M} \frac{v_{L,M+1}^N - v_{L,M-1}^N}{2\Delta y}] + \beta_{L,M} \frac{u_{L,M+1}^N - u_{L,M-1}^N}{2\Delta y}] \quad (16)
 \end{aligned}$$

For the predictor step, we have

$$\bar{u}_{L,M}^{N+1} = u_{L,M}^N + \Delta u_{L,M} \quad (17)$$

And,

$$\begin{aligned}
 \bar{\Delta u}_{L,M}^{N+1} &= -\Delta t [(\delta_{L,M}^N + w_{L,M} \bar{u}_{L,M}^{N+1}) \frac{\bar{u}_{L+1,M}^{N+1} - \bar{u}_{L,M}^{N+1}}{\Delta x} + (\beta_{L,M} \bar{v}_{L,M}^{N+1} \\
 & + \alpha_{L,M} \bar{u}_{L,M}^{N+1}) \frac{\bar{u}_{L,M+1}^{N+1} - \bar{u}_{L,M}^{N+1}}{\Delta y} - \frac{1}{\rho_{L,M}} (w_{L,M} \frac{\bar{p}_{L+1,M}^{N+1} - \bar{p}_{L,M}^{N+1}}{\Delta x} \\
 & + \alpha_{L,M} \frac{\bar{p}_{L,M+1}^{N+1} - \bar{p}_{L,M}^{N+1}}{\Delta y})] + QUT \quad (18)
 \end{aligned}$$

For the corrector step we have,

$$u_{L,M}^{N+1} = \frac{1}{2} (u_{L,M}^N + \bar{u}_{L,M}^{N+1} + \bar{\Delta u}_{L,M}^{N+1}) \quad (19)$$

where L = subscript for the axial mesh point

M = subscript for the radial mesh point

N = time step

QUT = viscous terms computed in the predictor step

The first step predicts a new solution at time $t = (N + 1)\Delta t$ at each mesh point (L, M) from the known solution at time $t = N\Delta t$, and uses backward differences to approximate the first derivatives and centered differences to approximate the second derivatives; that is, the viscous terms.

The second step corrects the predicted values with forward differences for the first derivatives. Note that the viscous terms are not computed in the corrector step for reasons given by Cline⁴. This method is stable if it satisfies the following condition given in the transformed plane,

$$\Delta t \leq \frac{A}{|U|w/\Delta \xi + |V|\beta/\Delta \eta + a \sqrt{w^2/\Delta \xi^2 + \beta^2/\Delta \eta^2}} \quad (20)$$

where A is a coefficient close to 1, but not greater than 1 for inviscid flows, and much less than 1, ($A \approx 0.4$ or less), for viscous flows and flows with shocks.

4.2 Boundary Grid Points

The boundary conditions for the present analysis are divided into 4 categories,

- centerline grid points
- wall grid points
- moving wall grid points
- exit grid points

4.2.1. Center Line Grid Points

The boundary condition at the centerline is that of

symmetry, i.e. $v = 0$. However, since there is a singular term at the axis ($y = 0$), it is replaced by its limiting form as y goes to zero, that is, $\lim_{y \rightarrow 0} \frac{v}{y} = \frac{\partial v}{\partial y}$.

4.2.2. Wall Grid Points

(1) Free slip walls

The boundary condition for free-slip walls is that the flow is tangent to the wall. This can be written as $v dx - u dy_w = 0$ (i.e. wall is a streamline)

The boundary conditions for no-slip walls are that the velocity components vanish at the wall.

The numerical treatment of the boundary conditions for free-slip walls is to prescribe one flow condition, and then calculate the remaining flow parameters by using one of the following schemes:

- (1) Extrapolation (zeroth order or first order)
- (2) Reflection principle
- (3) One sided difference (first order or second order accurate)
- (4) Reference plane characteristic

The methods (1) - (3) can cause some problems in some cases, such as in subsonic inflow and outflow (Ref.[5].) where errors made in prescribing the boundaries propagate in all directions. The method (4) has been originally proposed by Moretti⁵ and used by a number of fluid dynamicists, notably, Serra⁶, and Cline⁴. Cline has developed the method for all the boundaries (entrance, exit, and

walls). For this problem the methods (1), (2), and (4) have been used. Better solutions have been obtained with the method (4). Method(2) has failed to converge (see Figure 9). The reference plane characteristic method is derived in Appendix C, and it follows closely the work done by Cline⁴. Three characteristic relations that relate the interior flow to the wall boundary with the given boundary condition form a system of four coupled equations for u , v , ρ , and P . This can be written as,

$$\beta du - \alpha dv = (\beta \Psi_2 - \alpha \Psi_3) d\tau \quad (22)$$

$$dP - a^2 d\rho = \Psi_4 d\tau \quad (23)$$

along the characteristic $d\eta = v^* d\tau$

and,

$$dP + \rho a \alpha du/\alpha^* + \rho \beta a dv/\alpha^* = (\Psi_4 + a^2 \Psi_1 + \rho \alpha a \Psi_2/\alpha^* + \rho \beta a \Psi_3/\alpha^*) d\tau \quad (24)$$

along the characteristic $d\eta = (v^* + \alpha^* a) d\tau$

$$\text{The boundary condition is: } vdx - u dy_w = 0. \quad (25)$$

The coefficients Ψ_1 , Ψ_2 , Ψ_3 , and Ψ_4 are given in Appendix C. The prescribed boundary condition is the function representing the shape of the wall, i.e. $y_w(x)$. Equations (22) - (25) are written in finite difference form. The differentials du , dv , $d\rho$, and dP are replaced by differences along the characteristic curves. The coefficients are evaluated by the MacCormack scheme.

(2) No slip walls

The numerical treatment of the boundary conditions for no-slip walls is much simpler. The velocity components u and v vanish at the wall. The remaining variables P and ρ are calculated from the energy equation and the continuity equation, respectively.

4.2.3. Moving Wall Grid Points

The boundary condition at the moving wall is the velocity of the piston. The velocity is obtained from the equation of motion of the moving piston. This can be written

$$\frac{du_p}{dt} - \frac{1}{m_p} \left[\frac{2\pi}{\theta^2} \int_0^1 (P_{LHS} - P_{RHS}) \eta d\eta - D \right] dt = 0 \quad (26)$$

In this equation m_p is the piston mass, P_{LHS} is the pressure at the left hand side of the piston, P_{RHS} is the pressure at the right hand side of the piston, and D is the friction force. The characteristic expression that relates the interior points to the moving boundary can be written as,

$$dP - \rho a du = (\Psi_4 + a^2 \Psi_1 - \rho a \Psi_2) dt \quad (27)$$

along the characteristic $d\xi = (u^* - aw) dt$

The prescribed flow parameters are $v = 0$, ρ is extrapolated from the adjacent grid. The pressure P , and the velocity u are obtained from the solution of Equations (26) and (27). Equations (26) and (27) are written in finite

difference form. If the pressure is obtained by the method (2), then the solution diverges. (see Figure 9.) The finite difference form for Equation (26) can be written as,

$$U^{N+1}(L, M) = U^N(L, M) + \frac{\Delta t}{M_p} [(\bar{P}_{LHS} - \bar{P}_{RHS}) A_p - D] \quad (28)$$

the overbar denotes the average values.

4.2.4. Exit Grid Points

The boundary condition at the exit plane can be supersonic outflow or a subsonic outflow. For supersonic outflow, the normal velocity at the exit is greater than the speed of sound. Then no signals from outside can propagate into the discharge chamber. In this case the flow variables are evaluated from the interior values by a simple linear extrapolation, and the error generated from the extrapolation is not expected to propagate back and affect the upstream results. For subsonic outflow a special treatment is needed because any arbitrariness in the choice of the boundary values sends arbitrary signals inside the computational region. It can reflect pressure disturbances which are damped only by dissipative-effects. A non-reflecting outflow boundary condition has been used for this purpose by Serra⁵, Cline⁴, C. K. Chu and Aron Sereny⁷ and Rudy⁸. It seems advantageous to use this form in the present analysis. The prescribed boundary condition is

the ambient pressure, $P = P_A$. In this case the problem is well posed, i.e., the solution is unique and continuous at the boundary. The characteristic relations that relate the interior points to the exit boundary points can be written as,

$$dP - a^2 d\rho = \Psi_4 d\tau \quad (29)$$

$$dv = \Psi_3 d\tau$$

along the characteristic $d\zeta = u^* d\tau$,

$$dP + \rho a du = (\Psi_4 + a^2 \Psi_1 + \rho a \Psi_2) d\tau \quad (30)$$

along the characteristic $d\zeta = (u^* + aw) d\tau$

Equations (29) - (30) are solved by the same methods used in Part C.

5. Results

The function y_w describing the shape of the wall can now be approximated by the following equation,

$$y_w(x) = (R_i + R_e)/2 - (R_i - R_e) \tanh \sigma (x - x_0) \quad (31)$$

where R_i and R_e are, respectively, the radius at the entrance and at the exit of the discharge chamber, σ is a parameter that controls the curvature of the wall, and x_0 is the point about which the grid is concentrated. The mapping functions given by the Equations (6) - (8) become,

$$\xi(x, t) = \frac{\int_{x_0(t)}^x \sqrt{1 + \frac{(R_i - R_e)^2 \sigma^2}{\cosh^4 \sigma(x-x_0)}} dx}{\int_{x_0(t)}^{x_E} \sqrt{1 + \frac{(R_i - R_e)^2 \sigma^2}{\cosh^4 \sigma(x-x_0)}} dx} \quad (32)$$

$$\eta(y, x) = \frac{y}{(R_i + R_e)/2 - (R_i - R_e) \tanh \sigma(x-x_0)} \quad (33)$$

$$\tau = t \quad (34)$$

The coefficients β , w , α , and δ are also obtained by substituting the value of $y_w(x)$ given in Equation (31). They are computed in the mapping subroutine.

The inversion of the function ξ is done by interpolation of tabular values. The grid generated in the discharge chamber for four different wall shapes is, respectively, shown in Figures 4a, 4b, 4c, and 4d. In each case, the physical space grid consists of 50 by 16 grid points. The inlet and outlet radii are, respectively, set equal to 6.35 and 1.27 CM. The coefficient, σ , that controls the wall curvature is, respectively, set equal to 1.0, 6.0, 10.0, and 15.0. Therefore, a grid can be generated from Equations (32) and (33) to handle the rapid change along the steep wall of the discharge chamber.

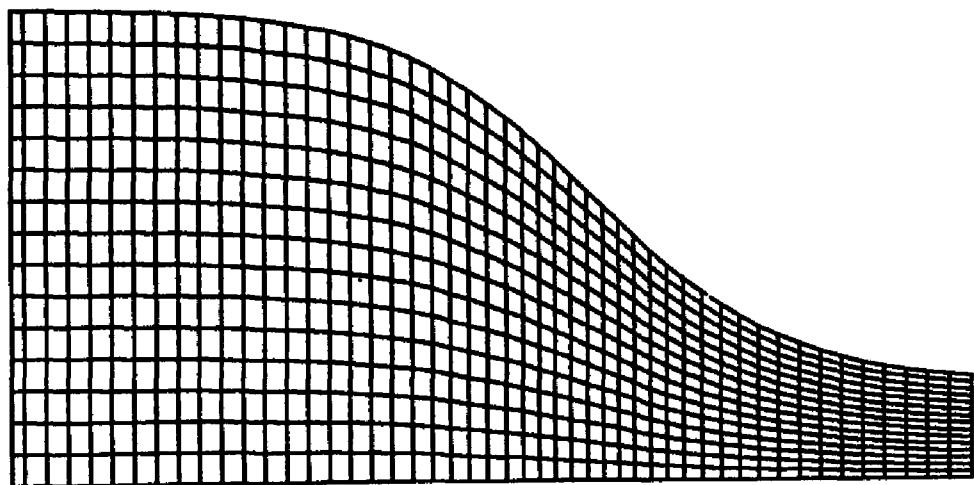


Figure 4a. Grid generated in the discharge chamber,
($\sigma = 1.0$)

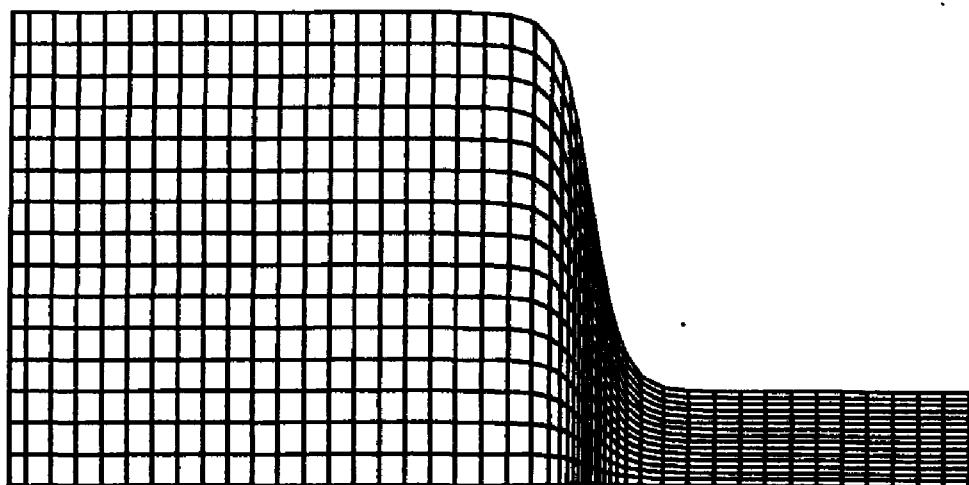


Figure 4b. Grid generated in the discharge chamber,
($\sigma = 6.0$)

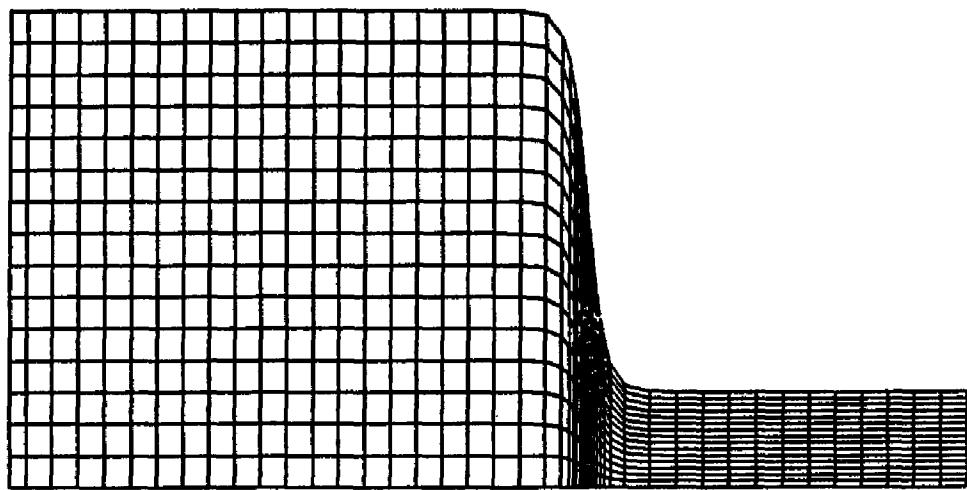


Figure 4c. Grid generated in the discharge chamber,
($\sigma = 10.0$)

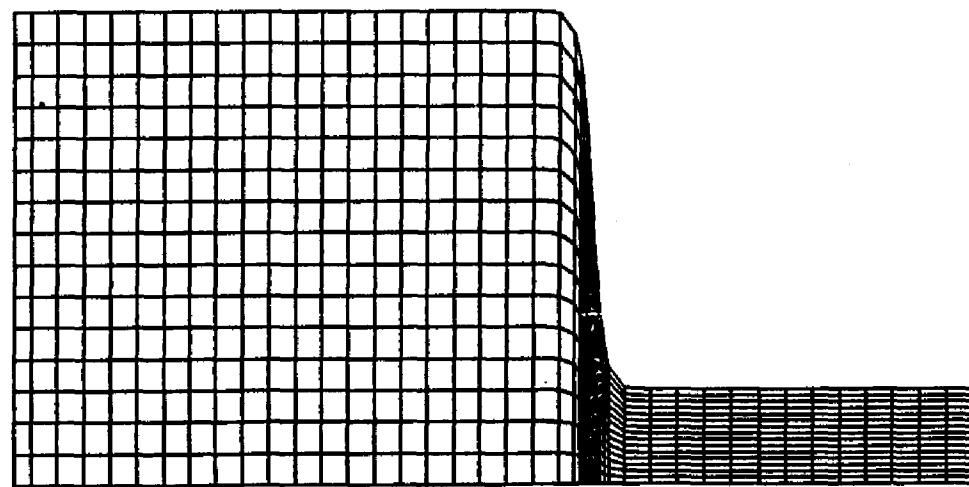


Figure 4d. Grid generated in the discharge chamber,
($\sigma = 15.0$)

Following the same technique described above, one can generate a grid in the discharge chamber used in Pakarat⁹'s experiment, Figure 5. For this case the shape of the wall is,

$$y_w(x) = \begin{cases} \frac{R_1 + R_2}{2} - \frac{R_1 - R_2}{2} \tanh \sigma_1 (x-x_1) & \text{if } x < \frac{x_1 + x_2}{2} \\ \frac{R_2 + R_3}{2} - \frac{R_2 - R_3}{2} \tanh \sigma_2 (x-x_2) & \text{if } x \geq \frac{x_1 + x_2}{2} \end{cases} \quad (35)$$

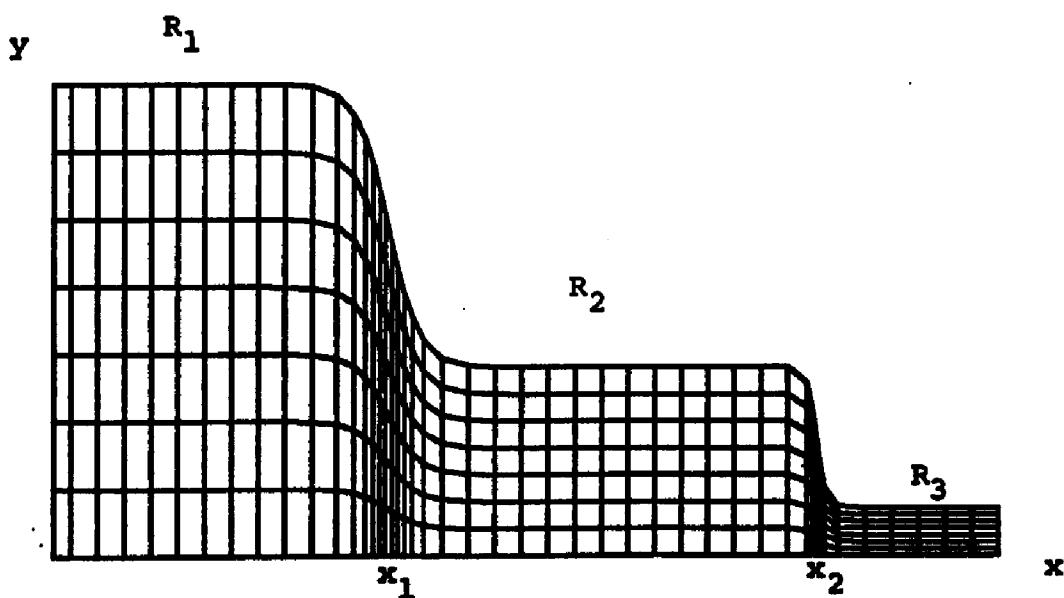


Figure 5. Grid generated in the discharge chamber used in Pakarat's experiment.

A grid generation can also be extended to the discharge chamber whose wall contains n different curvatures,

Figure 6. In this case the shape of the wall is given by Equation (36).

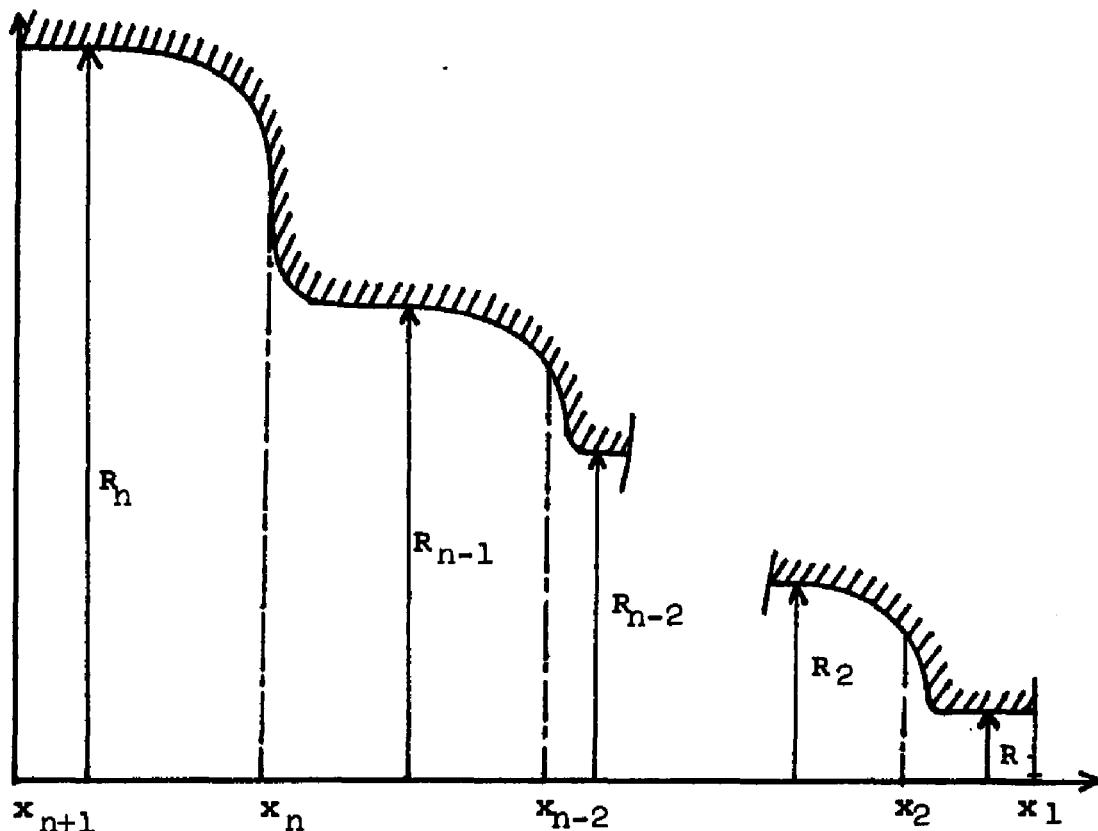


Figure 6. Discharge chamber in the case where the wall has n different curvatures.

$$y_w(x) = \sum_{j=2}^n \chi_K(x) \left(\frac{R_j + R_{j-1}}{2} - \frac{R_j - R_{j-1}}{2} \right) \tanh \sigma_j (x - x_j) \quad (36)$$

where $K = [\frac{x_{j+1} + x_j}{2}, \frac{x_j + x_{j-1}}{2}]$

Figures 7-9 show the velocity vector plots of the flow in the discharge chamber. The physical space grid consists of 25 by 8 grid points. The discharge pressure, P_c , is set equal to 986.KPA. The coefficient, a , that controls the wall curvature is set equal to 6.0. The inlet and outlet radii are, respectively, set equal to 6.35 and 1.27 CM, $X_0 = 7.62$ CM, and finally the maximum number of time steps, N MAX, is set equal to 15000. It can be seen in Figures 7-8 that the transient flow occurring during the discharge phenomenon is accurately simulated. The velocity vector plots are shown every 800 time steps. Figures 7a and 7b show that the flow is sonic at the exit if the pressure in the discharge chamber is above 200.KPA. Then as the pressure gets below 170 KPA, the flow is subsonic (see Figure 7C).

Figure 7d demonstrates a case of an inflow. This phenomenon can occur if the ambient pressure is higher than the pressure in the discharge chamber. It is seen from Figure 13, that the pressure varies only slightly with position. The same results have been observed by Pakarat⁹ in his experimental study. Comparison of the pressure distributions calculated by the present technique with the one dimensional solution are shown in Figure 10. This recent work shows improvements in the mathematical model of the discharge chamber flow. The one-dimensional solution used earlier (Ref. [2]) predicts a discharge

time of 5 ms. The present solution indicates a discharge time of 13 ms. However it requires a computational time of 11 min and 29 s.

This numerical model has also been applied to the discharge chamber used by Pakarat⁹ in his experimental study. The experimental time for the discharge chamber is about 160 ms. In this case the computational time is expected to be much larger because the grid surface generation has produced more mesh points and smaller grid sizes. The model simulates the discharge cycle only for a short time. The results are shown in Figures 14 through 17. These results are in better correlation with the experimental data.

In conclusion the Navier-Stokes equations can be solved numerically as a means of predicting the flow fields generated during the discharge. However it is too expensive to be accepted as an engineering solution.

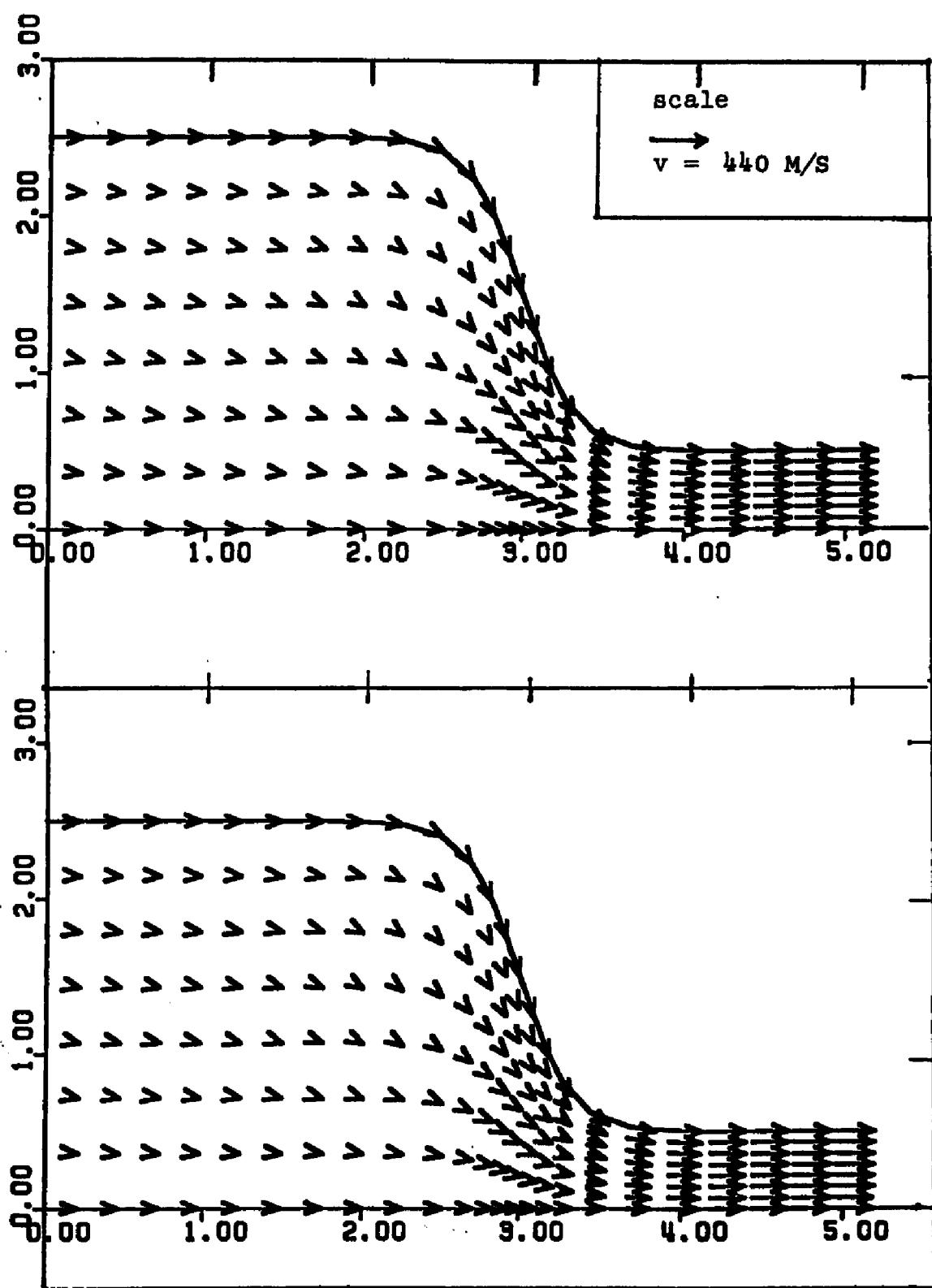


Figure 7. Velocity vector plots of the flow in the discharge chamber.

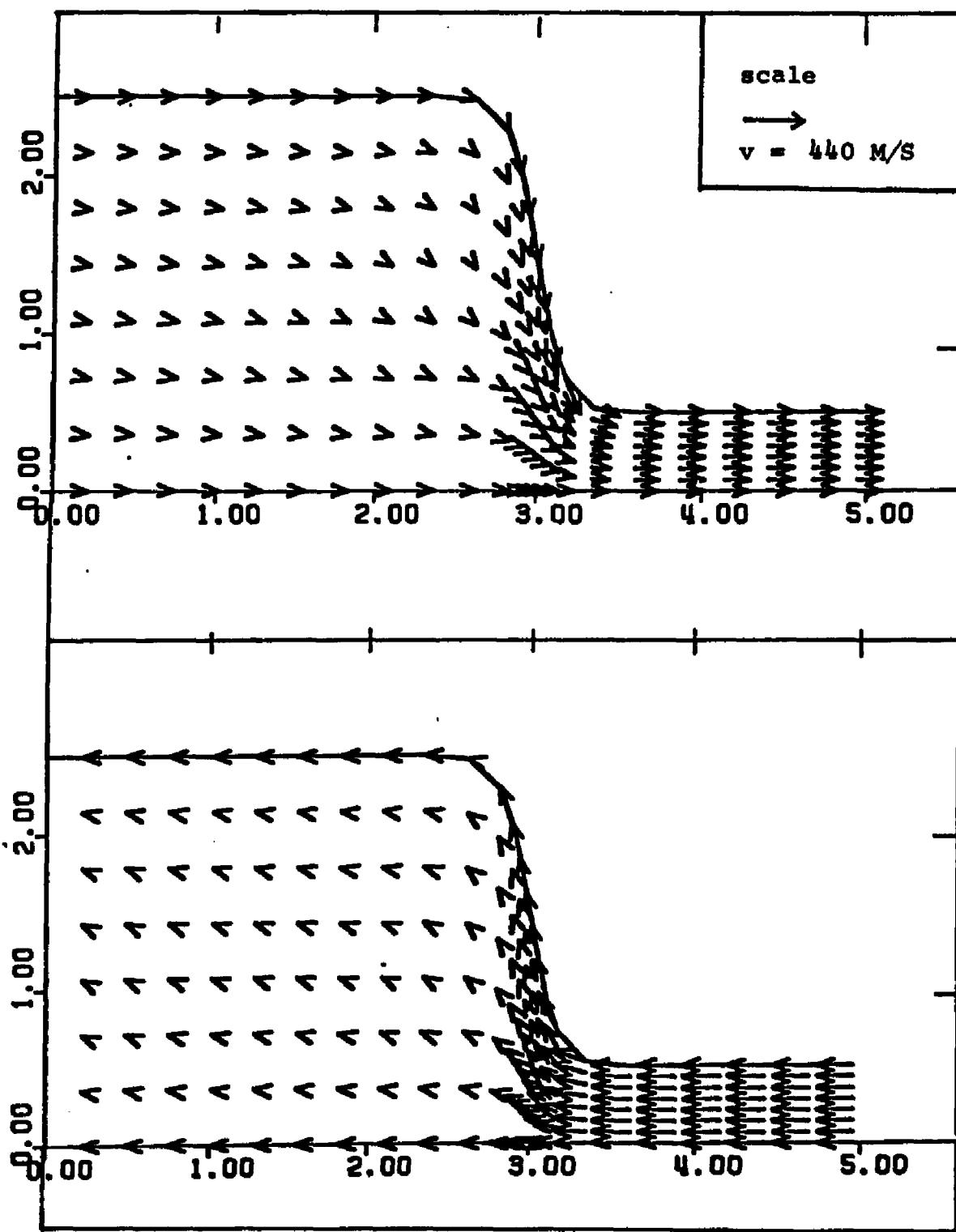


Figure 7 cont.

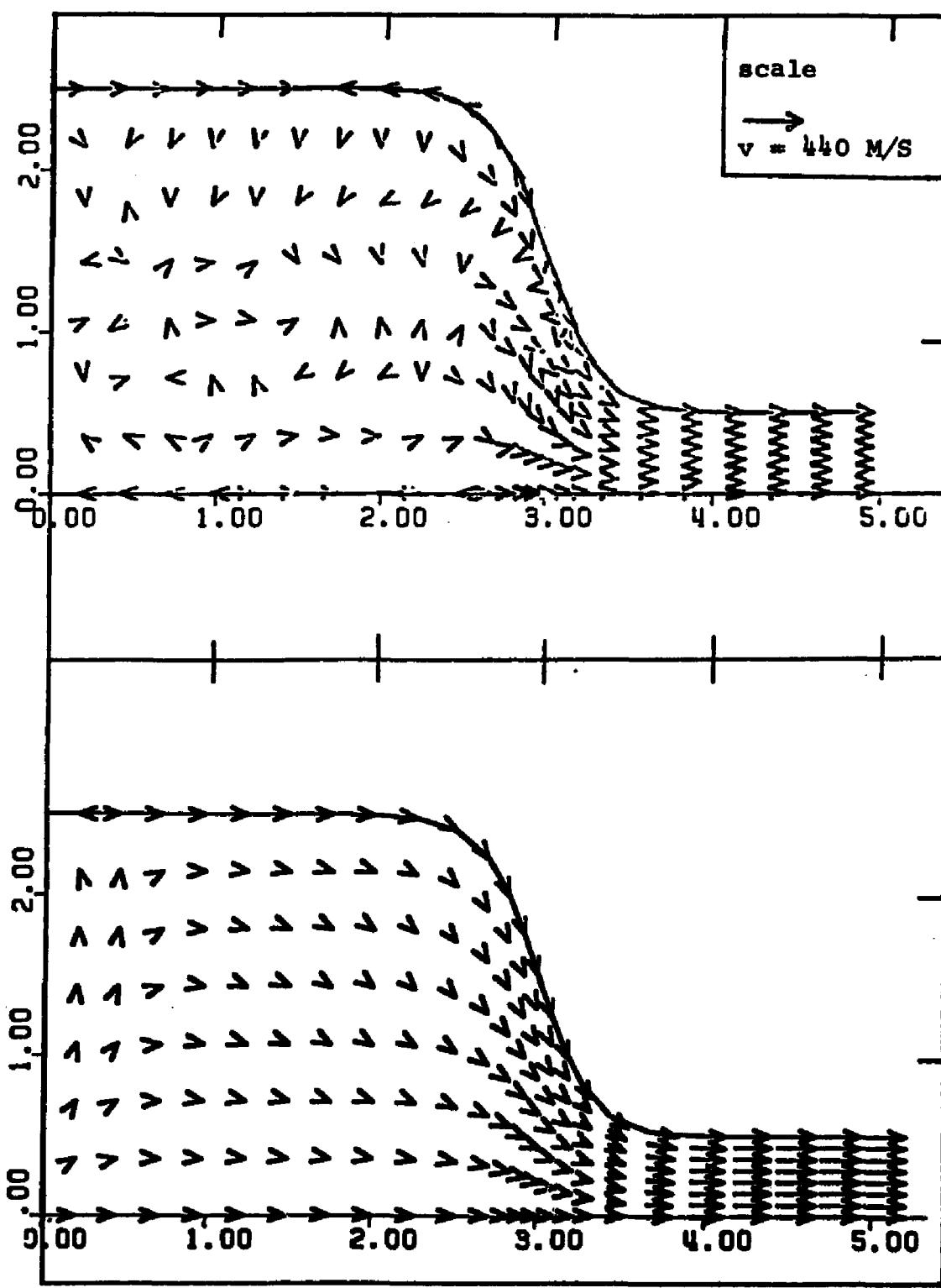


Figure 8. Velocity vector plots of the flow when smoothing devices are used.

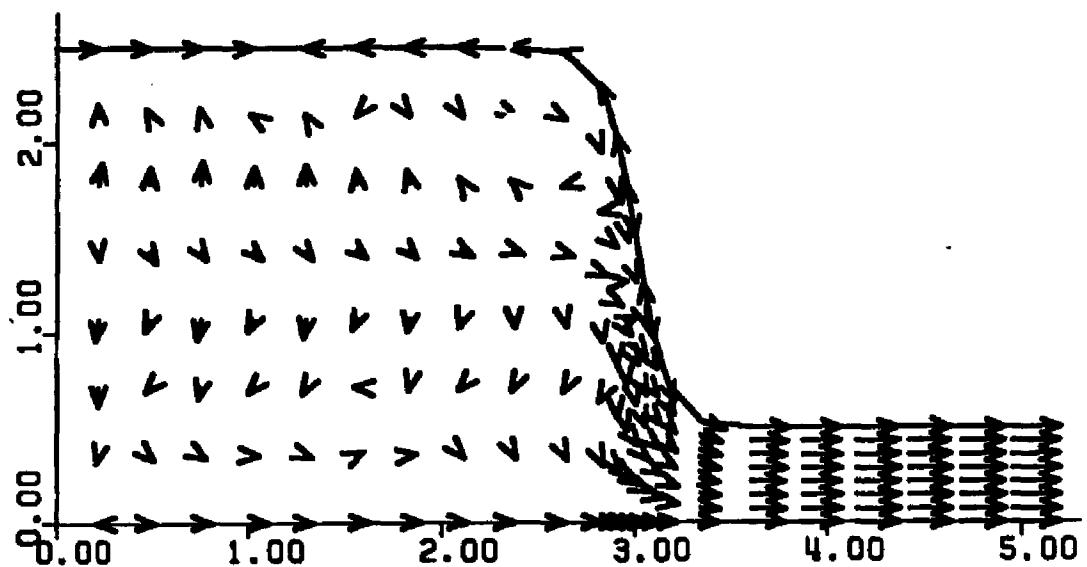


Figure 9. Velocity vector plots of the flow when the method of characteristics is not used at the boundaries.

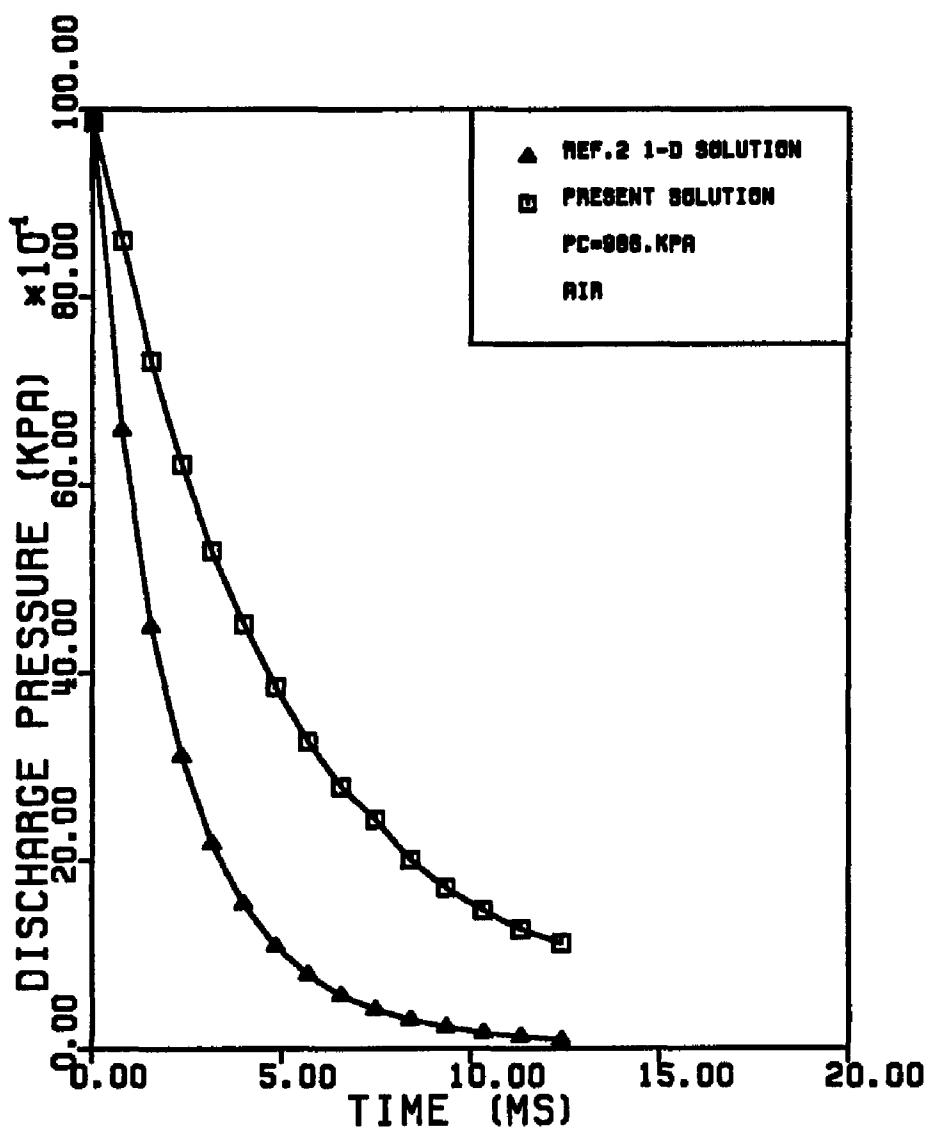


FIGURE 10. DISCHARGE PRESSURE VS TIME

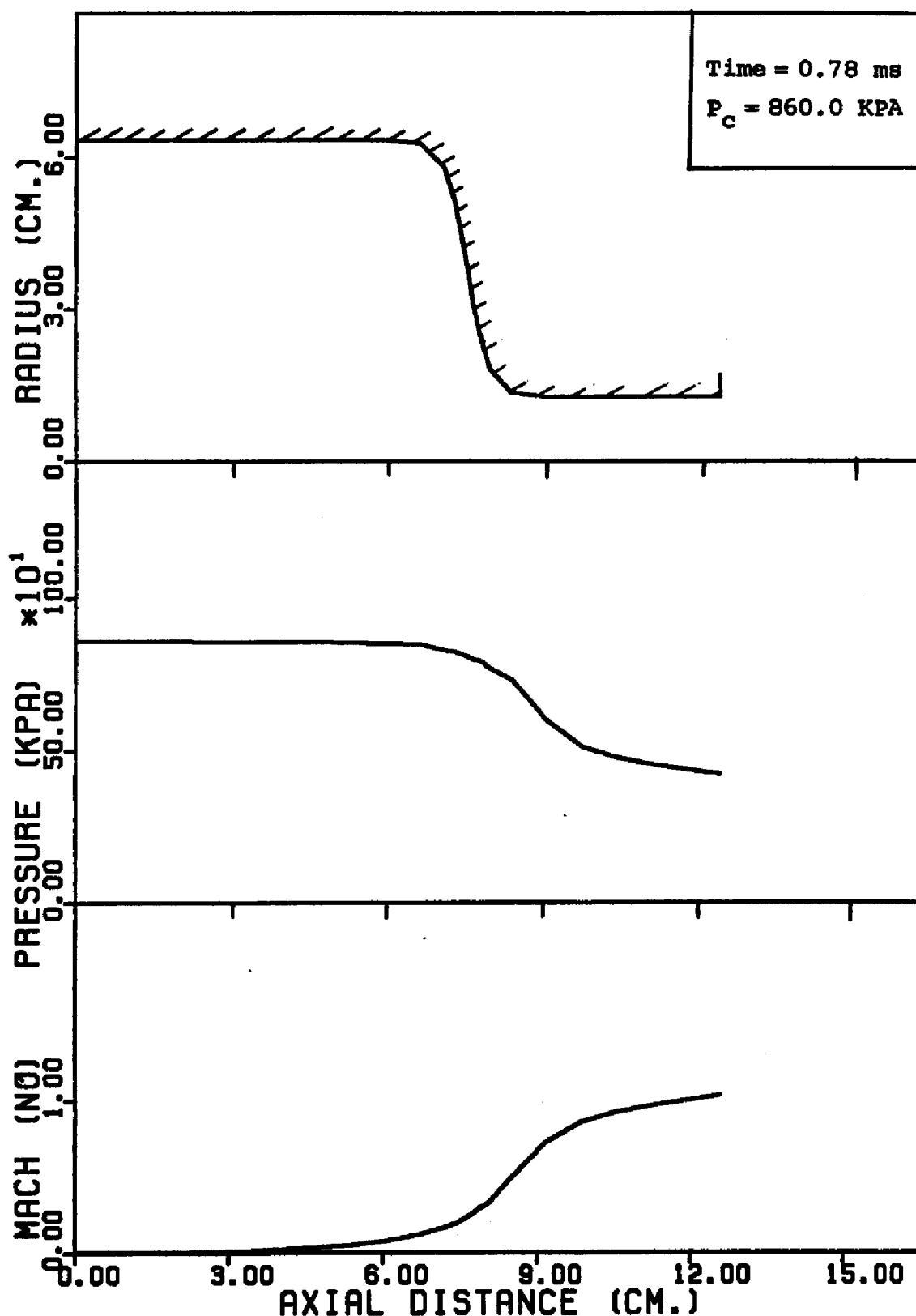


Figure 11. Discharge chamber geometry, and midplane pressure and Mach number for inviscid flow.

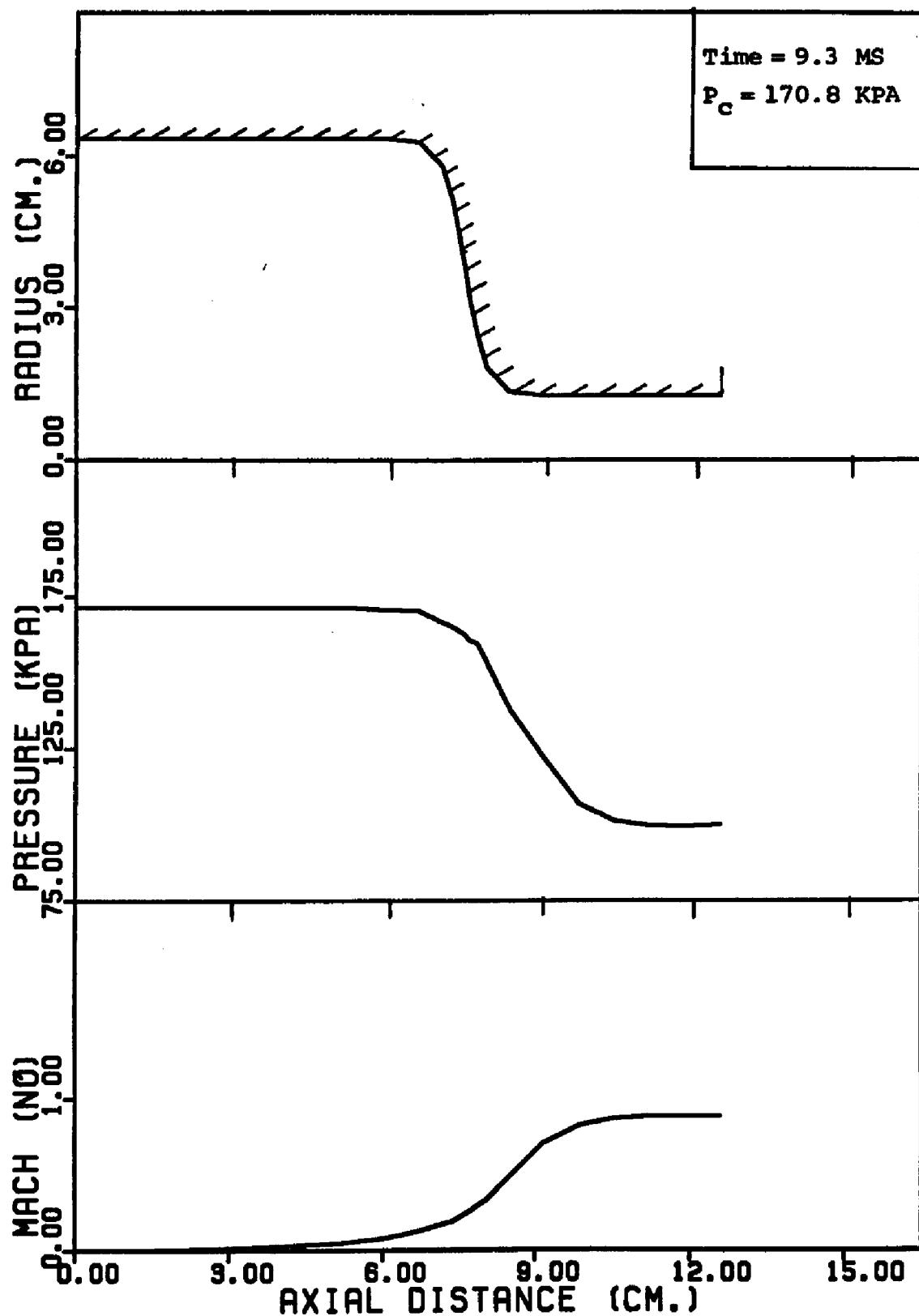


Figure 12. Discharge chamber geometry, and midplane pressure and Mach number for inviscid flow.

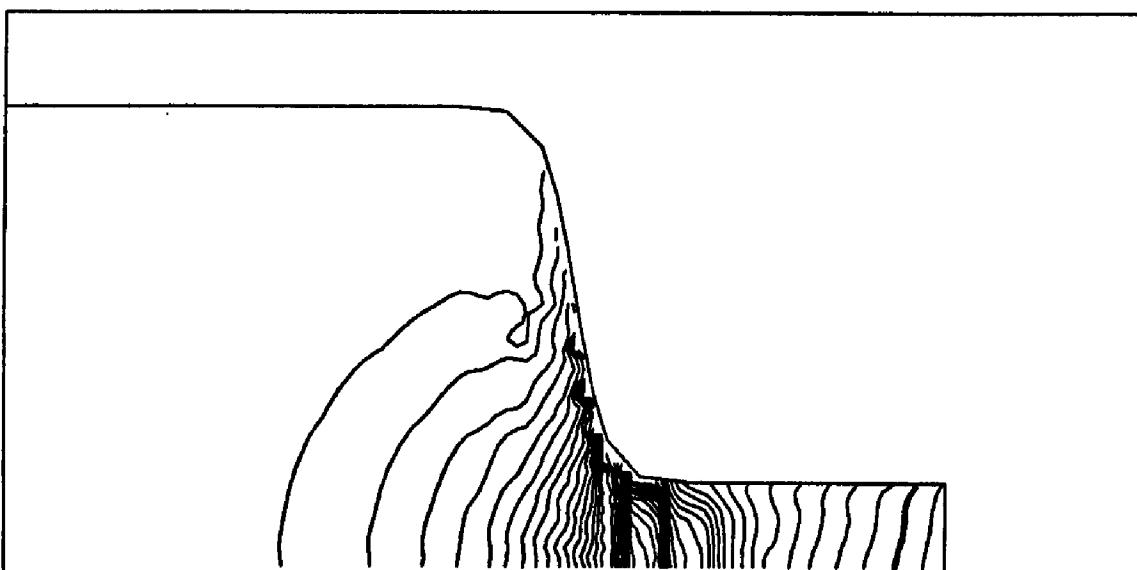
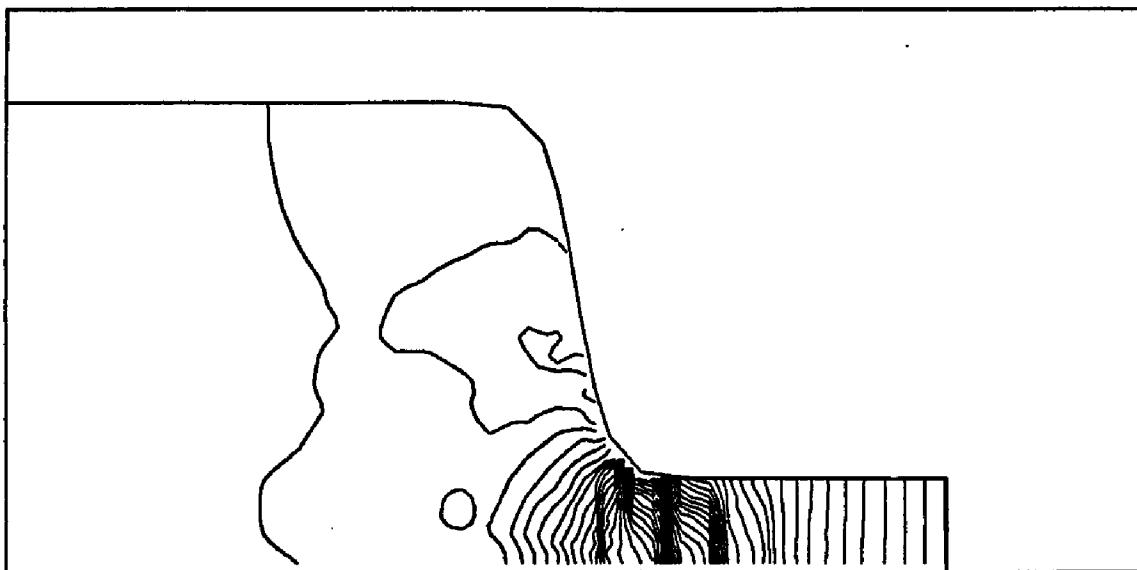


Figure 13. Pressure (top) and Mach number contours for inviscid flow.

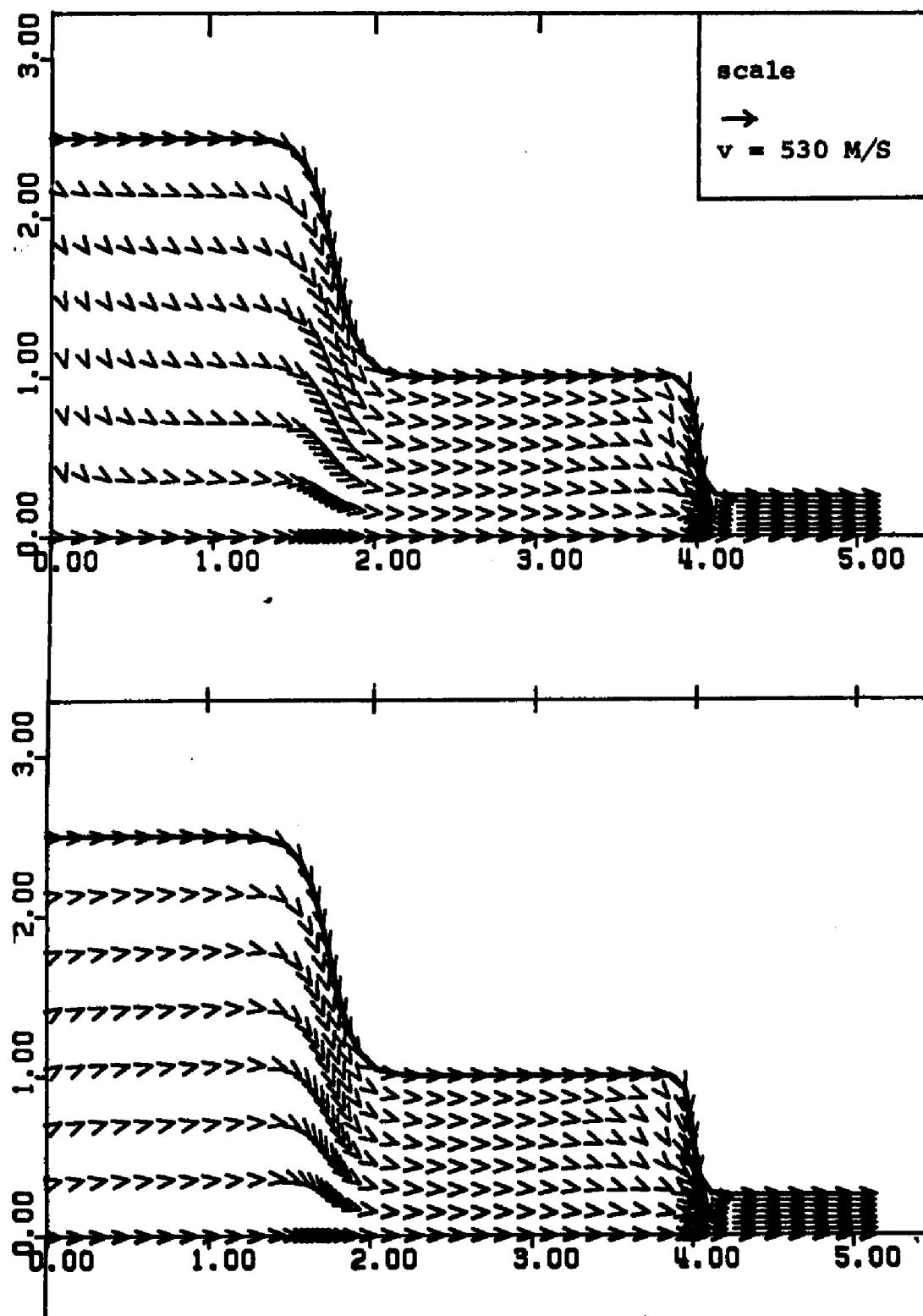


Figure 14. Velocity vector plots of the flow in the discharge chamber used in Pakarat's experiment

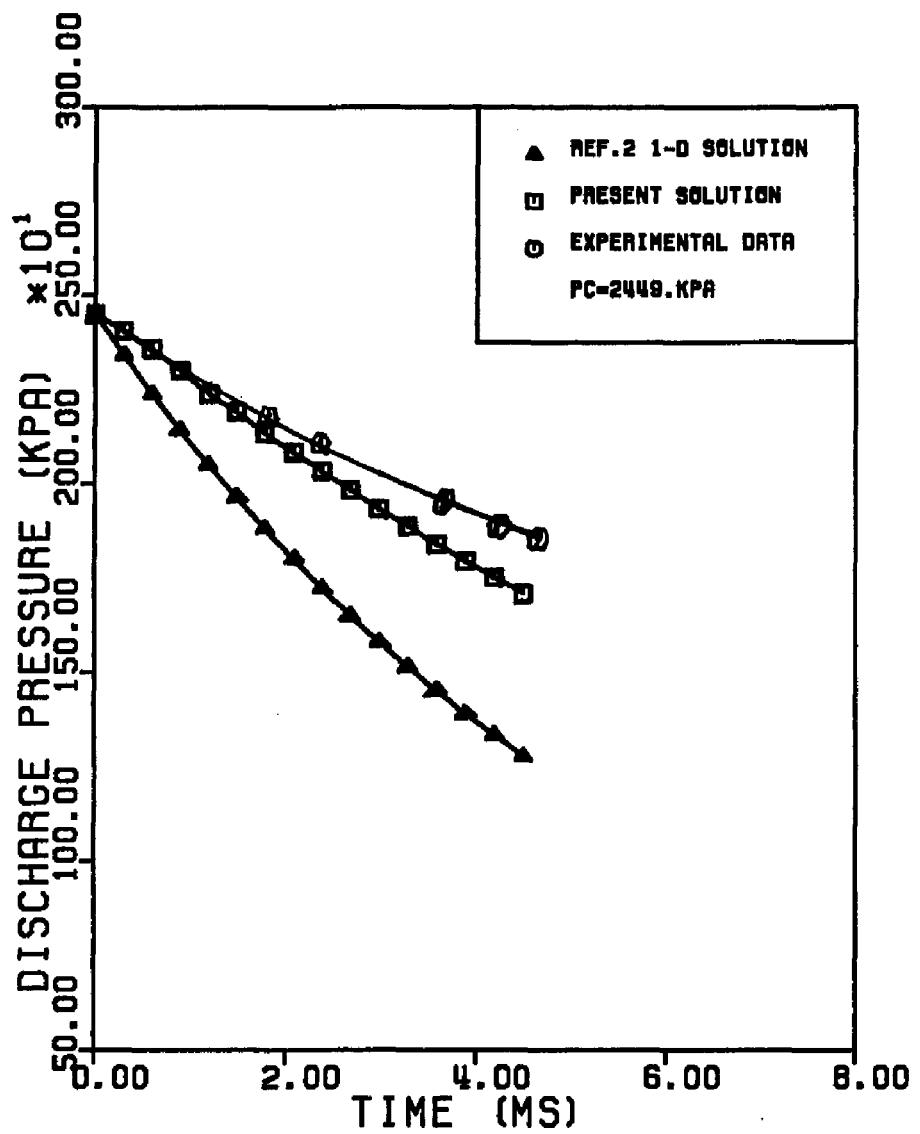


FIGURE 15. DISCHARGE PRESSURE VS TIME

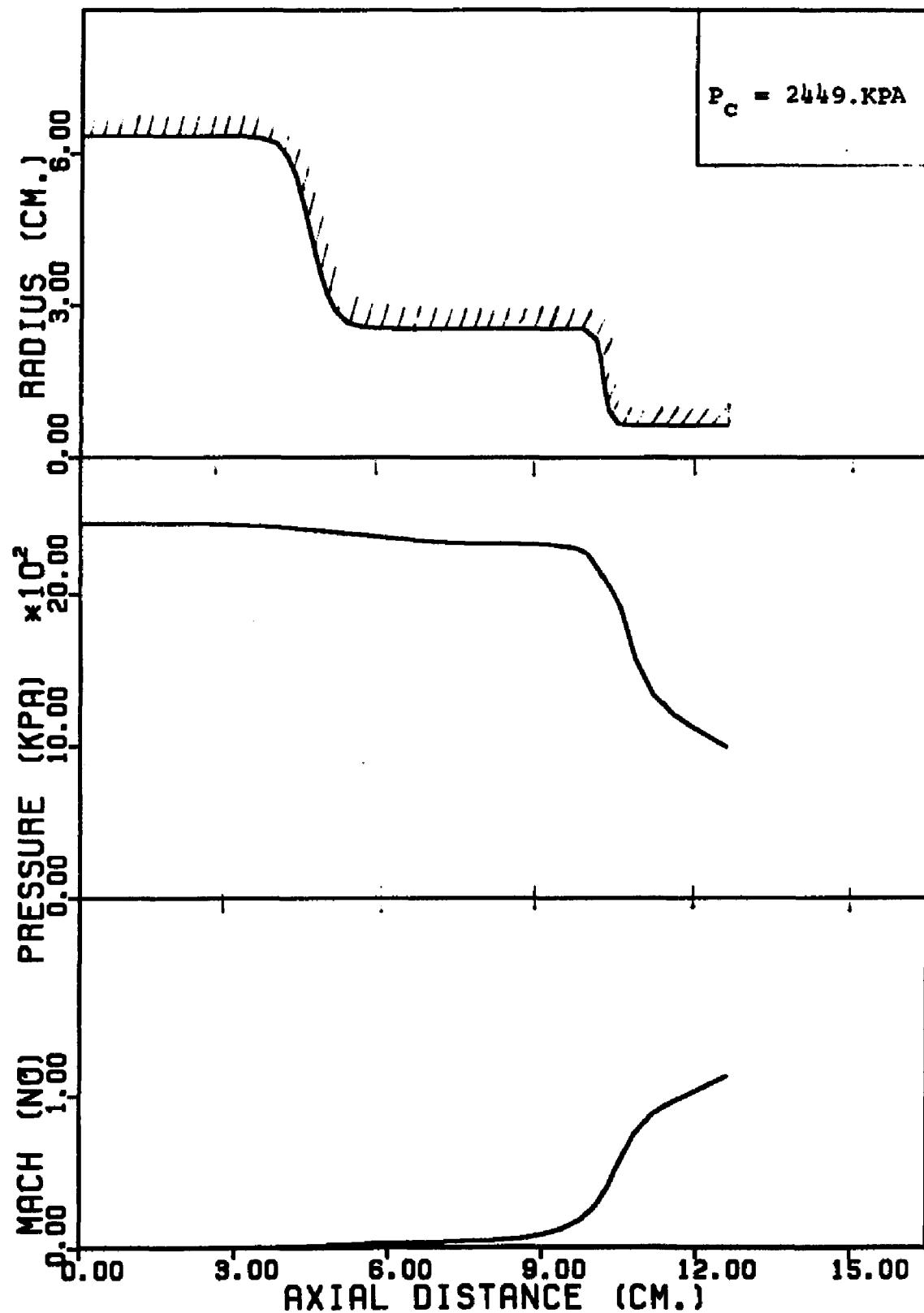


Figure 16. Discharge chamber geometry, and midplane pressure and Mach number for inviscid flow.

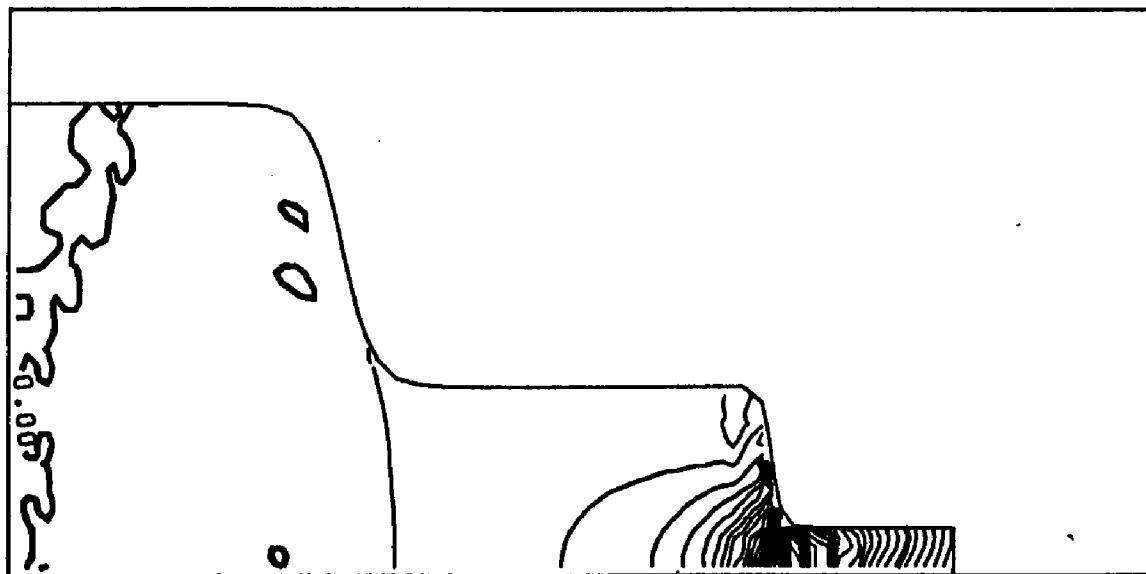
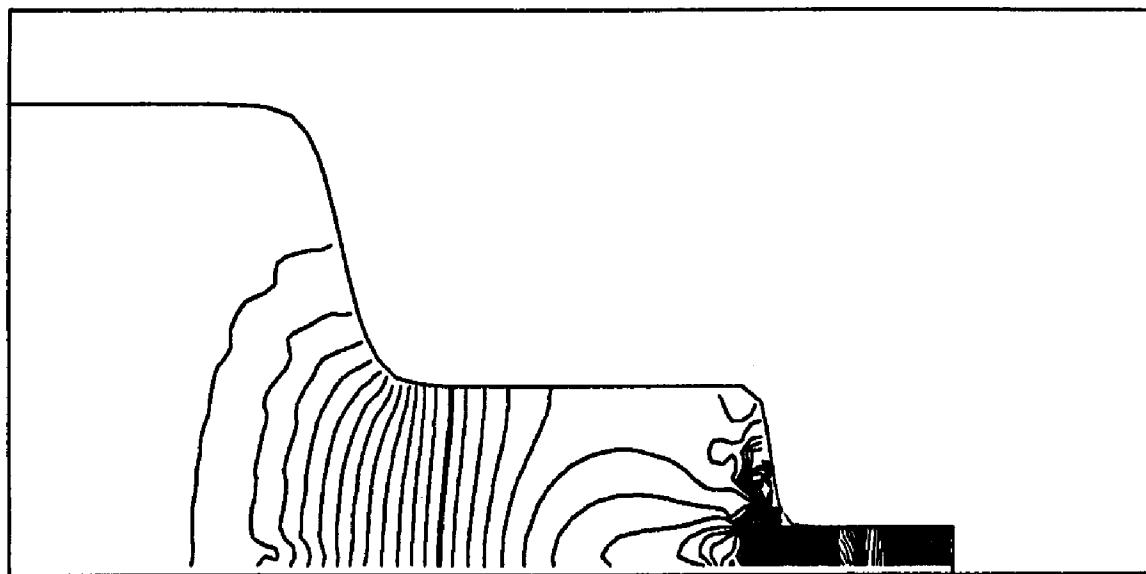


Figure 17. Pressure (top) and Mach number contours for inviscid flow.

B. BARREL CHAMBER FLOW

1. Governing Equations

The mathematical model simulates the gas flow through a chamber confined on one side by a moving boundary. An application of this model is in the barrel chamber of a gun where the moving boundary is the projectile. The model assumes a one dimensional tube flow, because the axial component of flow there is one of principal interest. Therefore, the flow properties depend only on the axial coordinate x and the time t . The model takes into account the effect of gas friction. However it neglects the effect of turbulence. The governing equations for this flow can be written as,

$$\frac{\partial \rho}{\partial t} + u \frac{\partial \rho}{\partial x} + \rho \frac{\partial u}{\partial x} = \Lambda \quad (37)$$

$$\frac{\partial u}{\partial t} + u \frac{\partial u}{\partial x} + \frac{1}{\rho} \frac{\partial \rho}{\partial x} = F \quad (38)$$

$$\frac{\partial P}{\partial t} + u \frac{\partial P}{\partial x} - a^2 (\frac{\partial \rho}{\partial t} + u \frac{\partial \rho}{\partial x}) = Y \quad (39)$$

$$\frac{P}{\rho} = RT, \quad a^2 = \gamma P/\rho \quad (40)$$

In the above equations the dependent variables are the cross-sectional-averages of density ρ , axial velocity u ,

the general term, Λ , contains the effect of mass addition and area change for the quasi-one dimensional flow. The general term, F , contains the effect of friction. The general term, Ψ , contains the effect of heat transfer and viscous dissipation.

2. Numerical Method

The mathematical model discussed in the previous section consists of a system of quasi-linear non-homogeneous partial differential equations of the first order. A quasi-linear partial differential equation of the first order is one that may be non-linear in the dependent variables but is linear in the first order partial derivatives. The system of governing Equations (37) - (39) is hyperbolic. There are several numerical methods which are applicable to this mathematical model. One of the most accurate numerical techniques for solving hyperbolic partial differential equations is the method of characteristics. In the present investigation the method of characteristics is used. A complete numerical algorithm can be developed. The development presented herein closely follows the technique presented by Zuckow and Hoffman¹⁰. A characteristic is defined mathematically as a curve along which the governing partial differential equation reduce to an interior operator; that is, a total differential equation known as the compatibility equation.

The dependent variables may not be specified arbitrarily

on a characteristic curve, but they must be compatible with the interior operator. From a physical point of view, a characteristic is defined as the path of propagation of a physical disturbance.

2.1 Characteristic and Compatibility Equations

The equations specifying the characteristic curves and the corresponding compatibility equations can now be written. There are three families of characteristic curves, two families of Mach lines, and a family of pathlines.

Along the pathline

$$dx = u dt \quad (41)$$

$$dP - a^2 d\rho = \Psi dt \quad (42)$$

Along the Mach lines

$$a) \quad dx = (u + a) dt \quad (43)$$

$$dP + \rho a du = (\Psi + a^2 \Lambda + \rho a F) \quad (44)$$

$$b) \quad dx = (u - a) dt \quad (45)$$

$$dP - \rho a du = (\Psi + a^2 \Lambda - \rho a F) \quad (46)$$

These ordinary differential equations are derived in Appendix A.

2.2 Numerical Integration Scheme

The characteristic and compatibility Equations (41) - (45) are ordinary differential equations. These ordinary differential equations may be integrated by simple numerical integration techniques which are applicable to ordinary

differential equations, such as Runge-Kutta, Taylor, Euler etc. The second order modified Euler predictor-corrector numerical integrator can be used for the integration of the compatibility equations. The basic features of this method are summarized below.

Consider the ordinary differential equation,

$$dy = f(x, y) dx \quad \text{with initial conditions } (x_0, y_0) \quad (47)$$

The integration of Equation (46) can be obtained in two steps.

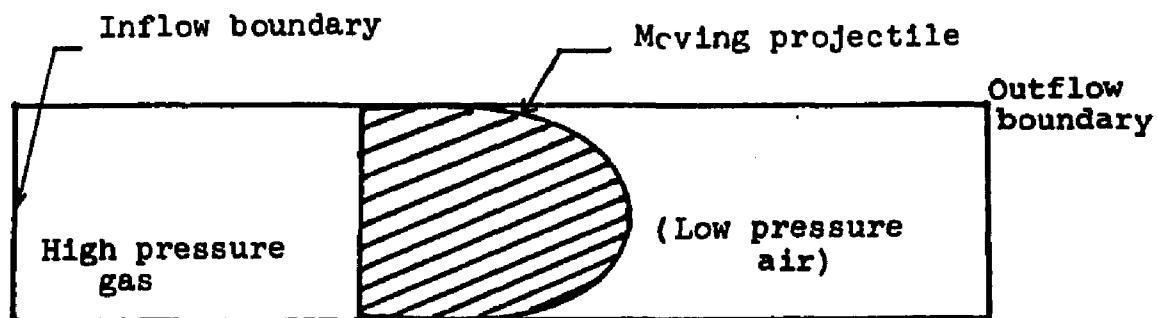
For the first step we have,

$$y_{i+1}^P = y_i + f(x_i, y_i) \Delta x \quad (48)$$

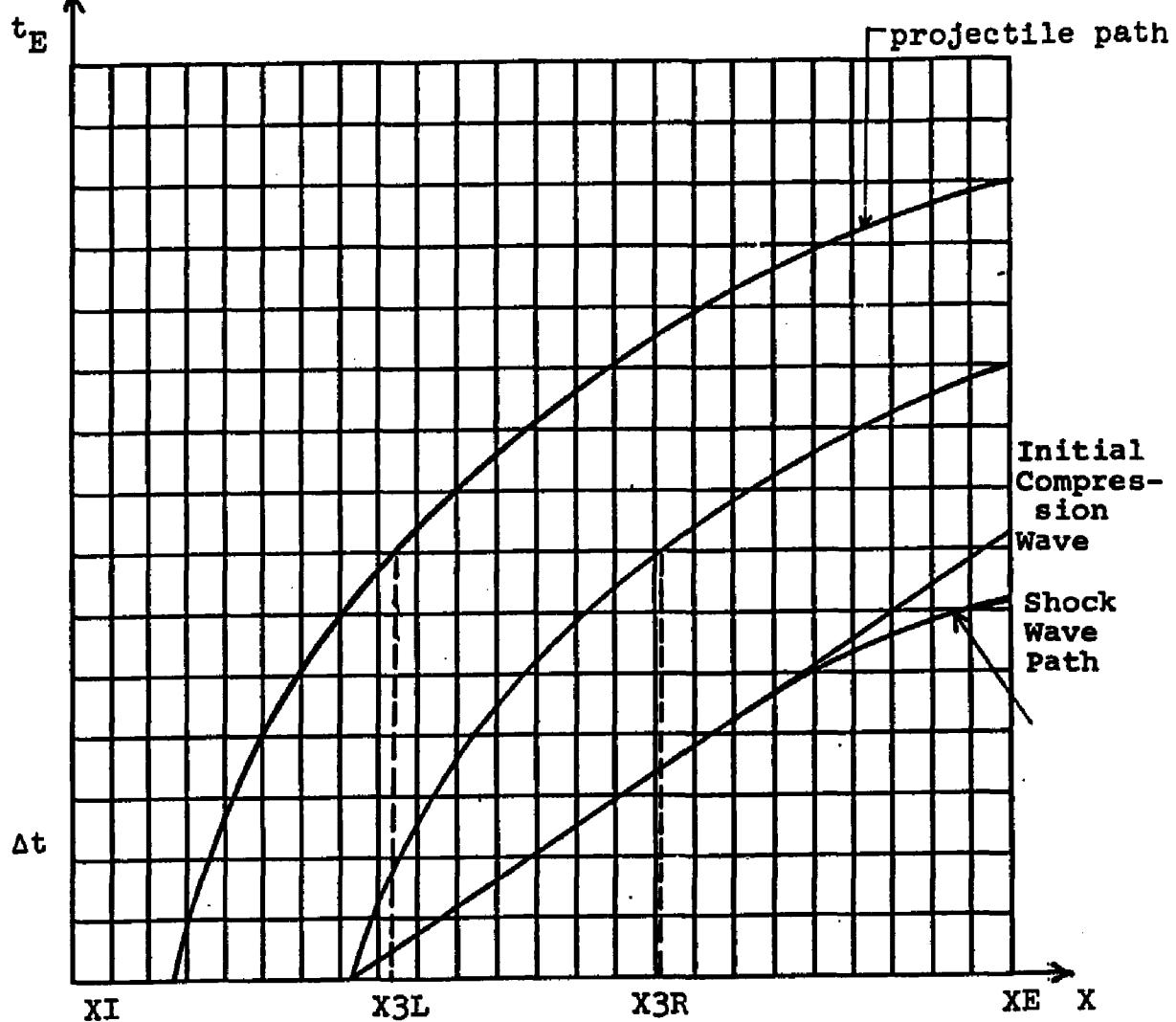
For the second step (with iterations) we have

$$y_{i+1}^C = y_i + f\left(x_i + \frac{\Delta x}{2}, \frac{y_i + y_{i+1}^P}{2}\right) \Delta x \quad (49)$$

Two different types of overall marching algorithm may be constructed in the numerical method of characteristics, the direct marching method in which the characteristics are projected forward (in time) to determine the location of the solution point, and the inverse marching method in which the solution points are prespecified in some manner, usually on a rectangular grid, and the characteristics are projected rearward (in time) to determine the initial-data points. The latter method is employed in the numerical integration schemes presented in this work. Figure 18 shows a typical computational grid. The solution is obtained on lines of constant t . The physical



a) physical plane



b) Computational flow plane.

Figure 18 . Physical and computational flow planes.

plane is subdivided into a number of fixed grid points. The time step Δt between successive solutions lines is chosen to satisfy the CFL stability criterion developed by Courant, Friedrichs, and Lewy. The CFL stability criterion states that the differential domain of dependence of the solution point must be contained within the finite difference domain of dependence of the solution point. This numerical integration scheme is, therefore, conditionally stable. The physical interpretation of this stability criterion is shown in Figure 19. In that figure the solution point (4) is influenced by the solution of each of the grid points at previous time steps contained within the two diagonals (4A) and (4C). Thus the region (4AC4) is the domain of dependence of point (4) in the finite difference domain.

If (4-1) and (4-2) are the backward characteristics of slope $(dx/dt)_{\pm} = u \pm a$ passing through the solution point (4), the CFL stability requires that line segment(1-2) fall completely within line segment (AC), as shown in Figure 19. However, from the theory of characteristics it is known that point (4) can receive signals only from the region 4-1-2-4 which is the domain of dependence of the differential equation, that is, the physical domain of dependence of the point (4). The computational grid shown in Figure 18 is divided into 5 sets of mesh points.

(1) interior grid points

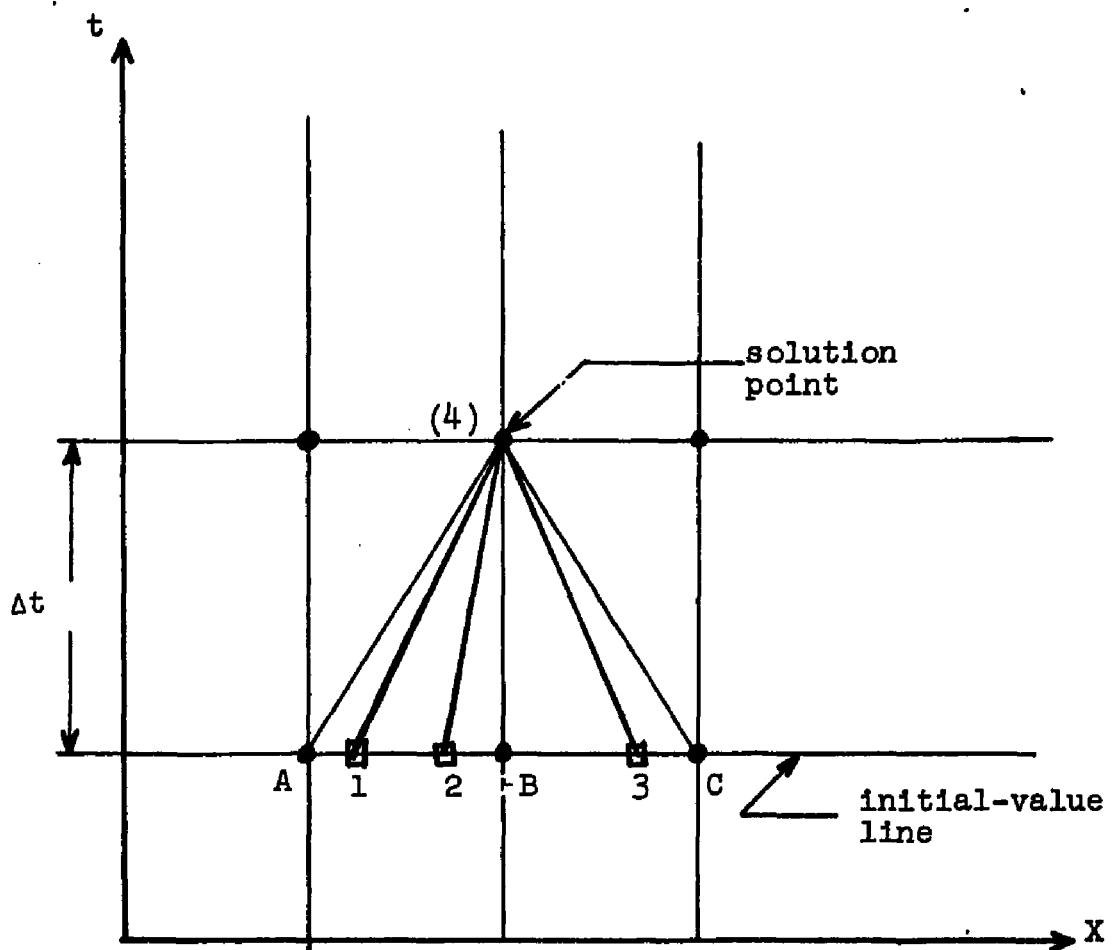


Figure 19. Physical interpretation of the CFL stability criterion.

- (2) inflow grid point
- (3) outflow grid point
- (4) projectile grid point
- (5) shock wave grid point

A numerical algorithm for each type of mesh point can now be presented.

2.2.1 Interior Grid Point

The finite difference grid for the interior point is shown in Figure 19. The flow properties at the solution point (4) are determined by numerically integrating the compatibility equations, Equations (40) - (46), along the pathline and the Mach lines, respectively.

Equations (40) - (45) are written in finite difference form. The differentials du , $d\rho$ and dP are replaced by differences along the characteristic curves.

These can be written as follows:

$$(P_4 - P_3) - \overline{(a^2)}_{34} (P_4 - P_3) = \overline{\Psi}_{34} \Delta t \quad (50)$$

$$(P_4 - P_1) + \overline{(\rho a^2)}_{14} (u_4 - u_1) = \overline{(\rho a F + a^2 \Lambda + \Psi)}_{14} \Delta t \quad (51)$$

$$(P_4 - P_2) - \overline{(\rho a^2)}_{24} (u_4 - u_2) = \overline{(-\rho a F + a^2 \Lambda + \Psi)}_{24} \Delta t \quad (52)$$

where the overbar denotes average values along the characteristic curves. If the locations and the flow properties at the points (1), (2), and (3) are known, then the above equations can be solved simultaneously for u_4 , ρ_4 , and P_4 . The locations of these points are obtained by numerically

integrating the characteristic curves passing through the solution point (4).

These can be written as follows,

$$(x_4 - x_3) = \overline{u_{34}} \Delta t \quad (53)$$

$$(x_4 - x_1) = (\overline{u+a})_{14} \Delta t \quad (54)$$

$$(x_4 - x_2) = (\overline{u-a})_{24} \Delta t \quad (55)$$

The initial values of $\overline{u_{34}}$, $(\overline{u+a})_{14}$, and $(\overline{u-a})_{24}$ are set equal respectively to u_B , $u_A + a_A$, $u_C - a_C$. Then an iterative technique is used to determine the locations and the flow parameters as follows,

Predictor algorithm.

step 1 Calculate x_1 , x_2 , and x_3 from Equations (53) - (55)

step 2 Determine the flow properties at each point found in step 1 by using a simple linear interpolation.

step 3 Set $\overline{u_{34}} = u_3$, $(\overline{u+a})_{14} = u_1$, and $(\overline{u-a})_{24} = u_2$.

step 4 Repeat steps 1-3 until the changes in locations are within a given tolerance.

step 5 Solve the system of equations (50) - (52) for u_4 , p_4 , and P_4 .

At this point the Euler predictor algorithm is complete.

This algorithm may be repeated a given number of times if desired by the corrector algorithm.

Corrector algorithm.

The corrector algorithm employs the predictor algorithm by

taking average values along each characteristic curve.

That is,

$$\overline{U}_{34} = (u_3 + u_4)/2,$$

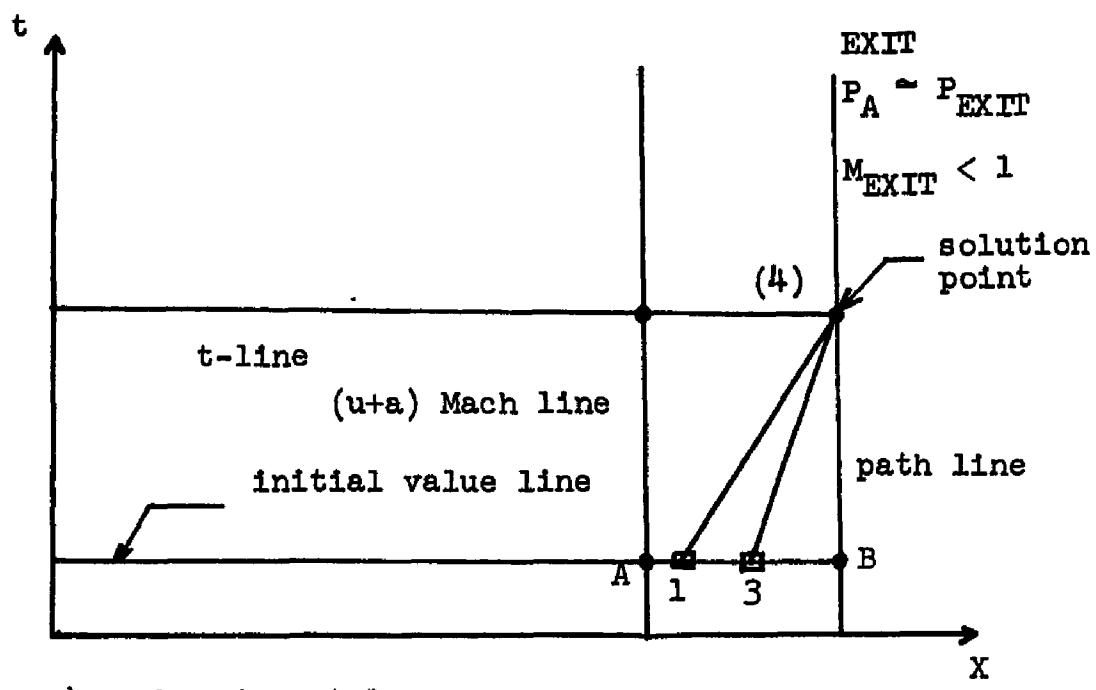
$$\overline{(u+a)}_{14} = \frac{(u_1 + u_4 + a_1 + a_4)}{2}, \text{ and}$$

$$\overline{(u-a)}_{24} = \frac{(u_2 + u_4 - a_2 - a_4)}{2}.$$

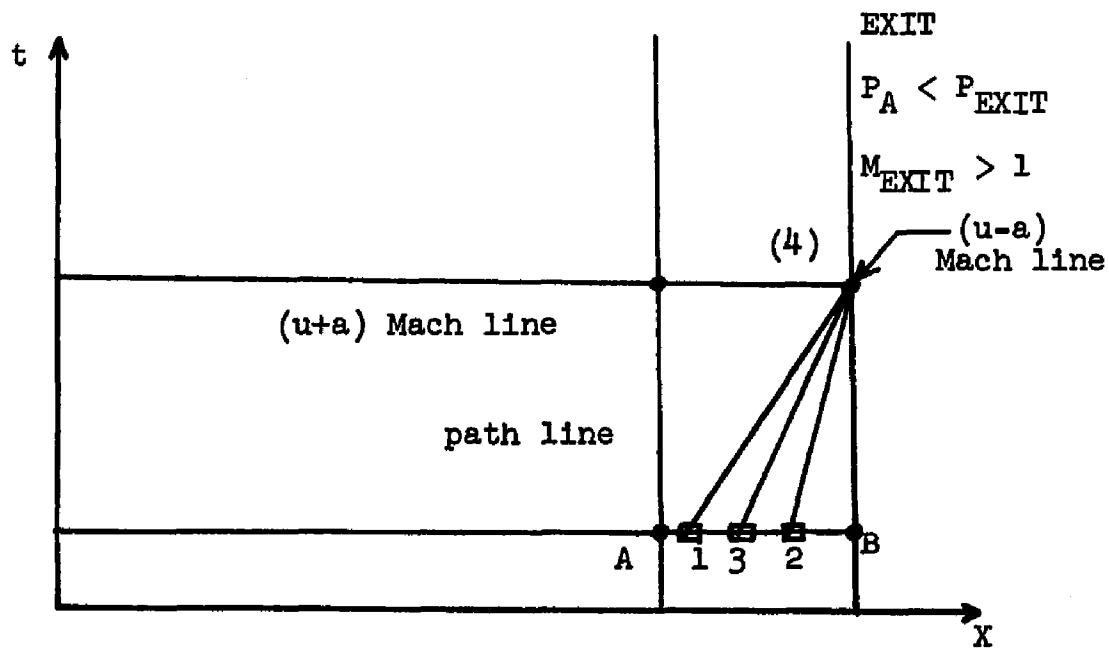
2.2.2 Outflow Grid Point.

The flow at the exit of the barrel can be a supersonic outflow or a subsonic outflow. For supersonic outflow, the finite difference grid is shown in Figure 20 b. Both characteristics originate within the physical domain. Therefore, point (4) can be considered as an interior point. In this case a reasonable approximation is to extrapolate from the interior values, since the error generated from the extrapolation can not affect the upstream conditions. For subsonic outflow, the finite difference is shown in Figure 20a. In this case the boundary condition is the specification of the ambient pressure, $P_4 = P_A$. The remaining flow properties are determined by numerically integrating the compatibility equations, Equations (42) and (44), along the path line and the Mach line, respectively.

The finite difference equations of Equations (42) and (44) can be written as follows:



a) subsonic outflow



b) supersonic outflow

Figure 20. Finite difference grid for an outflow boundary.

$$p_4 = p_A$$

$$(x_4 - x_1) = (\bar{u} + a)_{14} \Delta t \quad (56)$$

$$(x_4 - x_3) = (\bar{u})_{34} \Delta t \quad (57)$$

$$(p_4 - p_3) - (a^2)_{34} (p_4 - p_3) = \bar{\Psi}_{34} \Delta t \quad (58)$$

$$(p_4 - p_1) + (\rho a^2)_{14} (u_4 - u_1) = (\rho a F + a^2 \Lambda + \Psi)_{14} \Delta t \quad (59)$$

Using the numerical algorithm described in section 2.2.1. the properties at the solution point (4) can be obtained.

2.2.3 Inflow Grid Point

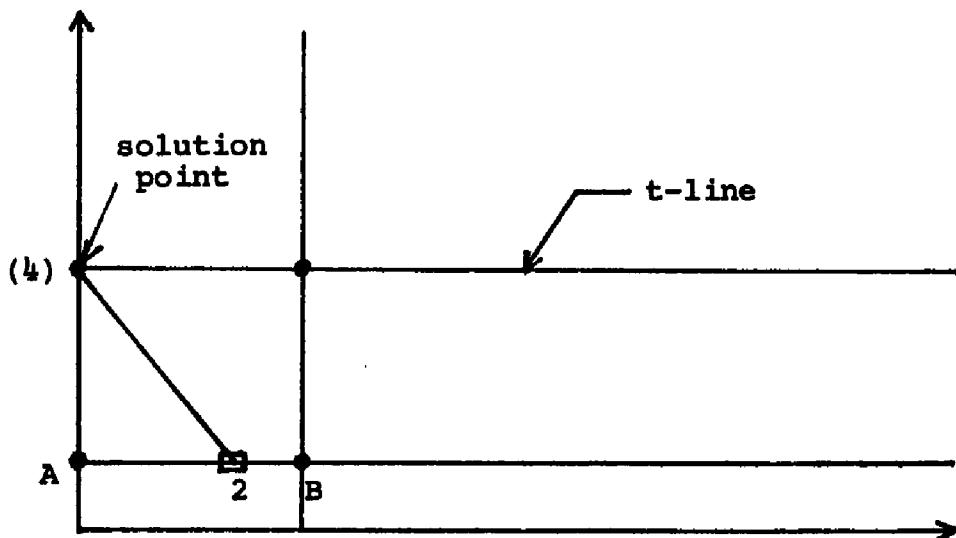


Figure 21. Finite difference grid for a subsonic inflow

The flow at the barrel entrance can be a supersonic inflow or a subsonic inflow. For supersonic inflow all flow parameters must be specified. For subsonic inflow,

the finite difference grid is shown in Figure 21 . In this case the specified flow parameters are the density and the velocity, i.e., u_4 and ρ_4 . The pressure, P_4 , can be determined by numerically integrating the compatibility equation, Equation (46), along the Mach line, Equation (45). In finite difference form this can be written,

$$(x_4 - x_2) = (\overline{u - a})_{24} \Delta t \quad (60)$$

$$(P_4 - P_3) - (\overline{\rho a^2})_{24} (u_4 - u_2) = (\overline{-\rho a F + a^2 \Lambda + \Psi})_{24} \Delta t \quad (61)$$

P_4 is then calculated by the algorithm presented in Section 2.2.1.

2.2.4 Projectile Grid Point

The finite difference grid for a moving projectile is shown in Figure 22 . The unspecified parameters are the flow properties at the left hand side and the right hand side of the projectile. The number of unknowns is equal to five. Five equations can now be written.

-Along the $(u + a)$ -Mach line

$$dP + \rho a^2 du = (\rho a F + a^2 \Lambda + \Psi) dt \quad (62)$$

-Along the path line

$$dP - a^2 d\rho = \Psi dt \quad (63)$$

At the right hand side of the projectile we have,

-Along the $(u - a)$ -Mach line

$$dP - \rho a^2 du = (\Psi + a^2 \Lambda - \rho a F) dt \quad (64)$$

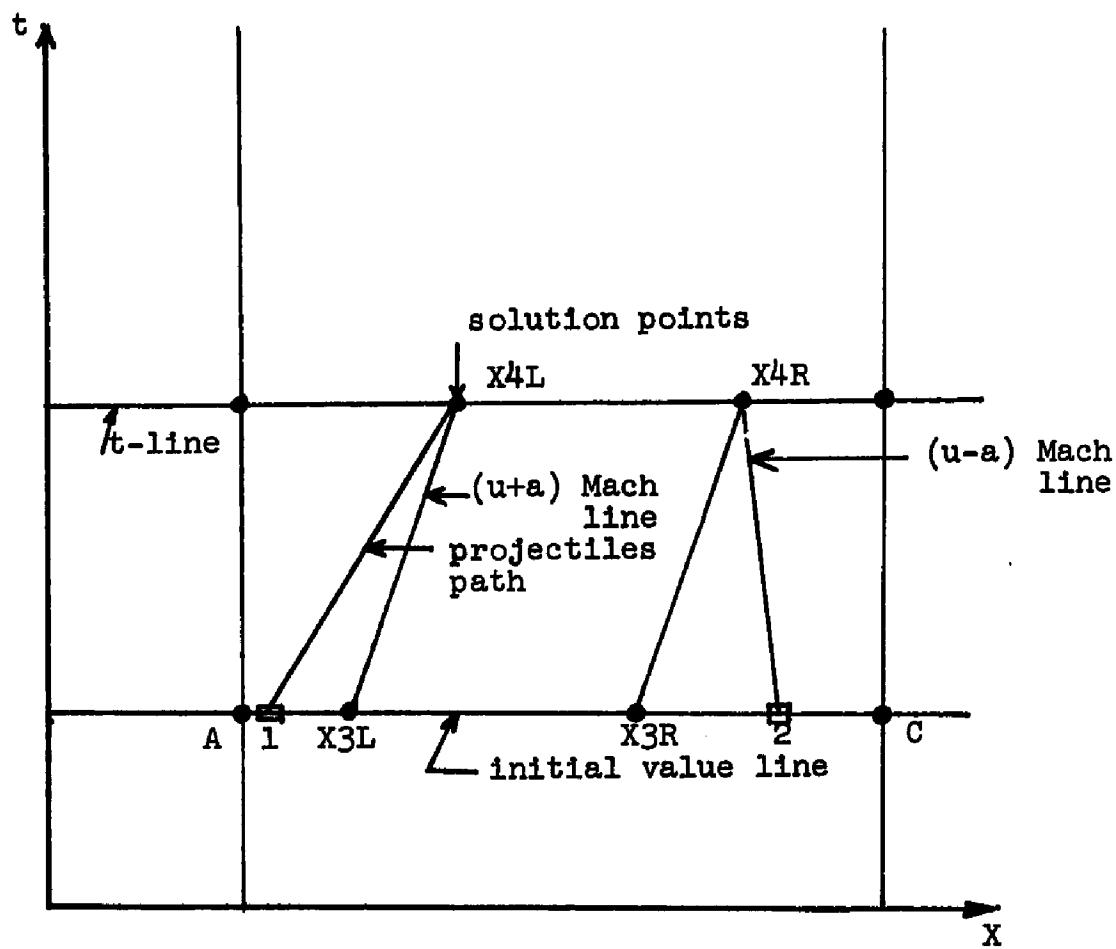


Figure 22 . Finite difference grid for a moving projectile.

-Along the path line

$$dP - a^2 d\rho = \gamma dt \quad (65)$$

Projectile's Motion.

$$du_s - [(P_{3L} - P_{3R})A_s - D_s] dt/m_s = 0 \quad (66)$$

In Equation (62) m_s is the projectile mass, P_{3L} is the pressure at the left hand side of the projectile, P_{3R} is the pressure at the right hand side of the projectile, A_s is the projectile's cross sectional area, and D_s is the friction force.

Equations (62) - (66) are written in finite difference form by the same methods presented in Section 2.2.1. Then they are solved simultaneously for u_4 , P_{4L} , R_{4L} , R_{4R} , and P_{4R} .

2.2.5 Shock Wave Grid Point

The finite difference grid for a shock wave point is shown in Figure 23. The flow properties are discontinuous at any shock wave point. However the governing equations (37) - (40) fail to apply across discontinuities. The shock wave will be treated as a moving boundary separating two regions of continuous flow. The boundary is defined by two points, one the first computational region (shown in Figure 23a. as X_{3L}) and one the second (shown in Figure 23b. as X_{3R}). The unknown flow parameters are the values of velocity, density, and pressure on the left-hand side and on the right-hand side of

the projectile, and the position and velocity of the shock at every time step. This problem is solved by combining the Rankine-Hugoniot equations and the method of characteristics as follows:

1. Detection of a shock wave.

A shock wave is detected if the pressure gradient, $\frac{\partial P}{\partial x}$, becomes infinite. However, such a detection is not possible. Instead, a shock is assumed to appear when $\frac{\partial x}{\partial P}$ vanishes. For this problem the prediction of the shock is given analytically by Moretti¹¹, and also by Thompson¹². This can be written as,

$$t_s = \frac{2a_0}{(\gamma + 1) \ddot{x}_s(0)} \quad (67)$$

$$x_s = a_0 t_s, \quad w_s = a_0 \quad (68)$$

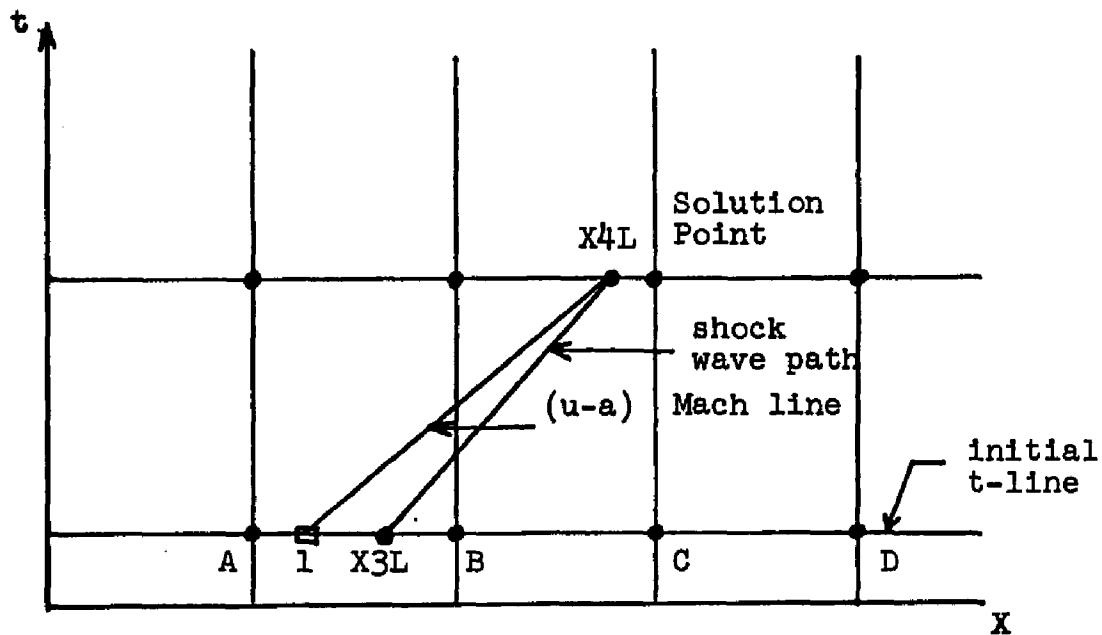
where x_s , w_s , $\ddot{x}_s(0)$, and a are, respectively, the position of the shock, velocity of the shock, initial acceleration of the projectile, and initial speed of sound.

2. Use of an iterative algorithm to solve for the flow properties.

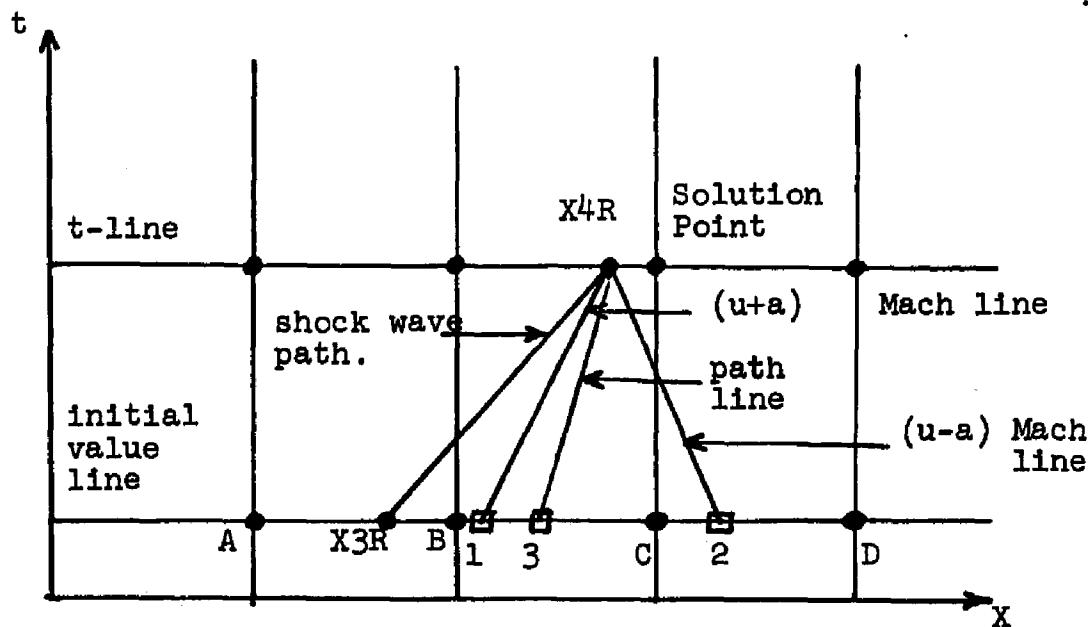
step 1 Assume velocity of shock and gas in the right-hand side of the shock.

step 2 Determine the location of the shock.

step 3 Use the numerical algorithm for the interior point to determine the flow properties at point
(see Figure 23b)



a) left-hand side of the shock wave



b) right-hand side of the shock wave

Figure 23 . Finite difference grid for a shock wave point.

- step 4 Use Rankine-Hugoniot Equations across the shock wave.
step 5 Determine the flow properties on the left-hand side of the projectile by numerically integrating the compatibility Equation (44).

3. Results

Figures 24 through 27 show the numerical results obtained from the mathematical model used for the barrel flow. Projectile displacement, velocity, and pressure as functions of time are shown for three different cases. In the first case, the charge pressure, P_c , is set equal to 8,000.KPA. From Figure 24, it is seen that the projectile reaches the end of the barrel at time, $t = 8.1$ ms. At this time all calculations were stopped. Projectile muzzle velocity is obtained from Figure 24, (velocity = 400 m/s) at time $t = 8.1$ ms. The pressure at the base of the projectile versus time is also shown in Figure 24. Similar plots are shown in Figure 25. In this case the charge pressure is set equal to 5,000.KPA. The muzzle velocity obtained, is equal to 300 m/s and the time it occurs is, $t = 10$ ms.

Finally, in the third case, the charge pressure is set equal to 50,000 KPA. A shock is predicted to occur inside the barrel. The shock is assumed to form at time, $t = 0$, and at the origin, $x = 0$. From Figure 26, it is seen that the projectile leaves the tube with a muzzle velocity of 700 m/s at time, $t = 4$ ms. However referring to Figure 27, it is seen that, the shock reaches the barrel in less than 2.5 ms.

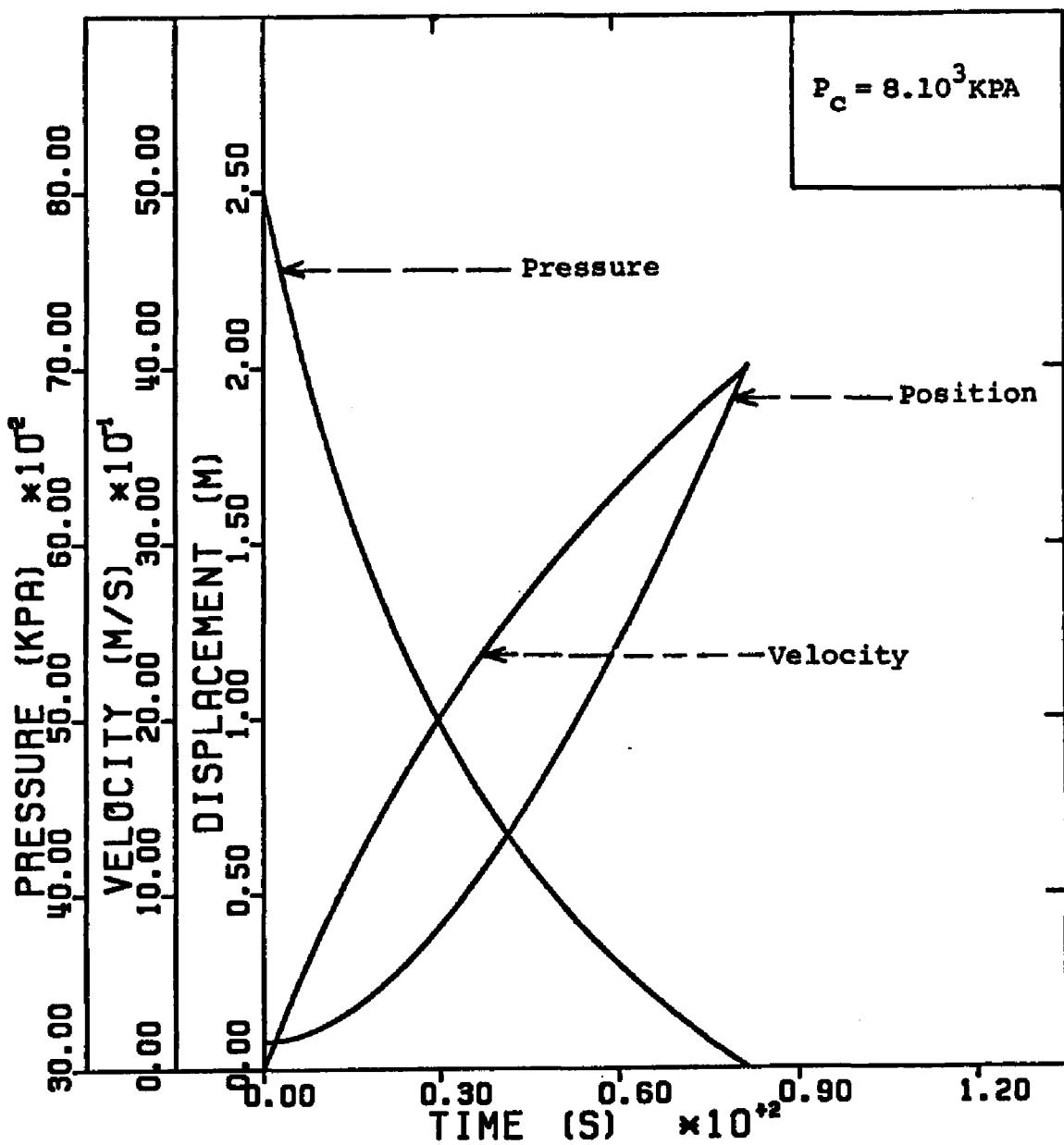


Figure 24. Projectile base pressure, velocity, and displacement as a function of time.

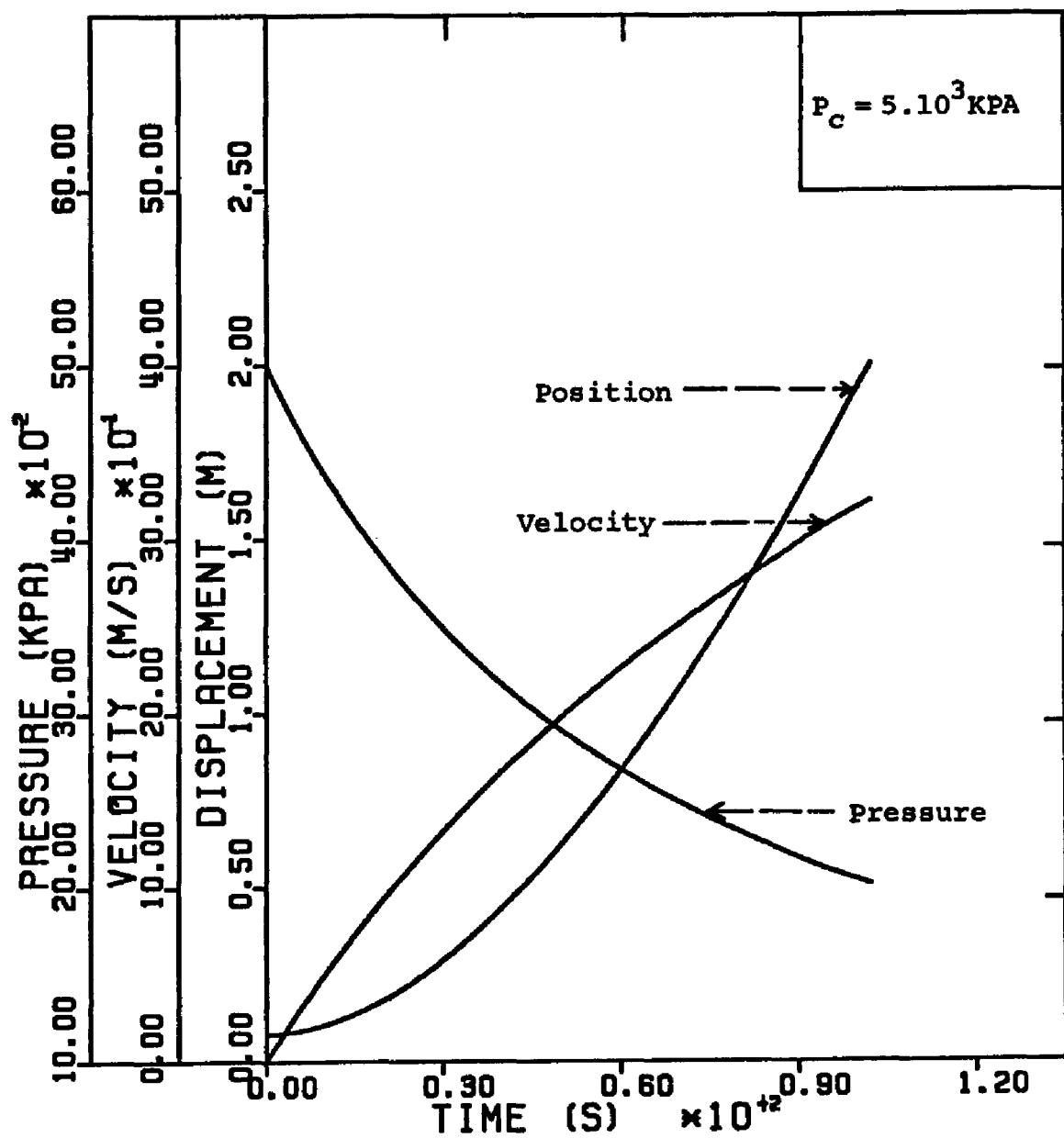


Figure 25. Projectile base pressure, velocity, and displacement as a function of time.

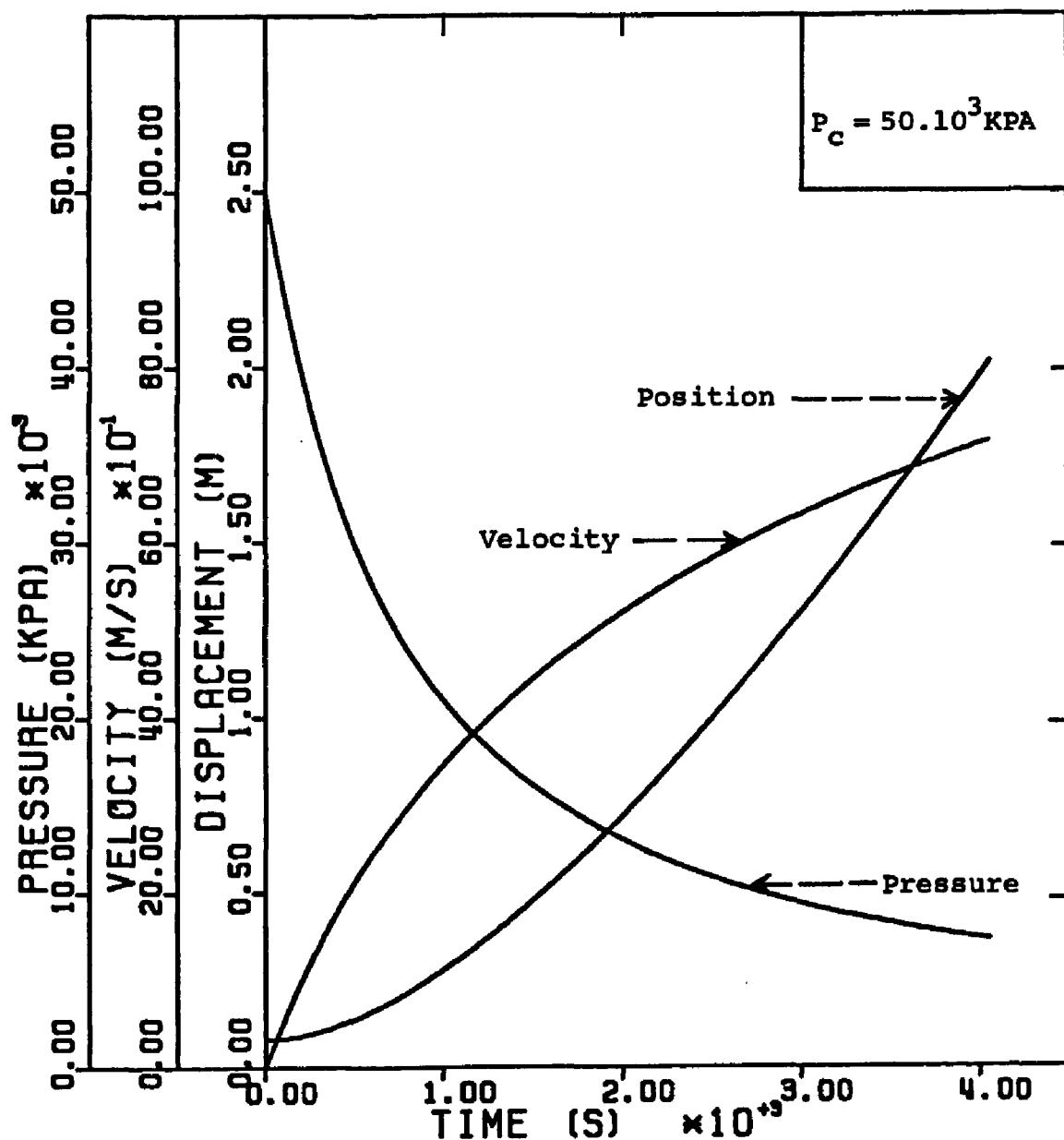


Figure 26. Projectile base pressure, velocity, and displacement as a function of time.

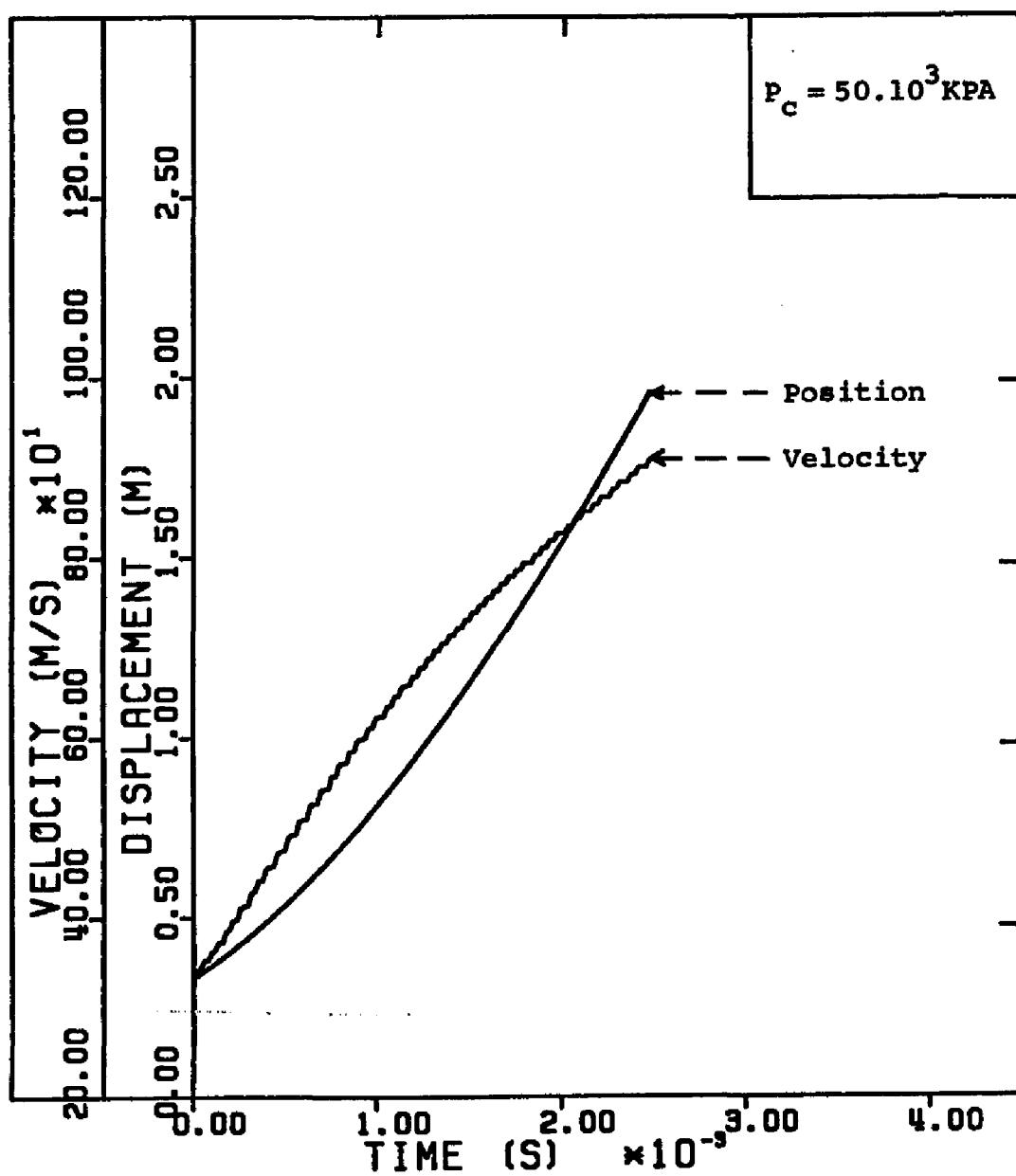


Figure 27. Shock position and velocity as a function of time.

C. MAIN CHAMBER FLOW

The flow in the main chamber is treated as a two-dimensional, time dependent flow. The governing equations, Equations (1) - (4), presented in Part A are then applicable for this case. However the boundary conditions and the geometry are more complex, see Figure 28 . The left and the right boundaries are both time dependent. The gas flows from the main chamber to the barrel through a time dependent area. Therefore, the model should be able to generate not only a variable grid but a time dependent one. Initially there is a very large pressure gradient at the barrel entrance. For this purpose an explicit artificial viscosity is used to stabilize the numerical method for shock wave calculations.

1. Physical and Computational Flow Spaces.

The physical space shown in Figure 28 is mapped into a computational space with uniform grid spacing by means of the following transformations

$$\xi(x, t) = \int_0^x \sqrt{1 + \left(\frac{\partial f}{\partial s}\right)^2(s, t)} ds \quad (69)$$

$$\eta(Y) = \int_0^Y \sqrt{1 + \left(\frac{dg}{ds}\right)^2} ds \quad (70)$$

$$\tau = t \quad (71)$$

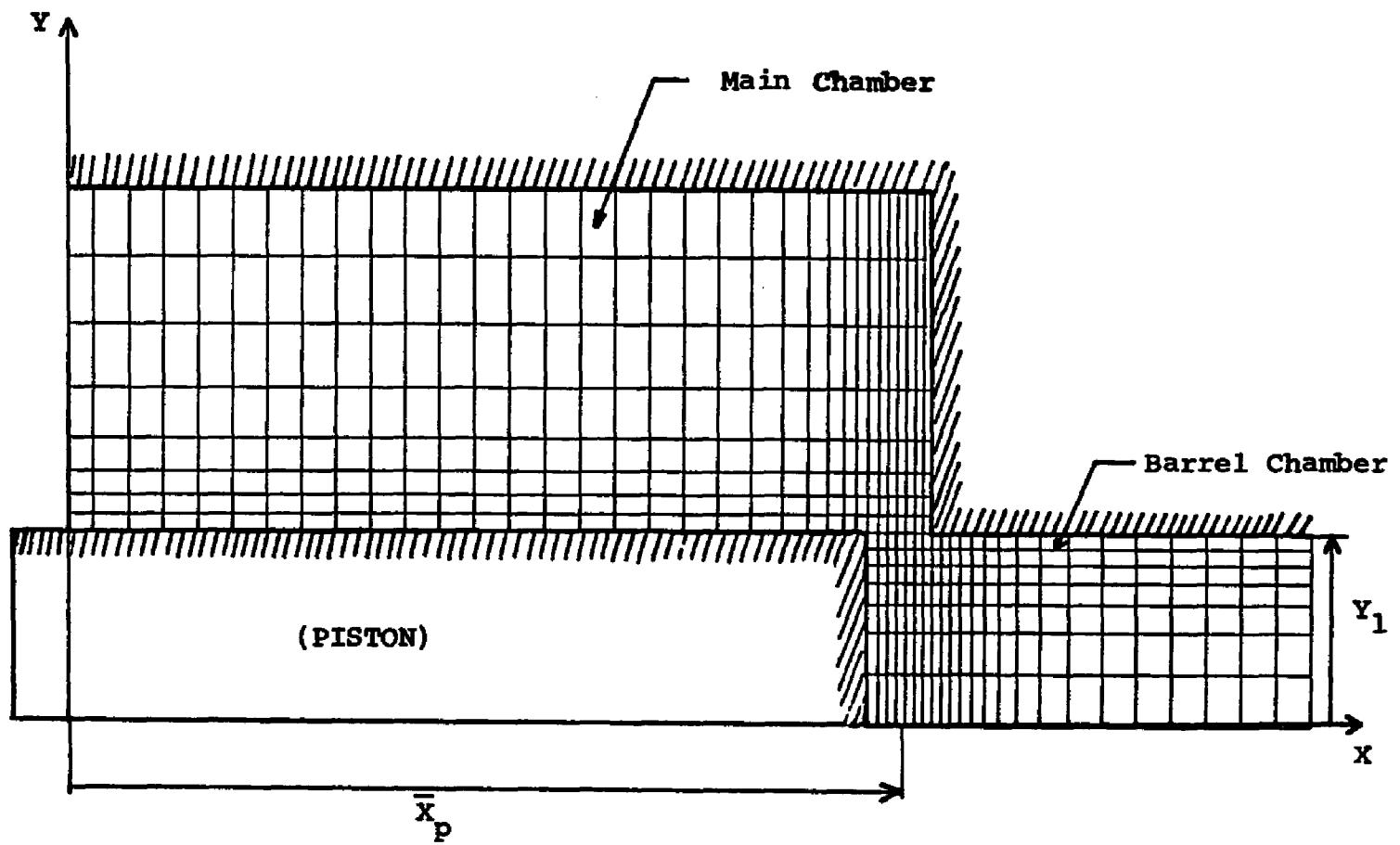


Figure 28 . Schematic illustration of the main chamber.

where,

$$f(x, t) = R_x \tanh(x - \bar{x}_p(t))$$

$$g(y) = R_y \tanh(y - Y_1)$$

R_x and R_y are parameters that control the distribution of the grid. $(\bar{x}_p(t), Y_1)$ denotes the location of the point about which the grid is concentrated. Note that the first coordinate is a function of time.

The spatial and the time partial derivatives become,

$$\frac{\partial}{\partial x} = w_x \frac{\partial}{\partial \xi} \quad (73)$$

$$\frac{\partial}{\partial y} = w_y \frac{\partial}{\partial \eta} \quad (74)$$

$$\frac{\partial}{\partial t} = \frac{\partial}{\partial \tau} + \alpha \frac{\partial}{\partial \xi} \quad (75)$$

where $w_x = \sqrt{1 + \frac{R_x^2}{\cosh^4(x - \bar{x}_p(t))}}$, (76)

$$w_y = \sqrt{1 + \frac{R_y^2}{\cosh^4(y - Y_1)}} \quad , \quad (77)$$

and $\alpha = \int_0^x \frac{\partial f}{\partial s} [1 + (\frac{\partial f}{\partial s})^2]^{-\frac{1}{2}} \frac{\partial^2 f}{\partial s \partial t} ds$ (78)

2. Transformed Governing Equations

Using Equations (73) - (75), the governing equations, Equations (1) - (4), can now be written in the transformed space as follows:

$$\frac{\partial \rho}{\partial \tau} + u^* \frac{\partial \rho}{\partial \xi} + v^* \frac{\partial \rho}{\partial \eta} + \rho \omega_x \frac{\partial u}{\partial \xi} + \rho \omega_y \frac{\partial v}{\partial \eta} + \rho \frac{v}{y} = QROT \quad (79)$$

$$\frac{\partial u}{\partial \tau} + u^* \frac{\partial u}{\partial \xi} + v^* \frac{\partial u}{\partial \eta} + \frac{\omega_x}{\rho} \frac{\partial p}{\partial \xi} = QUT \quad (80)$$

$$\frac{\partial v}{\partial \tau} + u^* \frac{\partial v}{\partial \xi} + v^* \frac{\partial v}{\partial \eta} + \frac{\omega_y}{\rho} \frac{\partial p}{\partial \eta} = QVT \quad (81)$$

$$\frac{\partial p}{\partial \tau} + u^* \frac{\partial p}{\partial \xi} + v^* \frac{\partial p}{\partial \eta} - a^2 (\frac{\partial \rho}{\partial \tau} + u^* \frac{\partial \rho}{\partial \xi} + v^* \frac{\partial \rho}{\partial \eta}) = QPT \quad (82)$$

where,

$$u^* = \alpha + \omega_x U$$

$$v^* = \omega_y v$$

QROT = Artificial viscosity added to the Continuity Equation.

QUT = Viscous terms in the ξ -Momentum.

QVT = Viscous terms in the η -Momentum.

QPT = Viscous terms in the Energy Equation.

3. Numerical Method

The numerical methods used to integrate Equations (79) - (82) are similar to the numerical techniques described in Part A of Chapter II. That is, the computational

space is divided into two sets of mesh points, the interior grid points and the boundary grid points. The numerical algorithms are described in detail in Part A. In this case the MacCormack scheme is stable if it satisfies the following condition given in the transformed space,

$$\Delta t \leq \frac{A}{|u|\omega_x/\Delta\xi + |v|\omega_y/\Delta\eta + a\sqrt{\omega_x^2/\Delta\xi^2 + \omega_y^2/\Delta\eta^2}} \quad (83)$$

To determine the location of the piston, i.e. $\bar{x}_p(t)$, and the velocity at the left boundary. The dynamics of the piston motion is needed. This can be written as,

$$M_p \frac{dv_p}{dt} = \int_{A_p} (P_c - P_g) dA - D \quad (84)$$

where,

P_c = Pressure in the Main Chamber

P_g = Pressure in the Discharge Chamber

M_p = Piston Mass

A_p = Piston Area

D = Frictional Drag

The finite difference approximation to Equation (84) is used to update the velocity and the displacement of the piston so that the grid can be redistributed according to the piston displacement, i.e. $\bar{x}_p(t)$. Recall that $\bar{x}_p(t)$ is the coordinate of the point about which the grid

is concentrated. This can be written as,

$$v_p^{N+1} = v_p^N + \frac{\Delta t}{M_p} (\bar{P}_c - \bar{P}_g) A_p \quad (85)$$

$$x_p^{N+1} = x_p^N + v_p^N \Delta t + \frac{\Delta t^2}{2M_p} (\bar{P}_c - \bar{P}_g) A_p \quad (86)$$

The overbar denotes the average values.

Now it is important to describe the artificial smoothing devices which can be used to stabilize the numerical method. To stabilize the numerical method for shock wave calculations, an explicit artificial viscosity is included. This can be written as,

$$\begin{aligned} QROT &= \frac{C_p}{\rho} [w_x \frac{\partial}{\partial \xi} (\mu_A w_x \frac{\partial \rho}{\partial \xi}) + w_y \frac{\partial \rho}{\partial \xi} (\mu_A w_y \frac{\partial \rho}{\partial \eta} \\ &+ \frac{\mu_A}{y} w_y \frac{\partial \rho}{\partial \eta})] \end{aligned} \quad (87)$$

$$\text{where } \mu_A = \frac{C_\mu C_C \lambda \Delta \xi \Delta \eta \rho}{w_x w_y} [w_x \frac{\partial u}{\partial \xi} + w_y \frac{\partial v}{\partial \eta} + \frac{v}{y}] \quad (88)$$

C , C_μ , $C\lambda$, and C_p are constants.

These artificial quantities, $QROT$ and μ_A , are included into the model only when significant compressions are going to occur. The prediction of shock waves is detected by checking at each cell the divergence of the velocity. If it is less than zero, a compression wave is indicated. The computation is then automatic. If the divergence is greater than zero, then the artificial viscosity is set equal to zero. Other smoothing devices, fourth order

damping terms and relaxation, are also used. It is very helpful to retain the fourth order damping terms to the second order MacCormack scheme in order to eliminate instabilities. Since discontinuities and errors made in the initial data can propagate and cause the solution to diverge. A fourth order term can be written as,

$$\frac{\partial^4 Q}{\partial t^4} = C_1 \frac{Q_{L+2} - 4Q_{L+1} + 6Q_L - 4Q_{L-1} + Q_{L-2}}{(\Delta t)^4} \quad (89)$$

where Q is a dependent variable. The dependent variables can also be smoothed by the following formula:

$$Q_{L,M} = SMP Q_{L,M} + (1 - SMP) (Q_{L+1,M} + Q_{L,M+1} \\ + Q_{L-1,M} + Q_{L,M-1}) / 4.0 \quad (90)$$

where SMP is a constant and must be between 0.0 and 1.0. Equation (90), obtained from Reference 4., is found to be very helpful for this numerical method. These are the basic differences with the numerical integration presented in Part A of this Chapter.

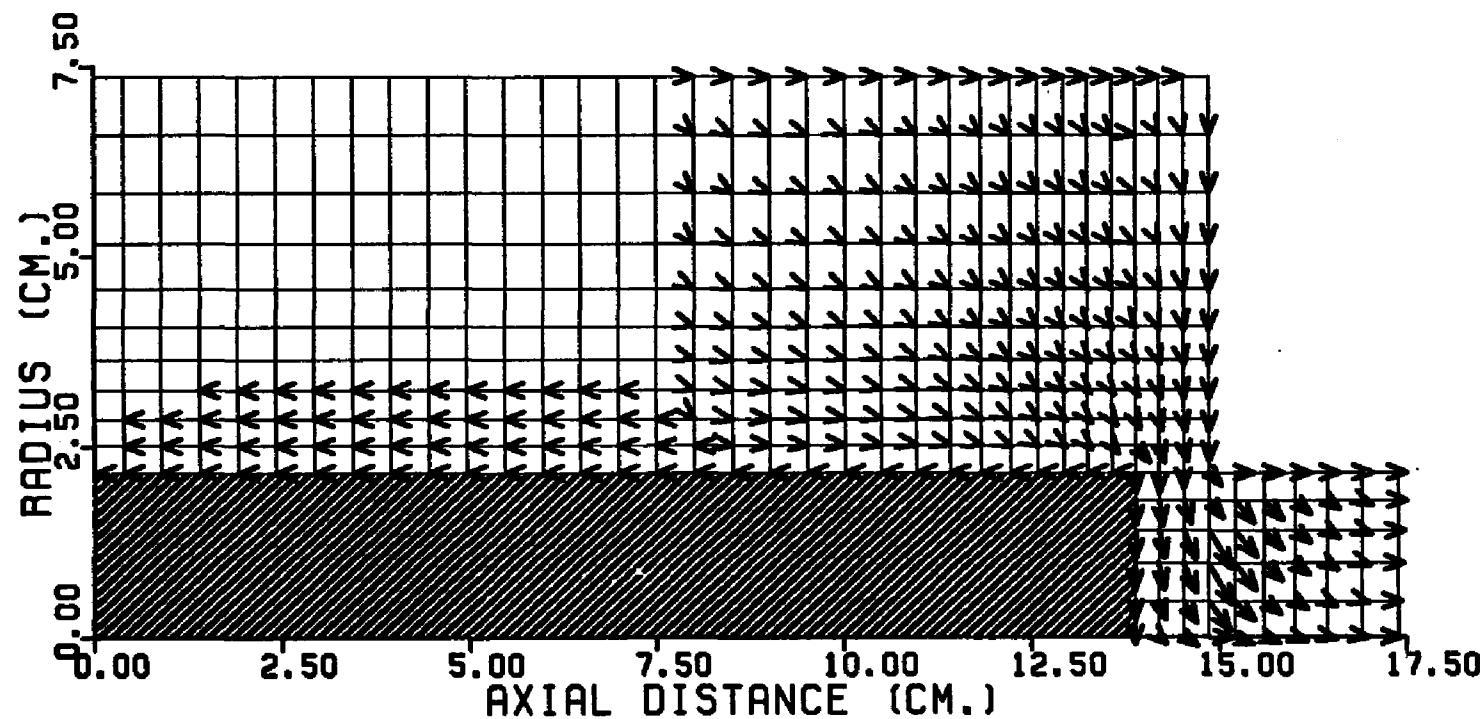
4. Results

Figures 29 through 34 show the velocity vector plots of the flow fields generated during main chamber operation. The flow development is shown for full travel of piston, (displacement = 3.81 CM). Piston displacement and velocity as functions of time, are also shown in Figures 29-34. The maximum velocity obtained in such a short distance is about 41 m/s. The full travel of piston lasted less than 1 ms. Therefore the role of the piston may be thought in some cases as a sudden opening of a valve. In these cases the motion of the piston will have a little effect on the muzzle velocity of the projectile. The assumption of a sudden opening valve may be then acceptable.

The model has generated a variable grid, but not time dependent. In this case, we let the piston grid points move through the grid lines. The flow properties at the piston boundaries are extrapolated from neighboring points.

TIME= 0.02 (MS) PISTON VELOCITY= 0.31 (M/S)

CHARGE PRESSURE=5,000. KPA

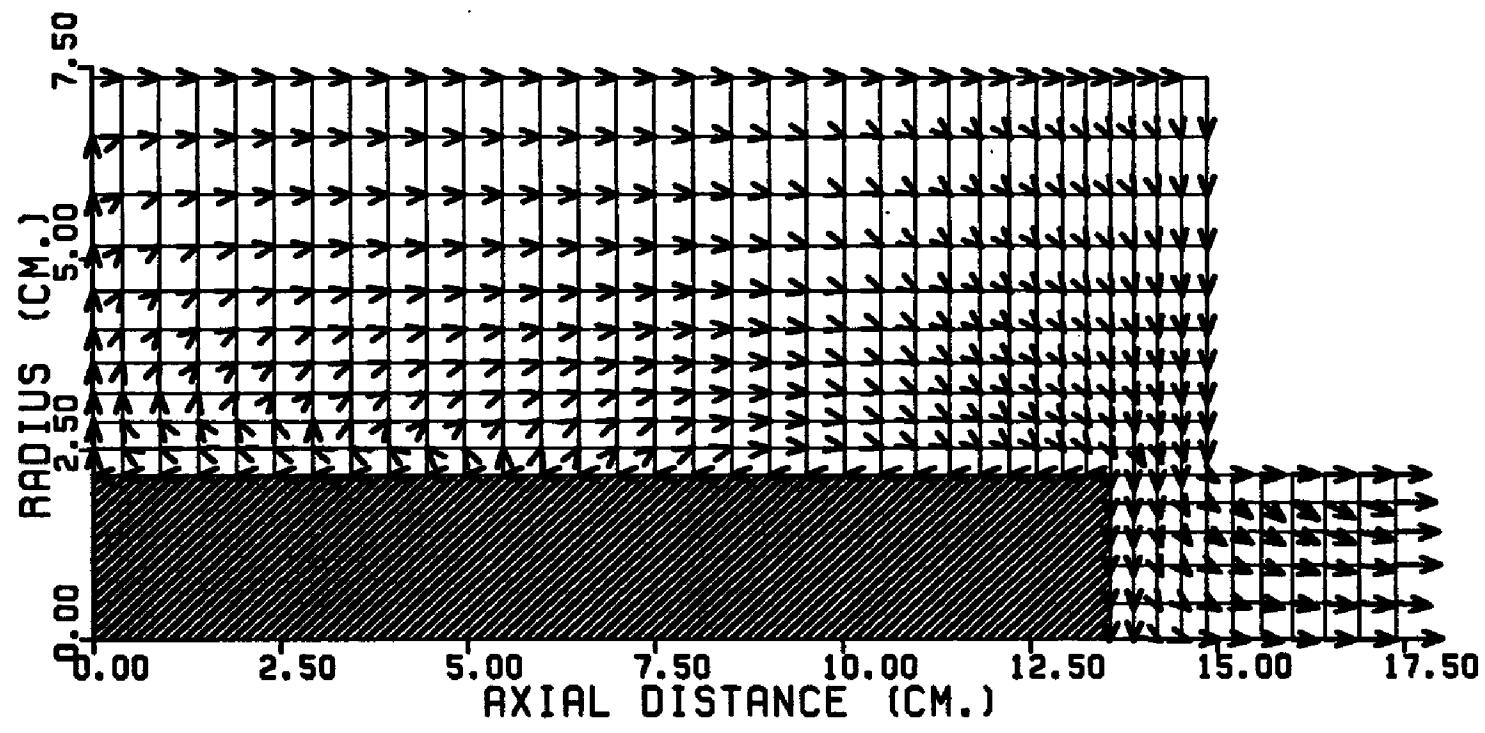


TIME STEP 20

FIGURE 29. VELOCITY DISTRIBUTION IN THE MAIN CHAMBER

TIME= 0.44 (MS) PISTON VELOCITY= 15.90 (M/S)

CHARGE PRESSURE=5,000. KPA

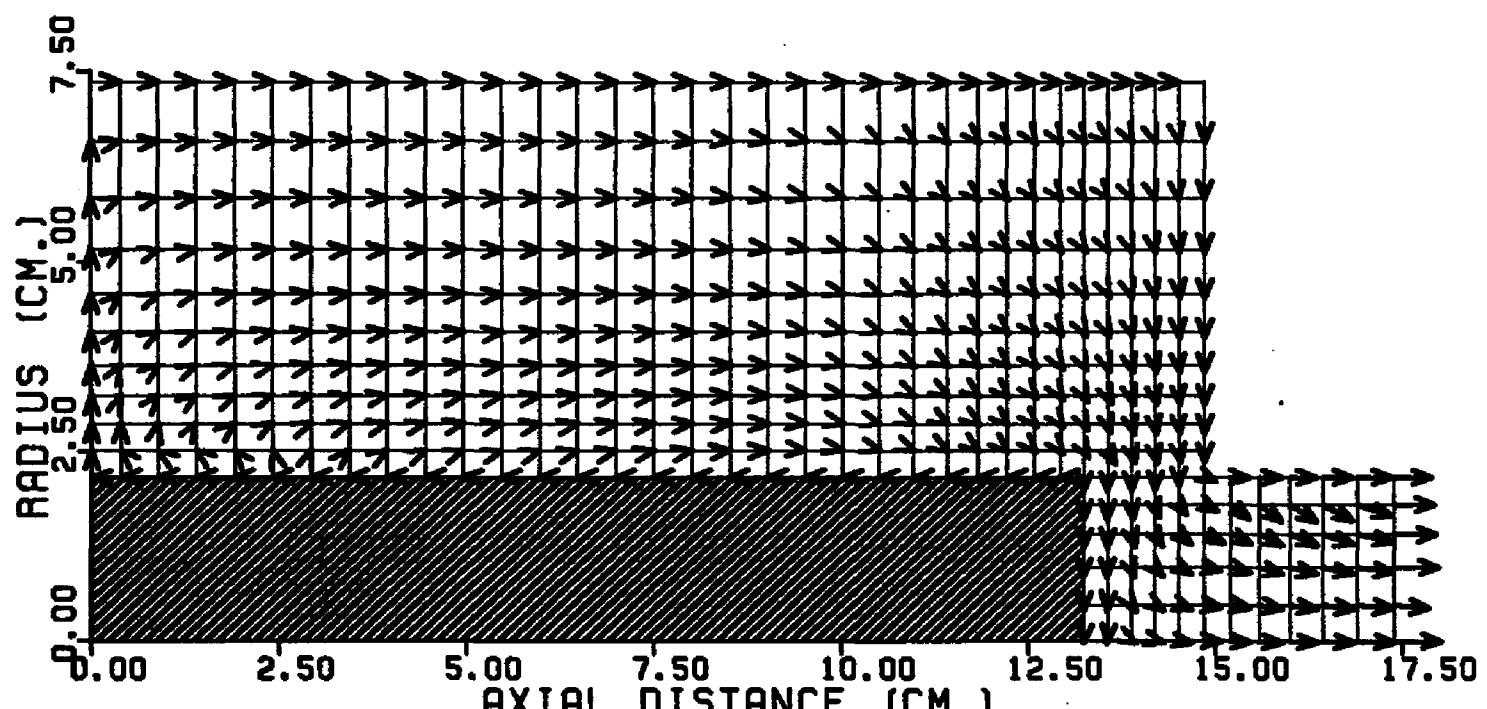


TIME STEP 339

FIGURE 30. VELOCITY DISTRIBUTION IN THE MAIN CHAMBER

TIME= 0.60 (MS) PISTON VELOCITY= 23.27 (M/S)

CHARGE PRESSURE=5,000.KPA

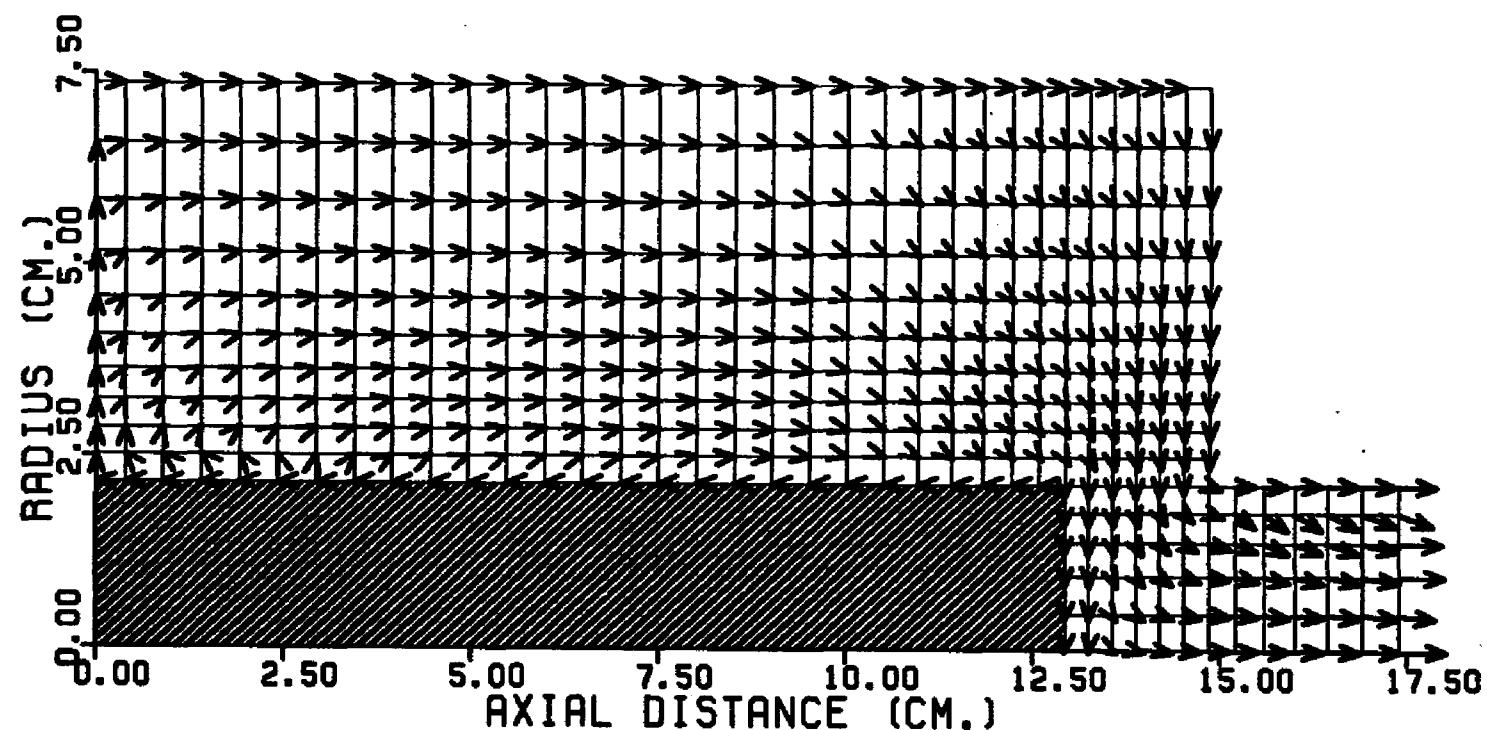


TIME STEP 451

FIGURE 31. VELOCITY DISTRIBUTION IN THE MAIN CHAMBER

TIME= 0.72 (MS) PISTON VELOCITY= 29.42 (M/S)

CHARGE PRESSURE=5,000.KPA

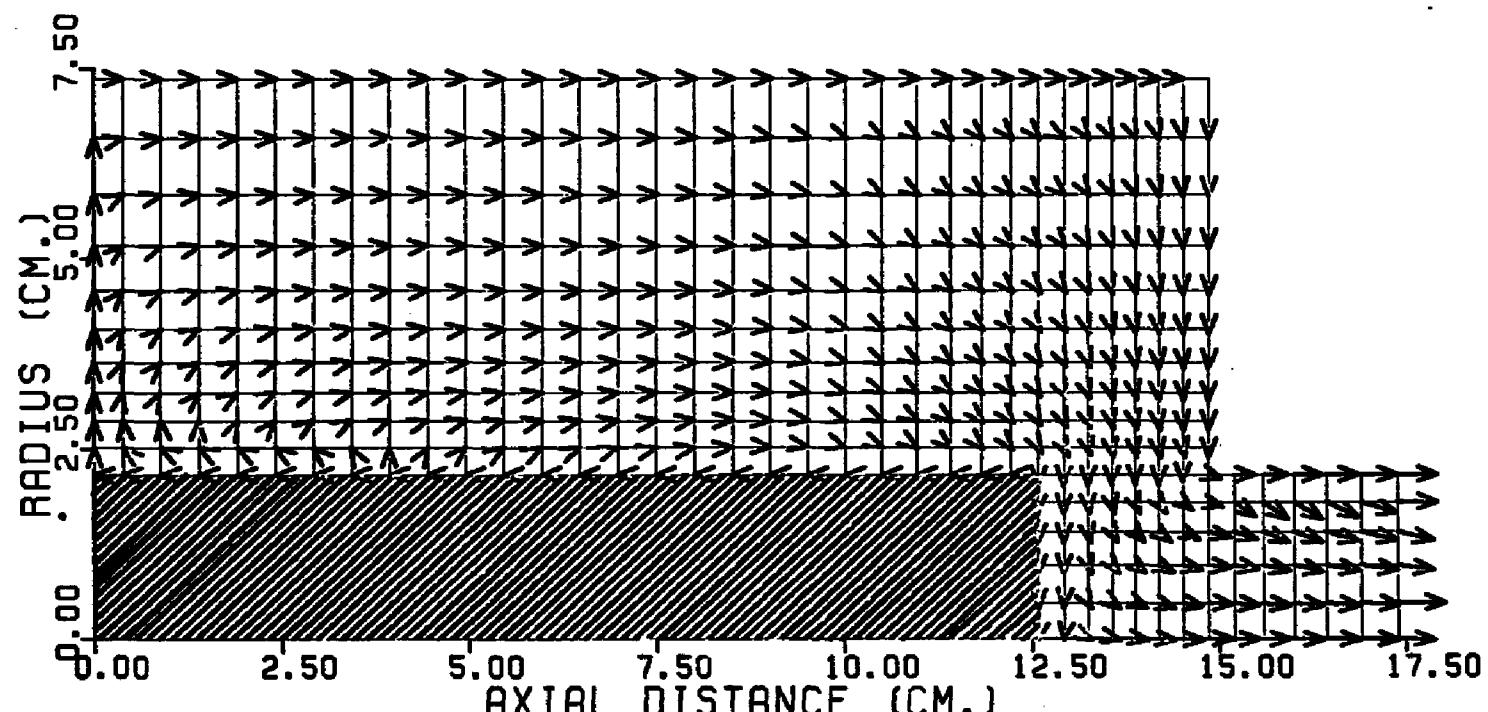


TIME STEP 533

FIGURE 32. VELOCITY DISTRIBUTION IN THE MAIN CHAMBER

TIME= 0.83 (MS) PISTON VELOCITY= 35.04 (M/S)

CHARGE PRESSURE=5,000. KPA

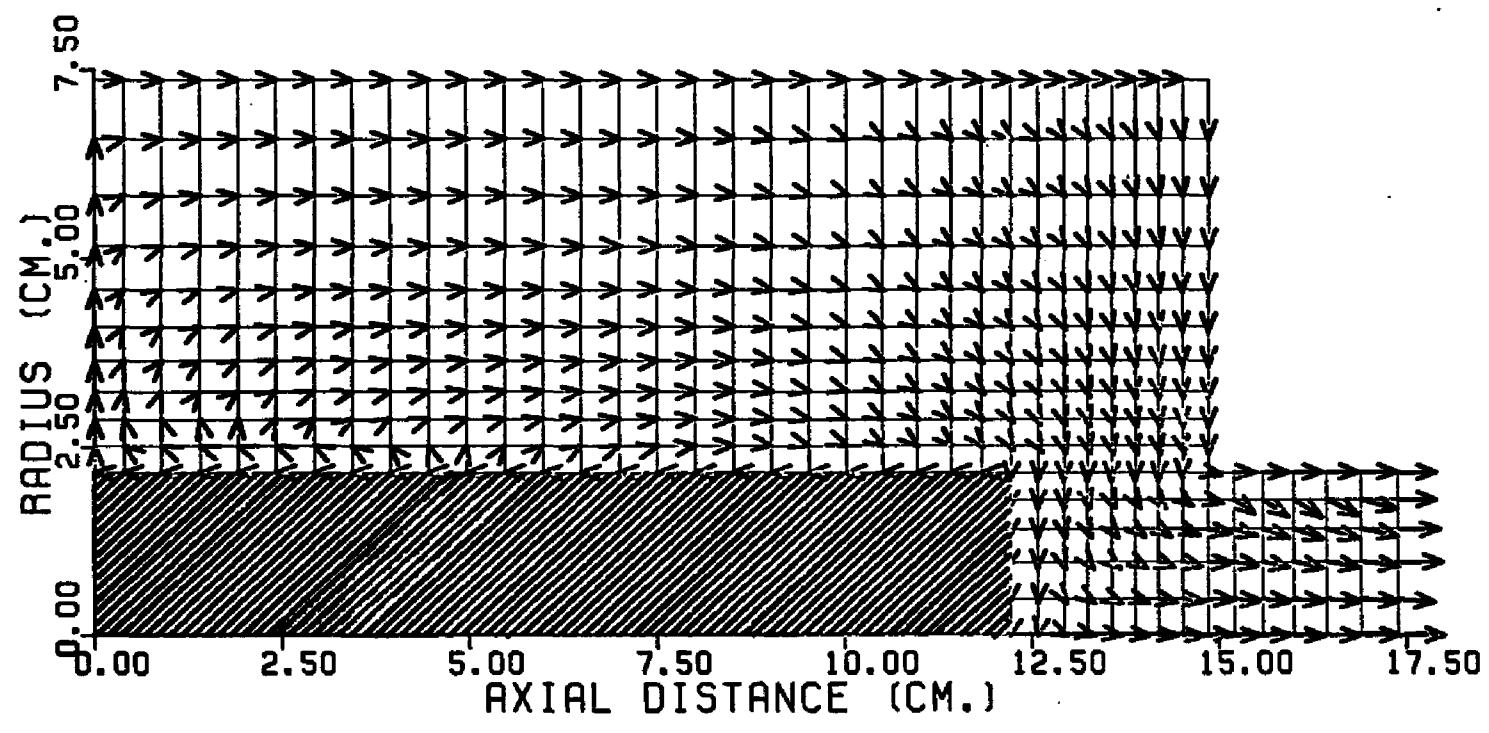


TIME STEP 604

FIGURE 33. VELOCITY DISTRIBUTION IN THE MAIN CHAMBER

TIME= 0.93 (MS) PISTON VELOCITY= 40.45 (M/S)

CHARGE PRESSURE=5,000.KPA



TIME STEP 671

FIGURE 34. VELOCITY DISTRIBUTION IN THE MAIN CHAMBER

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APPENDIX A
APPLICATION OF THE METHOD OF CHARACTERISTICS
FOR UNSTEADY ONE-DIMENSIONAL FLOW

I. GOVERNING EQUATIONS

The governing equations (37) - (40), can be written as

$$\rho_t + u\rho_x + \rho u_x = \Lambda \quad (A-1)$$

$$u_t + u u_x + P_x/\rho = F \quad (A-2)$$

$$P_t + u P_x - a^2 (\rho_t + u \rho_x) = \Psi \quad (A-3)$$

where the t and x subscripts denote the partial derivatives with respect to those variables.

II. CHARACTERISTIC CURVES

A linear combination of the equations of motion can be formed by multiplying Eqs.(A-1) - (A-3) by λ_i , ($i=1,2,3$) respectively, and then summing them. This linear combination can be written as

$$\begin{aligned} & \lambda_1(\rho_t + u \rho_x + \rho u_x - \Lambda) + \lambda_2(u_t + u u_x + P_x/\rho - F) \\ & + \lambda_3(P_t + u P_x - a^2 \rho_t - a^2 u \rho_x - \Psi) = 0 \end{aligned} \quad (A-4)$$

Rearrangement of Eq. (A-4) yields

$$\begin{aligned} & (u \lambda_1 - a^2 u \lambda_3) \rho_x + (\lambda_1 - a^2 \lambda_3) \rho_t + (u \lambda_2 + \lambda_1) u_x + \lambda_2 u_t \\ & + (\lambda_2/\rho + u \lambda_3) P_x + \lambda_3 P_t = (\lambda_1 \Lambda + \lambda_2 F + \lambda_3 \Psi) \end{aligned} \quad (A-5)$$

Equation (A-5) may be interpreted as the sum of directional derivatives of u , P , and ρ in the directions of vectors \overline{w}_1 , \overline{w}_2 , and \overline{w}_3 , respectively, where the components of the vectors w_i ($i=1,2,3$) are the coefficients of the x and t

derivatives in Eq. (A-5). Thus

$$\overline{w_1} = (u \ell_1 - a^2 u \ell_3, \ell_1 - a^2 \ell_3) \quad (A-6)$$

$$\overline{w_2} = (u \ell_2 + p \ell_1, \ell_2) \quad (A-7)$$

$$\overline{w_3} = (\ell_2/p + u \ell_3, \ell_3) \quad (A-8)$$

Therefore, Eq. (A-5) may be written as

$$v_p \cdot w_1 + v_u \cdot w_2 + v_P \cdot w_3 = (\ell_1 \Lambda + \ell_2 F + \ell_3 \Psi) \quad ((A-9))$$

where v is the gradient vector; that is, $v = (\frac{\partial}{\partial x}, \frac{\partial}{\partial t})$

Equation (A-9) may be written as,

$$d_{\overline{w_1}} p + d_{\overline{w_2}} u + d_{\overline{w_3}} P = (\ell_1 \Lambda + \ell_2 F + \ell_3 \Psi) \quad (A-10)$$

where $d_{\overline{w_1}} p = v_p \cdot w_1$ is the derivative of p in the

direction of the vector w_1 .

A question is now posed: can the arbitrary factors ℓ_1 , ℓ_2 , and ℓ_3 be chosen so that the vectors $\overline{w_i}$ ($i = 1, 2, 3$) all lie along a curve in the xt plane. If such ℓ_i ($i = 1, 2, 3$) do exist, the curve that contains the vectors w_i is called the characteristic curve. Let N be the normal to that curve. The vector N , which is called the characteristic normal, is shown in Figure (A-1).

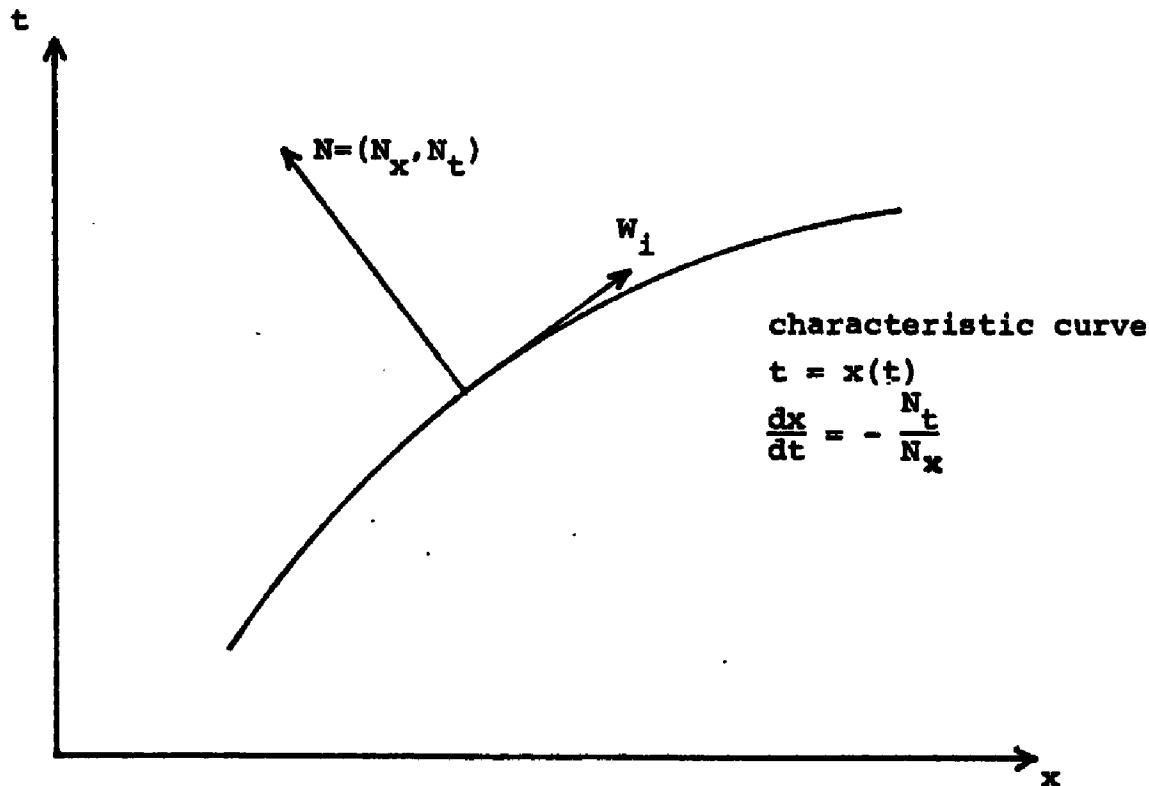


Figure A-1. Relationship between the vectors w_i and N , and the characteristic curve.

Since N is normal to $\overline{w_i}$ ($i = 1, 2, 3$), we obtain

$$N \cdot \overline{w_i} = 0 \quad (i = 1, 2, 3) \quad (\text{A-11})$$

Equation (A-11) may be written as a system of three equations

$$\begin{aligned} (u N_x + N_t) \ell_1 - a^2 (u N_x + N_t) \ell_3 &= 0 \\ \rho N_x \ell_1 + (u N_x + N_t) \ell_2 &= 0 \\ N_x / \ell_1 \ell_2 + (u N_x + N_t) \ell_3 &= 0 \end{aligned} \quad (\text{A-12})$$

In matrix form Eqs. (A-12) become

$$\begin{vmatrix} uN_x + N_t & 0 & -a^2(uN_x + N_t) \\ \rho N_x & uN_x + N_t & 0 \\ 0 & N_x/\rho & (uN_x + N_t) \end{vmatrix} \begin{vmatrix} \lambda_1 \\ \lambda_2 \\ \lambda_3 \end{vmatrix} = 0 \quad (\text{A-13})$$

For the system of equations specified by Eqs. (A-13) to have any solution other than the trivial solution $\lambda_1 = \lambda_2 = \lambda_3 = 0$, the determinant must equal zero. Setting the determinant equal to zero, we obtain

$$(uN_x + N_t)[(uN_x + N_t)^2 - a^2N_x^2] = 0 \quad (\text{A-14})$$

Equation (A-14), which is called the characteristic equation, contains the information required to determine the components of the characteristic normal, N , and hence, the characteristic curves.

Setting the first factor of Eq. (A-14) equal to zero, we obtain

$$uN_x + N_t = 0 \quad (\text{A-15})$$

Setting the second factor of Eq. (A-14) equal to zero, we obtain,

$$uN_x + N_t = \pm aN_x \quad (\text{A-16})$$

Eqs. (A-15) and (A-16) can be written as

$$\frac{dx}{dt} = u \quad (\text{A-17})$$

$$\frac{dx}{dt} = u \pm a \quad (\text{A-18})$$

Equation (A-17) represents the flow pathlines. Equation (A-18) represents the Mach lines.

III. SOLUTION FOR THE λ_i 's

The compatibility equation that is applicable on a characteristic curve is determined from Eq. (A-10).

First, the values of the λ_i 's must be determined along the characteristic curves. Consider first the characteristic curve given by Eq. (A-17). Recall that

$u N_x + N_t = 0$ along the pathline. Thus, Eqs. (A-13) become

$$\begin{vmatrix} 0 & 0 & 0 \\ \rho N_x & 0 & 0 \\ 0 & N_x/\rho & 0 \end{vmatrix} \begin{vmatrix} \lambda_1 \\ \lambda_2 \\ \lambda_3 \end{vmatrix} = 0 \quad (\text{A-19})$$

Since the rank of the matrix is two in Eq. (A-19), one of the λ_i 's is a free variable.

Thus a solution may be written,

$$\lambda_1 = 0, \lambda_2 = 0, \lambda_3 = 1 \quad (\text{A-20})$$

Consider next the characteristic curve given by Eq. (A-18).

Recall that $u N_x + N_t = \pm a N_x$ along the Mach lines. Thus, Eqs. (A-13) become

$$\begin{vmatrix} \pm a N_x & 0 & -a^3 N_x \\ \rho N_x & \pm a N_x & 0 \\ 0 & N_x/\rho & \pm a N_x \end{vmatrix} \begin{vmatrix} \lambda_1 \\ \lambda_2 \\ \lambda_3 \end{vmatrix} = 0 \quad (\text{A-21})$$

The rank of the coefficient matrix of Eq. (A-21) is two. There is one independent solution for λ_i . A solution can now be written

$$\begin{aligned}\lambda_1 &= a^2 \\ \lambda_2 &= \pm \rho a \\ \lambda_3 &= 1\end{aligned}\tag{A-22}$$

IV. COMPATIBILITY EQUATIONS

Substituting Eq. (A-20) into Eq. (A-4) yields

$$(P_t + u P_x) - a^2(\rho_t + u \rho_x) = \Psi \tag{A-23}$$

Substituting Eq. (A-22) into Eq. (A-4) yields

$$\begin{aligned}a^2(\rho_t + u \rho_x + \rho u_x - \Lambda) &\pm \rho a(u_t + u u_x + P_x/\rho - F) \\ + (P_t + u P_x - a^2 \rho_t - a^2 u \rho_x - \Psi) &= 0\end{aligned}\tag{A-24}$$

Equation (A-24) may be rearranged to yield

$$\begin{aligned}[P_t + (u \pm a) P_x] &\pm \rho a[u_t \pm (u \pm a) u_x] \\ &= (\Psi + a^2 \Lambda \pm \rho a F)\end{aligned}\tag{A-25}$$

Equations (A-23) and (A-25) can be written as

Along pathlines

$$\begin{aligned}dx &= u dt \\ dP - a^2 d\rho &= \Psi dt\end{aligned}\tag{A-26}$$

Along Mach lines

$$dx = (u \pm a)dt$$

$$dp \pm \rho a du = (\gamma + a^2 \Lambda \pm \rho a F) \quad (A-27)$$

APPENDIX B
CONSTANT ETA REFERENCE PLANE
CHARACTERISTIC RELATIONS

I. GOVERNING EQUATIONS

The governing equations (12) - (15) can be written as

$$\rho_{\tau} + u^* \rho_{\xi} + \rho w u_{\xi} = - v^* \rho_{\eta} - \rho \alpha u_{\eta} - \rho v/\bar{\eta} - \rho \beta v_{\eta} \quad (B-1)$$

$$u_{\tau} + u^* u_{\xi} + w P_{\xi}/\rho = - v^* u_{\eta} - \alpha P_{\eta} + QuT \quad (B-2)$$

$$v_{\tau} + u^* v_{\xi} = - v^* v_{\eta} - \beta P_{\eta}/\rho + QVT \quad (B-3)$$

$$P_{\tau} + u^* P - a^2 (\rho_{\tau} + u^* \rho_{\xi}) = - v^* P_{\eta} + a^2 v^* \rho_{\eta} + QPT \quad (B-4)$$

where the τ , ξ , and η subscripts denote partial derivatives with respect to those variables. QuT, QVT, and QPT denote, respectively, the viscous terms in the ξ -momentum, η -momentum, and energy equations. Letting ψ_1 , ψ_2 , ψ_3 , and ψ_4 be, respectively, the right hand side of Eqs (B-1) - (B-4). Then Eqs (B-1) - (B-4) become

$$\rho_{\tau} + u^* \rho_{\xi} + \rho w u_{\xi} = \psi_1 \quad (B-5)$$

$$u_{\tau} + u^* u_{\xi} + w P_{\xi}/\rho = \psi_2 \quad (B-6)$$

$$v_{\tau} + u^* v_{\xi} = \psi_3 \quad (B-7)$$

$$P_{\tau} + u^* P_{\xi} - a^2 (\rho_{\tau} + u^* \rho_{\xi}) = \psi_4 \quad (B-8)$$

II. CHARACTERISTIC CURVES

A linear combination of the equations of motion can

be formed by multiplying Eqs. (B-5) - (B-8) by P_i , ($i = 1, 2, 3, 4$) and then summing them. This linear combination can be written as

$$\begin{aligned} & \ell_1(\rho_\tau + u^*\rho_\xi + \rho w u_\xi - \psi_1) + \ell_2(u_\tau + u^*u_\xi + w P_\xi/\rho - \psi_2) \\ & + \ell_3(v_\tau + v^*v_\xi - \psi_3) + \ell_4(P_\tau + u^*P_\xi - a^2\rho_\tau - a^2\rho - a^2u^*\rho_\xi \\ & - \psi_4) = 0 \end{aligned} \quad (B-9)$$

Rearrangement of Eq. (B-9) yields

$$\begin{aligned} & (u^*\ell_1 - a^2u^*\ell_4)\rho_\xi + (\ell_1 - a^2\ell_4)\rho_\tau + (\rho w \ell_1 + u^*\ell_2)u_\xi \\ & + \ell_2 u_\tau + u^*\ell_3 v_\xi + \ell_3 v_\tau + (\ell_2 w/\rho + u^*\ell_4)P_\xi + \ell_4 P_\tau \\ & = \sum_{i=1}^4 \ell_i \psi_i \end{aligned} \quad (B-10)$$

Equation (B-10) may be interpreted as the sum of directional derivatives of ρ , u , v , and P in the directions of vectors \bar{w}_1 , \bar{w}_2 , \bar{w}_3 , and \bar{w}_4 , respectively, where the components of the vectors w_i ($i = 1, 2, 3, 4$) are the coefficients of the ξ and τ derivatives in Eq. (B-10). Thus

$$\bar{w}_1 = (u^*\ell_1 - a^2u^*\ell_4, \ell_1 - a^2\ell_4) \quad (B-11)$$

$$\bar{w}_2 = (\rho w \ell_1 + u^*\ell_2, \ell_2) \quad (B-12)$$

$$\bar{w}_3 = (u^*\ell_3, \ell_3) \quad (B-13)$$

$$\bar{w}_4 = (\ell_2 w/\rho + u^*\ell_4, \ell_4) \quad (B-14)$$

Therefore Eq. (B-10) may be written as

$$\nabla \rho \cdot W_1 + \nabla u \cdot W_2 + \nabla v \cdot W_3 + \nabla p \cdot W_4 = \sum_{i=1}^4 \ell_i \psi_i \quad (B-15)$$

A question is now posed: Can the arbitrary factors ℓ_1 , ℓ_2 , ℓ_3 , and ℓ_4 be chosen so that the vectors \bar{W}_i ($i = 1, 2, 3, 4$) all lie along a curve in the $\xi-\tau$ plane? If such ℓ_i ($i = 1, 2, 3, 4$) do exist, the curve that contains the vectors \bar{W}_i is called the characteristic curve. Let N be the characteristic normal to that curve. Since N is normal to \bar{W}_i ($i = 1, 4$), we obtain.

$$N \cdot \bar{W}_i = 0 \quad (i = 1, 4) \quad (B-16)$$

Equation (B-16) may be written as a system of equations

$$(u^* \ell_1 - a^2 u^* \ell_4) N_\xi + (\ell_1 - a^2 \ell_4) N_\tau = 0 \quad (B-17)$$

$$(\rho w \ell_1 + u^* \ell_2) N_\xi + \ell_2 N_\tau = 0 \quad (B-18)$$

$$u^* \ell_3 N_\xi + \ell_2 N_\tau = 0 \quad (B-19)$$

$$\ell_2 w/\rho + u^* \ell_4 N_\xi + \ell_4 N_\tau = 0 \quad (B-20)$$

In matrix form Eqs (B-17) - (B-20) become

$$\begin{vmatrix} u^* N_\xi + N_\tau & 0 & 0 & -a^2 (u^* N_\xi + N_\tau) \\ \rho w N_\xi & u^* N_\xi + N_\tau & 0 & 0 \\ 0 & 0 & u^* N_\xi + N_\tau & 0 \\ 0 & 0 & 0 & u^* N_\xi + N_\tau \end{vmatrix} \begin{vmatrix} \ell_1 \\ \ell_2 \\ \ell_3 \\ \ell_4 \end{vmatrix} = 0 \quad (B-21)$$

For the system of equations specified by Eqs. (B-21) to have any solution other than the trivial solution

$\lambda_1 = \lambda_2 = \lambda_3 = 0$, the determinant must equal zero. Setting the determinant equal to zero, we obtain

$$(u^*N_\xi + N_\tau)^2 [(u^*N_\xi + N_\tau)^2 - a_w^2 N_\xi^2] = 0 \quad (B-22)$$

Equation (B-22), which is called the characteristic equation, contains the information required to determine the components of the characteristic normal N and hence the characteristic curves.

Setting the first factor of Eq. (B-22) equal to zero, we obtain

$$u^*N_\xi + N_\tau = 0 \quad (B-23)$$

Setting the second factor of Eq. (B-22) equal to zero, we obtain

$$u^*N_\xi + N_\tau = \pm a_w N_\xi \quad (B-24)$$

Equations (B-23) and (B-24) can be written

$$d\xi/d\eta = u^* \quad (B-25)$$

$$d\xi/d\eta = u^* \pm a_w \quad (B-26)$$

Equation (B-24) represents the projection of the flow pathlines on the $\eta = \text{constant}$ planes. Equation (B-26) represents the projection of the Mach cones on the η constant planes.

III. SOLUTION FOR THE λ'_i 's

The compatibility equation that is applicable on a characteristic curve is determined from Eq. (B-15).

First the values of the λ_i 's must be determined along the characteristic curves. Consider first the characteristic curve given by Eq. (B-23). Substituting Eq. (B-23) into Eqs. (B-21) yields

$$\begin{vmatrix} 0 & 0 & 0 & \pm a^3 w N_g \\ \rho w N_g & \pm a w N_g & 0 & 0 \\ 0 & 0 & \pm a w N_g & 0 \\ 0 & w N_g / \rho & 0 & \pm a w N_g \end{vmatrix} \begin{vmatrix} \lambda_1 \\ \lambda_2 \\ \lambda_3 \\ \lambda_4 \end{vmatrix} = 0 \quad (B-28)$$

since the rank of the coefficient matrix of Eq. (B-28) is two. Thus, there are two free variables. Therefore, two solutions are possible

$$(0, 0, 1, 0) \text{ and } (0, 0, 0, 1) \quad (B-27)$$

Consider next the characteristic curve given by Eq. (B-24). Substituting Eq. (B-24) into Eq. (B-21) yields

$$\begin{vmatrix} \pm a w N_g & 0 & 0 & \pm a^3 w N_g \\ \rho w N_g & \pm a w N_g & 0 & 0 \\ 0 & 0 & \pm a w N_g & 0 \\ 0 & w N_g / \rho & 0 & \pm a w N_g \end{vmatrix} \begin{vmatrix} \lambda_1 \\ \lambda_2 \\ \lambda_3 \\ \lambda_4 \end{vmatrix} = 0 \quad (B-28)$$

The rank of the coefficient matrix of Eq. (B-28) is three. There is only one free variable. Therefore, the solution is

$$(a^2, \pm \rho a, 0, 1) \quad (B-29)$$

IV. COMPATIBILITY EQUATIONS

Substituting Eq. (B-27) into Eq. (B-10) yields

$$p_\tau + u^* p_\xi - a^2 (\rho_\tau + u^* \rho_\xi) = \psi_4 \quad (B-30)$$

$$u^* v_\xi + v_\tau = \psi_3 \quad (B-31)$$

For $d\xi = u^* d\tau$.

Substituting Eq. (B-29) into Eq. (B-10) yields

$$dp \pm \rho a du = (\psi_4 + a^2 \psi_1 \pm \rho a \psi_2) d\tau \quad (B-32)$$

For $d\xi = (u^* \pm aw) d\tau$.

APPENDIX C
CONSTANT ZETA REFERENCE PLANE
CHARACTERISTIC RELATIONS

I. GOVERNING EQUATIONS

The governing equations (12) - (15), can be written as

$$\rho_{\tau} + v^* \rho_{\eta} + \rho \alpha u_{\eta} + \rho \beta v_{\eta} = - u^* - \rho w u_{\xi} - \rho v / \bar{\eta} \quad (C-1)$$

$$u_{\tau} + v^* u_{\eta} + \alpha p_{\eta} / \rho = - u^* u_{\xi} - w p_{\xi} / \rho + QUT \quad (C-2)$$

$$v_{\tau} + v^* v_{\eta} + \beta p_{\eta} / \rho = - u^* v_{\xi} + QVT \quad (C-3)$$

$$p_{\tau} + v^* p_{\eta} - a^2 (\rho_{\tau} + v^* \rho_{\eta}) = - u^* p_{\xi} + a^2 u^* \rho_{\xi} + QPT \quad (C-4)$$

where the τ , ξ , and η subscripts denote partial derivatives with respect to those variables. Letting ψ_1 , ψ_2 , ψ_3 , and ψ_4 be respectively the right hand side of Eqs. (C-1) - (C-4). Then Eqs. (C-1) - (C-4) become

$$\rho_{\tau} + v^* \rho_{\eta} - \rho \alpha u_{\eta} + \rho \beta v_{\eta} = \psi_1 \quad (C-5)$$

$$u_{\tau} + v^* u_{\eta} + p_{\eta} / \rho = \psi_2 \quad (C-6)$$

$$v_{\tau} + v^* v_{\eta} + \beta p_{\eta} / \rho = \psi_3 \quad (C-7)$$

$$p_{\tau} + v^* p_{\eta} - a^2 (\rho_{\tau} + v^* \rho_{\eta}) = \psi_4 \quad (C-8)$$

II. CHARACTERISTIC CURVES

Using the same method as that described in Appendix B, we have

$$\nabla p \cdot \bar{w}_1 + \nabla u \cdot \bar{w}_2 + \nabla v \cdot \bar{w}_3 + \nabla p \cdot \bar{w}_4 = \sum_{i=1}^4 \ell_i v_i \quad (C-9)$$

where

$$\begin{aligned}\bar{w}_1 &= (\rho \ell_1 v^* - a^2 v^* \ell_4, \ell_1 - a^2 \ell_4) \\ \bar{w}_2 &= (\rho \alpha \ell_1 + v^* \ell_2, \ell_2) \\ \bar{w}_3 &= (\rho \beta \ell_1 + v^* \ell_3, \ell_3) \\ \bar{w}_4 &= (\alpha \ell_2 / \rho + \beta \ell_2 / \rho + v^* \ell_4, \ell_4)\end{aligned}\quad (C-10)$$

The matrix form of Eq. $N \cdot w_i = 0 \quad i = 1, 4$ can be written as follows.

$$\begin{vmatrix} v^* N_\eta + N_\tau & 0 & 0 & -a^2 (v^* N_\eta + N_\tau) \\ \rho \alpha N_\eta & v^* N_\eta + N_\tau & 0 & 0 \\ \rho \beta N_\eta & 0 & v^* N_\eta & 0 \\ 0 & \alpha N_\eta / \rho & \beta N_\eta / \rho & v^* N_\eta + N_\tau \end{vmatrix} \begin{vmatrix} \ell_1 \\ \ell_2 \\ \ell_3 \\ \ell_4 \end{vmatrix} = 0 \quad (C-11)$$

Setting the determinant equal to zero yields,

$$v^* N_\eta + N_\tau = 0 \quad (C-12)$$

$$v^* N_\eta + N_\tau = \pm a \sqrt{\alpha^2 + \beta^2} N_\eta \quad (C-13)$$

This can also be written as,

$$\frac{dv}{d\tau} = v^* \quad (C-14)$$

$$\frac{d\zeta}{d\tau} = v^* \pm a\alpha^* \quad \text{where } \alpha^* = \sqrt{\alpha^2 + \beta^2} \quad (\text{C-15})$$

Equations (C-14) and (C-15) are the characteristic curves.

III. SOLUTIONS FOR THE λ_i 's.

- 1) Substituting Eq. (C-12) into Eq. (C-11) yields,

$$\lambda_1 = 0, \lambda_2 = \beta/a, \lambda_3 = 1, \lambda_4 = 0. \quad (\text{C-16})$$

$$\text{and } \lambda_1 = \lambda_2 = \lambda_3 = 0, \lambda_4 = 1 \quad (\text{C-17})$$

- 2) Substituting Eq. (C-13) into Eq. (C-11) yields,

$$\lambda_1 = a^2, \lambda_2 = \pm \rho\alpha a/\alpha^*, \lambda_3 = \pm \rho\beta a/\alpha^*, \lambda_4 = 1 \quad (\text{C-18})$$

IV. COMPATIBILITY EQUATIONS

- 1) Substituting Eqs. (C-16) and (C-17) into Eq. (C-9) yields

$$\beta du - \alpha dv = (\beta \psi_2 - \alpha \psi_3) d\tau \quad (\text{C-19})$$

$$dp - a^2 dp = \psi_4 d\tau \quad (\text{C-20})$$

along the characteristic $d\eta = v^* d\tau$

- 2) Substituting Eq. (C-18) into Eq. (C-9) yields

$$\begin{aligned} dp \pm \rho a du/\alpha^* \pm \rho \beta a dv/\alpha^* &= (\psi_4 + a^2 \psi_1 \pm \rho \alpha a \psi_2/\alpha^* \\ &\pm \rho \beta a \psi_3/\alpha^*) d\tau \end{aligned} \quad (\text{C-21})$$

along the characteristics $d\eta = (v^* \pm \alpha^* a) d\tau$.

* APPENDIX D *

FORTRAN IV LISTING OF THE PROGRAM
FOR THE DISCHARGE CHAMBER FLOW

* MAIN PROGRAM *

THIS COMPUTER PROGRAM IS FOR THE
COMPUTATION OF TWO DIMENSIONAL,
TIME DEPENDENT FLOW,
REPRESENTATIVE OF A DISCHARGE
CHAMBER FLOW. THE DISCHARGE CHAMBER
WALL MAY BE ARBITRARY CURVED.

```
DIMENSION KSI(1000),XP(1000),XP1(50,20),YP1(50,20)
DIMENSION TIME(1502),PRES1(1502),PRES2(1502),PRES3(1502)
COMMON/UNESID/UD(4),VD(4),PD(4),ROD(4)
COMMON/SOLUIN/U(50,20,2),V(50,20,2),P(50,20,2),
$ R0(50,20,2)
COMMON/CNIKLC/LMAX,MMAX,NMAX,TCONV,FDT,GAMMA,RGAS,GAM1,
$ GAM2,GAM3,L1,L2,M1,M2,DY,DT,ICHAR,N1D,IEER,IUI,IUO,
$ DAR,DYR,RSTAR,N,N1,N3,RSTARS,G,PC,TC,LC,PLOW,ROLW,RG,
$ DAX,KEI
COMMON/GENTRL/XW(50),YW(50),NXY(50),LT,XT,XI,XE,NDIM,
$ XWI(50),YWI(50)
COMMON/BCC/PT(20),TT(20),PE(20),IEXTRA,TW(20),ISUPER,
$ UI(20),VI(20),PI(20),ROI(20),IEX
COMMON/AV/CAV,NST,LSS,XMU,XLA,RKMU,XRO,QUT(50,20),
$ QVT(50,20),QPT(50,20),QRQT(50,20)
COMMON/RV/CMU,CLA,CK,EMU,ELA,EK,CHECK,ITM,TML
REAL MN3,MANY,LC,KSI
```

C
C
C

SET DEFAULT VALUES

```
RSTARS=0.0
SMACH=0.0
PE(1)=0.
TT(1)=0.0
PE(1)=14./
LC=12.0
XMU=0.0
XLA=1.0
RKMU=0.7
NCOUNT=1
XRO=0.6
TW(1)=-1.0
PE(2)=-1
```

C

```
TCONV=0.003
FDT=0.5
CMU=0.0
```

```
T=0.0
NPOLT=800
JCOUNT=1
N1=1
N3=2
E=1.
NDIM=1
GAMMA=1.4
RGAS=53.35
G=32.174
PC=144.0
GAM1=GAMMA/(GAMMA-1.0)
GAM2=(GAMMA-1.0)/2.0
GAM3=(GAMMA+1.0)/(GAMMA-1.0)
RG=RGAS*G
PLOW=0.01*PC
RLOW=0.0001/G

C
C      ARTIFICIAL VISCOSITY
C
CAV=0.0
XMU=.4
XLIA=1.0
RKMU=.7
XHO=.6
NST=0
SMP=0.95
LSS=2
SMACH=0.0
IAV=1

C
C      PARAMETERS DEFINING THE MOLECULAR VISCOSITY
C
CMU=0.0000009643
CLA=-0.0000006429
CK=0.00127
EMU=0.5
ELA=0.5
EK=0.5
LSS=2
CAV=0.4
SMACH=0.0
ITM=0.0
CMU=CMU/47.88/1.88**EMU
CLA=CK*0.125/1.8**EK
CMU=CMU*LC
CLA=CLA*LC
CK=CK*LC
NOSLIP=1
NOSLIP=0
```

```

CHECK=ABS(CMU)+ABS(CLX)+ABS(CK)
CHECK=0.0
BI=1000000.
BI=2500000.0
PT(1)=BI
PE(1)=14.7*PC
TT(1)=540.
IUI=1
IJO=1
IEXTA=0
IEK=1
ISUPER=0
NID=1
MMAX=8
LMAX=48
C
C      INITIAL CONDITIONS
C
DO 120 M=2,MMAX
PT(M)=PT(1)
TT(M)=TT(1)
PE(M)=PE(1)
120 CONTINUE
C
C      SET INDICES AND ZERO VISCOSITY TERM ARRAYS
C
DO 320 M=1,MMAX
DO 320 L=1,LMAX
QUT(L,M)=0.0
QVI(L,M)=0.0
QPT(L,M)=0.0
QRROT(L,M)=0.0
320 CONTINUE
LJET=23
JFLAG=1
NID=0
ISUPER=-1
C
C      GRID GENERATION IN THE DISCHARGE CHAMBER
C
R1=2.5
RE=0.5
XO=1.75
RF=0.25
XF=4.0
XI=0.0
XE=5.0
GA=6.0
GF=10.
GF=20.

```

```

LMAX=48
MMAX=8
KMAX=1000
L1=LMAX-1
M1=MMAX-1
K1=KMAX-1
DS=XE/FLOAT(K1)
XP(1)=0.0
KSI(1)=0.0
SLOPE1=0.0
S=0.0
X=0.0
DO 2 L=2,KMAX
X=X+DS
IF ( X.GT.3.0) GO TO 144
SLOPE2=-(K1-RE)*GA*0.5/(COSH(GA*(X-X0))*COSH(GA*(X-X0)))
GO TO 145
144 SLOPE2=-(KE-RF)*GF*0.5/(COSH(GF*(X-XF))*COSH(GF*(X-XF)))
145 S1=S+T(1.+SLOPE1*SLOPE1)
S2=SQRT(1.+SLOPE2*SLOPE2)
S=S+DS*0.5*(S1+S2)
KSI(L)=S
XP(L)=X
SLOPE1=SLOPE2
2 CONTINUE
XE1=KSI(KMAX)
DKSI=XE1/FLOAT(L1)
J=1
SS=-DKSI
DO 3 L=1,LMAX
SS=SS+DKSI
DO 4 K=J,KMAX
IF ( ABS(SS-KSI(K)).LT.0.05) GO TO 5
4 CONTINUE
5 J=K
XW(L)=XP(K)
X=XW(L)
IF ( X.GT.3.0) GO TO 146
NXNY(L)=+(RI-RE)*GA*0.5/(COSH(GA*(X-X0))*COSH(GA*(X-X0)))
YW(L)=(RI+KE)*0.5-(RI-RE)*TANH(GA*(X-X0))*0.5
GO TO 147
146 NXNY(L)=+(RE-RF)*GF*0.5/(COSH(GF*(X-XF))*COSH(GF*(X-XF)))
YW(L)=(KE+RF)*0.5-(RE-RF)*TANH(GF*(X-XF))*0.5
147 WRITE(6,100) KS1(K),XW(L),YW(L),NXNY(L)
3 CONTINUE
C      PLOT THE PHYSICAL GRID
C
CALL IDENT
CALL PLOT (2.0,6.0,-3)

```

```

CALL VECTOR(-1.93,7.88,6.67,7.88)
CALL VECTOR(6.67,7.88,6.67,-3.12)
CALL VECTOR(6.67,-3.12,-1.93,-3.12)
CALL VECTOR(-1.93,-3.12,-1.93,7.88)
DO 6 L=1,LMAX
DYIO=YW(L)/FLOAT(M1)
Y=-DYIO
DO 6 M=1,MMAX
Y=Y+DYIO
XP1(L,M)=YW(L)
YP1(L,M)=Y
6 CONTINUE
DO 55 L=1,LMAX
CALL VTHICK(3)
CALL VTHICK(3)
CALL VECTOR(XP1(L,1),YP1(L,1),XP1(L,MMAX),YP1(L,MMAX))
55 CONTINUE
DO 44 M=1,L1
DO 44 M=1,MMAX
CALL VTHICK(3)
CALL VTHICK(3)
CALL VECTOR(XP1(L,M),YP1(L,M),XP1(L+1,M),YP1(L+1,M))
44 CONTINUE
ME=MMAX
L1=LMAX-1
L2=LMAX-2
L3=LMAX-3
M1=MMAX-1
M2=MMAX-2
DY=1./FLOAT(MMAX-1)
DX=XE1/FLOAT(LMAX-1)
DXR=1./DX
DXA=DXR
DYR=1./DY
DXAS=DXR*DXA
DYRS=DYR*DYR
PA=103000.
ROA=1.22
COI=ROA*(B1/PA)**(0.714)
B1=B1*PC/6894.8
COA=1.22/(G*16.02)
COI=COI/(G*16.02)
DO 14 L=1,LMAX
DO 14 M=1,MMAX
U(L,M,N1)=0.0001
V(L,M,N1)=0.0001
P(L,M,1)=B1
RO(L,M,1)=COI
14 CONTINUE
ISUPER=-1

```

```

UD(1)=U(LJET-1,MMAX,N1)
VD(1)=V(LJET-1,MMAX,N1)
PD(1)=P(LJET-1,MMAX,N1)
ROD(1)=RO(LJET-1,MMAX,N1)
UD(2)=UD(1)
VD(2)=VD(1)
PD(2)=PD(1)
ROD(2)=ROD(1)
DO 131 L=1,LMAX
XWI(L)=XW(LJET-1)
YWI(L)=YW(L)
131 CONTINUE
MMAX=15000
J=1
TIME(1)=0.0
PRES1(1)=P(1,1,1)/PC
PRES2(1)=P(1,MMAX,1)/PC
PRES3(1)=P(5,MMAX,1)/PC
B=14.7*1.05
C
C      ENTER THE TIME STEP INTEGRATION
C
DO 660 M=1,MMAX
DO 450 L=1,LMAX
W=SQRT(1.+NXNY(L)*NXNY(L))
CALL MAP(0,L,MMAX,AL,BE,DE,LD1,AL1,BE1,DE1)
DXDY=DADS*N*W+DYRS*BE*BE
DO 450 M=1,MMAX
CALL EOS(1,P(L,M,N1),RO(L,M,N1),TEMP,AS,D2,D3)
UPA=ABS(U(L,M,N1))*W*DXDY+ABS(V(L,M,N1))*BE*DYR+SQRT(AS*DXDY)
IF (L.EQ.1.AND.M.EQ.1) UPAM=UPA
IF (UPA.GT.UPAM) UPAM=UPA
450 CONTINUE
DT=YDT/UPAM
DTT=DT/LC
T=T+DTT
ICHAR=1
CALL INTEK
CALL YWALL
DO 132 M=1,MMAX
U(1,M,N3)=0.0
V(1,M,N3)=0.0
RO(1,M,N3)=RO(2,M,N3)
132 CONTINUE
CALL INLET
CALL EXIT
ICHAR=2
CALL INTEK
CALL YWALL
DO 133 M=1,MMAX

```

```

U(1,M,N3)=0.0
V(1,M,N3)=0.0
RO(1,M,N3)=RO(2,M,N3)
133 CONTINUE
CALL INLET
CALL EXIT
CALL SMOOTH
IF ( JCOUNT.EQ.400 ) GO TO 85
JCOUNT=JCOUNT+1
GO TO 84
85 JCOUNT=1
C
C      PRINT THE SOLUTIONS
C
      WRITE(6,88) DTT,T
      DO 580 L=1,LMAX
      X=XW(L)
      DY10=DY*YW(L)
      CALL MAP(0,L,1,AL,BE,DE,LD1,AL1,BE1,DE1)
      DY10=DY/(BE)
      Y=-DY10
      DO 580 M=1,MMAX
      Y=Y+DY10
      VELMAG=SQRT(U(L,M,N3)**2+V(L,M,N3)**2)
      CALL EOS(5,P(L,M,N3),RO(L,M,N3),TEMP,AS,D2,D3)
      XMACH=VELMAG/SQRT(AS)
      PRES=P(L,M,N3)/PC
      RHO=RO(L,M,N3)*G
      XX=X
      YP=Y
      XP1(L,M)=XX
      YP1(L,M)=YP
      UP=U(L,M,N3)
      VP=V(L,M,N3)
      WRITE(6,920) L,M,XX,YP,UP,VP,PRES,RHO,VELMAG,XMACH
580 CONTINUE
      CALL GRAPH(XP1,YP1,N3)
84 NNN=N 1
     N1=N3
     N3=NNN
660 CONTINUE
C
C      FORMAT STATEMENTS
C
88 FORMAT (/,5X,'DT=',F12.9,3X,'T=',F12.9,/)

100 FORMAT(5X,4F14.5)
910 FORMAT(1H,25X,4H(CH),7X,4H(CM),6X,5H(M/S),7X,5H(M/S),7X,
     3,6H(KPA),7X,7H(KG/M3),5X,5H(M/S),10X,2HNO)
920 FORMAT(1H,7X,2I5,4F12.4,F13.5,F12.6,2F12.4)
1190 FORMAT(1H,10X,'-----')

```

5, '-----'
5, '-----')
CALL EOJOB
STOP
END

```

SUBROUTINE MAP(IP,L,M,AL,BE,DE,LD1,AL1,BE1,DE1)
C
C ***** *****
C THIS SUBROUTINE CALCULATES THE MAPPING FUNCTIONS
C
C ***** *****
C
DIMENSION KSI(1000),XP(1000),XP1(50,20),XP1(50,20)
DIMENSION TIME(1502),PRES1(1502),PRES2(1502),PRES3(1502)
COMMON/ONESID/UD(4),VD(4),PD(4),RD(4)
COMMON/SOLUTH/U(50,20,2),V(50,20,2),P(50,20,2),
$ RD(50,20,2)
COMMON/CNTALC/LKAX,MMAX,NMAX,ICONV,FDT,GAMMA,RGAS,GAM1,
$ GAM2,GAM3,L1,L2,M1,M2,DY,D1,ICHAR,N1D,IEER,IUI,IUC,
$ DAK,DYK,&STAR,N,N1,N3,&STARs,G,PC,TC,LC,PLOW,ROLOW,RG,
$ DXX,XE1
COMMON/GEOTRL/XW(50),YW(50),NXNY(50),LT,XT,XI,XE,NDIM,
$ XWI(50),YWI(50)
COMMON/BCC/PT(20),TT(20),FE(20),IEXTRA,TW(20),ISUPER,
$ UI(20),VI(20),PI(20),BOI(20),IEK
COMMON/AV/CAV,NST,LSS,XMU,XLA,RKMU,XRO,QUT(50,20),
$ QVT(50,20),QPT(50,20),QROT(50,20)
COMMON/RV/CMU,CLA,CK,EMU,ELA,EK,CHECK,ITM,TML
REAL MN3,MANY,LC
BE=1.0/YW(L)
IF(IP.EQ.0) RETURN
Y=FLOAT(M-1)*DY
AL=BE*Y*MANY(L)
DE=0.
IF(IP.EQ.1) RETURN
BE1=1.0/YW(LD1)
AL1=BE1*Y*MANY(LD1)
DE1=0.
RETURN
END

```

```

C SUBROUTINE EOS(I1,PRESS,RHO,TEMP,D1,D2,M3)
C ****
C THIS SUBROUTINE CALCULATES THE EQUATION OF STATE
C QUANTITIES
C ****
C DIMENSION KSI(1000),XP(1000),XP1(50,20),YF1(50,20)
C DIMENSION TIME(1502),PRES1(1502),PRES2(1502),PRES3(1502)
C COMMON/ONESID/UD(4),VD(4),PD(4),R0D(4)
C COMMON/SOLUTN/U(50,20,2),V(50,20,2),P(50,20,2),
$ R0(50,20,2)
C COMMON/CNTRLC/LMAX,MMAX,NMAX,TCONV,FDT,GAMMA,RGAS,GAM1,
$ GAM2,GAM3,L1,L2,M1,M2,DX,DY,DT,ICHAR,N1D,IERR,IUI,IUO,
$ DXR,DYR,BSTAR,N,N1,N3,RSTARS,G,PC,TC,LC,PLOW,ROLOW,RG,
$ DXX,XE1
C COMMON/GEMTRL/XW(50),YW(50),NXNY(50),LT,XT,XI,XE,NDIM,
$ XW1(50),YW1(50)
C COMMON/BCC/PT(20),TT(20),PE(20),IEXTBA,TW(20),ISUPER,
$ U1(20),VI(20),PI(20),BCI(20),IEK
C COMMON/AV/CAV,NST,LSS,XMU,RKMU,XRO,QUT(50,20),
$ QVT(50,20),QPT(50,20),QRROT(50,20)
C COMMON/RV/CMU,CLA,CK,EMU,ELA,EK,CHECK,ITM,TML
REAL RHO,MANY,LC
GO TO (10,20,30,40,50,60,70,80,90,100,110,120,130,140,
$ 150),I1
C
C CALCULATE THE SOUND SPEED SQUARED(D1=AS)
C
10 D1=GAMMA*PRESS/RHO
RETURN
C
C CALCULATE THE TEMPERATURE
C
20 TEMP=PRESS/(RHO*RG)
RETURN
C
C CALCULATE THE PRESSURE
C
30 PRESS=TEMP*RHO*RG
RETURN
C
C CALCULATE THE DENSITY
C
40 RHO=PRESS/(TEMP*RG)
RETURN
C
C CALCULATE THE TEMPERATURE AND SOUND SPEED SQUARED

```

```

C      ND1=AS)
C
50    TEMP=PRESS/(RHO*RG)
D1=GAMMA*PRESS/RHO
RETURN

C      CALCULATE THE DENSITY AND SOUND SPEED SQUARED ND1=AS)
C
60    RHO=PRESS/(TEMP*RG)
D1=GAMMA*PRESS/RHO
RETURN

C      CALCULATE THE SOUND SPEED SQUARED FROM THE TEMPERATURE
C      ND1=AS)
C
70    D1=GAMMA*RG*TEMP
RETURN

C      CALCULATE THE PRESSURE AND TEMPERATURE FROM THE
C      STAGNATION
C      CONDITIONS ND1=PT,D2=TT,MN3=MACH NO)
C
80    DEM=1.0+GAM2*MN3*MN3
PRESS=D1/DEM**GAM1
TEMP=D2/DEM
RETURN

C      CALCULATE THE STAGNATION TEMPERATUTE FROM THE
C      STATIC CONDITIONS ND1=PT,D2=TT,MN3=MACH NO)
C
90    DEM=1.0+GAM2*MN3*MN3
D1=PRESS*DEM**GAM1
D2=TEMP*DEM
RETURN

C      CALCULATE THE MACH NUMBER ND2=TT,MN3=MACH NO)
C
100   MN3=SQRT((D2/TEMP-1.0)/GAM2)
RETURN

C      CALCULATE THE DENSITY FOR THE UNDER-EXPANDED JET(D1=PD
C      ,D2=OD)
C
110   RHO=D2*(PRESS/D1)**(1.0/GAMMA)
RETURN

C      CALCULATE THE DENSITY FOR THE OVER-EXPANDED JET (ND1=PO
C      ,D2=OD)
C
120   PRO=PRESS/D1

```

```
RHO=D2*(GAM3*PRO+1.0)/(PRC*GAM3)
RETURN
C
C      CALCULATE THE SQUARE ROOT OF GAM3 FOR THE SHARP
C      EXPANSION CORNER
C
130  D1=SQRT(GAM3)
RETURN
C
C      CALCULATE GAMMM=1.0 FOR THE INTERNAL ENERGY EQUATION R.M.S
C      R,M,S
C
140  D1=GAMMA-1.0
RETURN
C
C      CALCULATE THE TERM FOR THE ARTIFICIAL CONDUCTIVITY
C
150 D1=GAM1*KG/RKMU
RETURN
END
```

```

C SUBROUTINE INTER
C ****
C THIS SUBROUTINE CALCULATES THE INTERIOR MESH POINTS
C ****
C
C DIMENSION KSI(1000),XP(1000),XP1(50,20),YP1(50,20)
C DIMENSION TIME(1502),PRES1(1502),PRES2(1502),PRES3(1502)
C COMMON/ONESID/UD(4),VD(4),PD(4),ROD(4)
C COMMON/SOLUTN/U(50,20,2),V(50,20,2),P(50,20,2),
C $ RO(50,20,2)
C COMMON/CNTRLC/LMAX,MMAX,NMAX,TCONV,FDT,GAMMA,RGAS,GAM1,
C $ GAM2,GAM3,L1,L2,M1,M2,DX,DY,DT,ICHAR,N1D,IERR,IUI,IUO,
C $ DAR,DIA,MSTAR,N,N1,N3,RSTARS,G,PC,TC,LC,PLOW,ROLOW,RG,
C $ DXX,XE1
C COMMON/GENTRL/XW(50),YW(50),NXNY(50),LT,XT,XI,XE,NDIM,
C $ XWI(50),YWI(50)
C COMMON/BCC/PT(20),TT(20),PE(20),IEXTRA,TW(20),ISUPER,
C $ U1(20),V1(20),P1(20),ROI(20),IEX
C COMMON/AV/CAV,NST,LSS,XMU,XLA,EKMU,XRO,QUT(50,20),
C $ QVT(50,20),QPT(50,20),QRROT(50,20)
C COMMON/EV/CMU,CLA,CK,EMU,ELA,EK,CHECK,ITM,TML
C REAL MN3,NANY,LC
C DP=0.0
C ATERM=0.
C IF(ICHAR.NE.1) GO TO 40
C
C COMPUTE THE TENTATIVE SOLUTION AT T+DT
C
C DO 30 L=2,L1
C W=SQRT(1.+NXNY(L)*NANY(L))
C DO 30 M=1,M1
C CALL MAP(1,L,M,AL,BE,DE,LD1,AL1,BE1,DE1)
C UB=U(L,M,N1)
C VB=V(L,M,N1)
C PB=P(L,M,N1)
C ROB=RO(L,M,N1)
C
C CALL EOS(1,PB,ROB,T,ASB,D2,D3)
C
C U1B=W*UB+DP
C IF(W.NE.1) GO TO 10
C
C COMPUTATION OF THE MIDPLANE MESH POINTS
C
C COMPUTATION OF THE DERIVATIVES WITH RESPECT TO X

```

```

C
DUDX=(UE-U(L-1,M,N1))*DXR
DPDX=(PB-P(L-1,M,N1))*DXR
DRODX=(ROB-RO(L-1,M,N1))*DXR

C COMPUTATION OF THE DERIVATIVES WITH RESPECT TO X
C
DVDY=(4.0*V(L,2,N1)-V(L,3,N1))*5*DVR
V(L,M,N3)=0.0
UKHS=-U1B*DUDX-W*DPDX/ROB+QUT(L,M)
RKHS=-U1B*DRODX-ROB*W*DUDX-FLOAT(1+NDIM)*ROB*DUDY*BE+
$ QROT(L,M)
P&nS=-U1B*DPDX+ASB*(ROHS+U1B*DRODX)+QPT(L,M)
GO TO 20
10 IF (NDIM.EQ.1) ATERM=ROB*VB/(FLOAT(M-1)*DY/BE)

C COMPUTATION OF THE DERIVATIVES WITH RESPECT TO X
C
UVB=UB*AL+VB*BE+DE
DUDX=(UB-U(L-1,M,N1))*DXR
DVDX=(VB-V(L-1,M,N1))*DXR
DPDX=(PB-P(L-1,M,N1))*DXR
DRODX=(ROB-RO(L-1,M,N1))*DXR
DUDY=(UB-U(L,M-1,N1))*DVR
DVDY=(VB-V(L,M-1,N1))*DVR
DPDY=(PB-P(L,M-1,N1))*DVR
DRODY=(ROB-RO(L,M-1,N1))*DVR
UKHS=-U1B*DUDX-UVB*DUDY-(W*DPDX+AL*DPDY)/ROB+QUT(L,M)
VRHS=-U1B*DUDX-UVB*DUDY-BE*DPDY/ROB+QVT(L,M)
RKHS=-U1B*DRODX-UVB*DRODY-ROB*(W*DUDX+AL*DUDY+BE*DUDY)
$ -ATERM+QROT(L,M)
P&nS=-U1B*DPDX-UVB*DPDY+ASB*(RKHS+U1B*DRODX+UVB*DRODY)
$ + QPT(L,M)
V(L,M,N3)=V(L,M,N1)+VRHS*DT
20 U(L,M,N3)=U(L,M,N1)+URHS*DT
P(L,M,N3)=P(L,M,N1)+PRHS*DT
RO(L,M,N3)=RO(L,M,N1)+RKHS*DT
IF(P(L,M,N3).LE.0.0) P(L,M,N3)=PLOW
IF(RO(L,M,N3).LE.0.0) RO(L,M,N3)=ROLOW
30 CONTINUE
RETURN

C COMPUTE THE FINAL SOLUTION AT T+DT
C
40 DO 70 L=2,L1
W=SQR(T(1.+NXNY(L)*NXNY(L)))
DO 70 M=1,M1
CALL MAP(1,L,M,AL,BE,DE,LD1,AL1,BE1,DE1)
UB=U(L,M,N3)
VB=V(L,M,N3)

```

```

PB=P(L,M,N3)
ROB=RO(L,M,N3)

C CALL EOS(I,PB,ROB,T,ASB,D2,D3)

C U1B=W*UB+DP
IF(M.NE.1) GO TO 50
DUDX=(U(L+1,M,N3)-UB)*DXR
DVDX=(V(L+1,M,N3)-VB)*DXR
DPDX=(P(L+1,M,N3)-PB)*DXR
DRODX=(RO(L+1,M,N3)-ROB)*DXR
DVDY=(4.0*V(L,2,N3)-V(L,3,N3))*5*DVR
V(L,M,N3)=0.0
URHS=-U1B*DUDX-W*DPDX/ROB+QUT(L,M)
RORHS=-U1B*DRODX-ROB*W*DUDX-FLOAT(1+NDIM)*ROB*DVDY*BE+
      QUT(L,M)
PRHS=-U1B*DPDX+ASB*(RORHS+U1B*DRODX)+QPT(L,M)
GO TO 60

C 50 IF (NDIM.EQ.1) ATERM=ROB*VB/(FLOAT(M-1)*DY/BE)
UVB=UB*AL+VB*BE+DE
DUDX=(U(L+1,M,N3)-UB)*DXR
DPDX=(P(L+1,M,N3)-PB)*DXR
DVDX=(V(L+1,M,N3)-VB)*DXR
DRODX=(RO(L+1,M,N3)-ROB)*DXR

C DUDY=(U(L,M+1,N3)-UB)*DVR
DVDY=(V(L,M+1,N3)-VB)*DVR
DPDY=(P(L,M+1,N3)-PB)*DVR
DRODY=(RO(L,M+1,N3)-ROB)*DVR

C URHS=-U1B*DUDX-UVB*DUDY-(W*DPDX+AL*DPDY)/ROB+QUT(L,M)
VRHS=-U1B*DUDX-UVB*DUDY-BE*DPDY/ROB+QUT(L,M)
RORHS=-U1B*DRODX-UVB*DRODY-ROB*(W*DUDX+AL*DUDY+BE*DUDY)
      -ATERM+QUT(L,M)
PRHS=-U1B*DPDX-UVB*DPDY+ASB*(RORHS+U1B*DRODX+UVB*DRODY)
      +QPT(L,M)
V(L,M,N3)=(V(L,M,N1)+V(L,M,N3)+VRHS*DT)*.5
60 U(L,M,N3)=(U(L,M,N1)+U(L,M,N3)+URHS*DT)*.5
P(L,M,N3)=(P(L,M,N1)+P(L,M,N3)+PRHS*DT)*.5
RO(L,M,N3)=(RO(L,M,N1)+RO(L,M,N3)+RORHS*DT)*.5
IF(P(L,M,N3).LE.0.0) P(L,M,N3)=PLOW
IF(RO(L,M,N3).LE.0.0) RO(L,M,N3)=ROLLOW
70 CONTINUE
RETURN
END

```

```

SUBROUTINE INLET
C ****
C
C THIS SUBROUTINE CALCULATES THE INLET MESH POINTS
C ****
C
DIMENSION KSI(1000),XP(1000),XP1(50,20),YP1(50,20)
DIMENSION TIME(1502),PRES1(1502),PRES2(1502),PRES3(1502)
COMMON/ONESID/UD(4),VD(4),PD(4),ROD(4)
COMMON/SOLUTN/U(50,20,2),V(50,20,2),P(50,20,2),
$ RO(50,20,2)
COMMON/CNTRLC/LMAX,MMAX,NMAX,TCONV,FDT,GAMMA,RGAS,GAMI,
$ GAM2,GAM3,L1,L2,M1,M2,DY,DT,ICHAE,N1D,IERR,IUI,IUO,
$ DAR,DTR,RSTAR,N,N1,N3,RSTARS,G,PC,TC,LC,PLOW,BOLOW,RG,
$ DXI,XE1
COMMON/GEMTRL/XW(50),YW(50),NXNY(50),LT,XT,XI,XE,NDIM,
$ XI(50),YI(50)
COMMON/BCC/PT(20),TT(20),PE(20),IEXTBA,TW(20),ISUPER,
$ UI(20),VI(20),PI(20),FOI(20),IEK
COMMON/AV/CAV,NST,LSS,XMU,XLA,RKMU,XKO,QUT(50,20),
$ QVT(50,20),QPT(50,20),QROT(50,20)
COMMON/LV/CMU,CLA,CK,EMU,ELA,EK,CHECK,ITM,TML
REAL MN3,NXNY,LC
C
L=1
XI=0.0
XJ=XI
ATERM2=0.
ATERM3=0.
ATERM2=0.0
ATERM3=0.0
DO 240 M=1,MMAX
CALL MAP (Z,1,M,AL,BE,DE,2,AL1,BE1,DE1)
U2=U(L,M,N1)
CALL EOS (1,P(L,M,N1),RO(L,M,N1),T,AS,D2,D3)
A2=SQRT(AS)
IF (ICHAE.NE.1) GO TO 40
IF (ISUPER.EQ.-1) GO TO 30
U(L,M,N3)=U2
V(L,M,N3)=V(L,M,N1)
30 A3=A2
C
C CALCULATE THE PROPERTY INTERPOLATING POLYNOMIAL
C COEFFICIENTS
C
40 BU=(U(L+1,M,N1)-U(L,M,N1))*DXR
BV=(V(L+1,M,N1)-V(L,M,N1))*DXR
BP=(P(L+1,M,N1)-P(L,M,N1))*DXR

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BKO=(AO(L+1,M,N1)-RO(L,M,N1))*DXR
BAL=(AL1-AL)*DXR
BBE=(BE1-BE)*DXR
CU=U(L,M,N1)-BU*X3
CV=V(L,M,N1)-BV*X3
CP=P(L,M,N1)-BP*X3
CR0=AO(L,M,N1)-BKO*X3
CAL=AL-BAL*X3
CBE=BE-BBE*X3

C
C   CALCULATE THE CROSS DERIVATIVE INTERPOLATING POLYNOMIAL
C   COEFFICIENTS
C

IF(M.EQ.1) GO TO 50
DU=(U(L+1,M,N1)-U(L+1,M-1,N1))*DYR
DV=(V(L+1,M,N1)-V(L+1,M-1,N1))*DYR
DP=(P(L+1,M,N1)-P(L+1,M-1,N1))*DYR
DEO=(RO(L+1,M,N1)-RO(L+1,M-1,N1))*D1R
DU1=(U(L,M,N1)-U(L,M-1,N1))*DYR
DV1=(V(L,M,N1)-V(L,M-1,N1))*DYR
DP1=(P(L,M,N1)-P(L,M-1,N1))*DYR
DRO1=(RO(L,M,N1)-RO(L,M-1,N1))*DYR
GO TO 70
50 DU=0.
DV=(4.0*V(2,2,N1)-V(2,3,N1))*0.5*DYF
DP=0.
DRO=0.0
DU1=0.0
DV1=(4.0*V(L,2,N1)-V(L,3,N1))*0.5*DYR
DP1=0.0
DRO1=0.0
70 BDU=(DU-DU1)*DXR
BDV=(DV-DV1)*DXR
BDP=(DP-DP1)*DXR
BDKO=(DKO-DRO1)*DXR
CDU=DU1-BDU*X3
CDV=DV1-BDV*X3
CDP=DP1-BDP*X3
CDRO=DRO1-BDRO*X3

C
C   CALCULATE X2
C

IF(ICHAH.EQ.1) GO TO 80
CALL EOS(1,P(L,M,N3),RO(L,M,N3),T,AS,D2,D3)
A3=SQRT(AS)
80 DO 90 LL=1,2
X2=A3-(U(L,M,N3)-A3+U2-A2)*0.5*DT
C
C   INTERPOLATE FOR THE PROPERTIES
C

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U2=B0*X2+CU
P2=bP*X2+CP
R02=BR0*X2+CRO
CALL EOS(1,P2,R02,T,AS,D2,D3)
A2=SQRT(AS)
90 CONTINUE
V2=BV*X2+CV
AL2=BA1*X2+CAL
BE2=BBE*X2+CBE
UV2=U2*AL2+V2*BE2

C
C      INTERPOLATE FOR THE CROSS DERIVATIVES
C
DU2=BDU*X2+CDU
DV2=BDV*X2+CDV
DP2=BDP*X2+CDP
DR02=BDR0*X2+CDR0

C
C      CALCULATE THE PSI TERMS
C
IF(M.EQ.1) GO TO 100
ATERM2=R02*V2/(DY*FLOAT(M-1)/BE2)
GO TO 110
100 ATERM2=R02*BE2*D2
110 PSI12=-UV2*DR02-R02*AL2*DU2-R02*BE2*D2-ATERM2
PSI22=-UV2*DU2-AL2*D2/R02
PSI42=-UV2*DP2+A2*A2*UV2*DR02
IF(ICHAIR.EQ.1) GO TO 170

C
C      CALCULATE THE CROSS DERIVATIVES AT THE SOLUTION POINT
C
IF(M.EQ.1) GO TO 120
IF(M.EQ.MMAX) GO TO 130
DU3=(U(L,M+1,N3)-U(L,M,N3))*DYR
DV3=(V(L,M+1,N3)-V(L,M,N3))*DYR
DP3=(P(L,M+1,N3)-P(L,M,N3))*DYR
DR03=(RO(L,M+1,N3)-RO(L,M,N3))*DYR
GO TO 140
120 DU3=0.0
DV3=(4.0*V(L,2,N3)-V(L,3,N3))*0.5*DYR
DP3=0.0
DR03=0.0
GO TO 140
130 DU3=(U(L,MMAX,N3)-U(L,M1,N3))*DYR
DV3=(V(L,MMAX,N3)-V(L,M1,N3))*DYR
DP3=(P(L,MMAX,N3)-P(L,M1,N3))*DYR
DR03=(RO(L,MMAX,N3)-RO(L,M1,N3))*DYR

C
C      CALCULATE THE PSI TERM AT THE SOLUTION POINT
C

```

```

140 IF (M.EQ.1) GO TO 150
    ATERM3=RO(L,M,N3)*V(L,M,N3)/(DY*FLOAT(M-1)/BE)
    GO TO 160
150 ATERM3=RO(L,M,N3)*BE*DVS
160 UV3=U(L,M,N3)*AL+V(L,M,N3)*BE
    PSI13=-UV3*DRC3-RO(L,M,N3)*AL*DUB-RO(L,M,N3)*BE*DVS-
$ ATERM3
    PSI23=-UV3*DUB-AL*DP3/RO(L,M,N3)
    PSI43=-UV3*DP3+A3*A3*UV3*DRO3
    GO TO 160
170 PSI23=PSI22
    PS143=PSI42
    PS113=PSI12
180 PS11B=0.5*(PSI12+PSI13)
    PS12B=0.5*(PSI22+PSI23)
    PS14B=0.5*(PSI42+PSI43)

C
C      SOLVE THE COMPATIBILITY EQUATION FOR P
C
IF (ISUPER.EQ.0) GO TO 190
    ROAB=0.5*(RO2*A2+RO(L,M,N3)*A3)
    AB=0.5*(A2+A3)
    P(L,M,N3)=P2+ROAB*(U(L,M,N3)-U2)+(PSI4B-ROAB*PSI2B+AB*-
$ AB*PS11B)*DT
    U(1,M,N3)=0.0
    V(1,M,N3)=0.0
    RO(1,M,N3)=RO(2,M,N3)
    GO TO 240

C
C      SOLVE THE COMPATIBILITY EQUATIONS FOR U,V,P, AND RO
C
190 MN3=SQRT(U(L,M,N3)*U(L,M,N3)+V(L,M,N3)*V(L,M,N3))/A3
    CALL EOS(2,P2,RO2,T2,AS,D2,D3)
    DO 220 ITEA=1,20
    CALL EOS(6,P(L,M,N3),RO(L,M,N3),T3,PT(M),TF(M),MN3)
    PB=(P2+P(L,M,N3))*0.5
    TB=(12+13)*0.5
    CALL EOS(6,PB,ROB,TB,AS,D2,D3)
    U(L,M,N3)=U2+DT*PS12B+(P(L,M,N3)-P2-(PSI4B+AS*PSI1B)*DT)*
$ /(ROB*SQRT(AS))
    OMN3=MN3
    CALL EOS(7,PB,ROB,TB,AS,D2,D3)
    MN3=SQRT((U(L,M,N3)**2+V(L,M,N3)**2)/AS)
    IF (OMN3.NE.0.0) GO TO 210
    IF (ABS(MN3-OMN3).LE.0.0001) GO TO 230
    GO TO 220
210 IF (ABS((MN3-OMN3)/OMN3).LE.0.001) GO TO 230
220 CONTINUE
C
230 CALL EOS(4,P(L,M,N3),RO(L,M,N3),T3,AS,D2,D3)

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C
240 CONTINUE
RETURN
END

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SUBROUTINE EXIT

***** THIS SUBROUTINE CALCULATES THE BOUNDARY MESH POINTS AT
***** THE EXIT PLANE

***** DIMENSION KSI(1000),XP(1000),XP1(50,20),YP1(50,20)
DIMENSION TIME(1502),PRES1(1502),PRES2(1502),PRES3(1502)
COMMON/ONESID/UD(4),VD(4),PD(4),ROD(4)
COMMON/SOLUTN/U(50,20,2),V(50,20,2),P(50,20,2),
$ RO(50,20,2)
COMMON/CNTL/LMAX,MMAX,NMAX,TCONV,FDT,GAMMA,RGAS,GAM1,
$ GAM2,GAM3,L1,L2,M1,M2,DY,DT,ICHAR,NID,IERR,IUI,IUO,
$ DAR,DYR,ASTAR,N,N1,N3,ESTARS,G,PC,TC,LC,PLOW,ROLOW,RG,
$ DXX,XE1
COMMON/GEMTRL/XW(50),YR(50),NXNY(50),LT,XT,XI,XE,NDIM,
$ XW_(50),YWI(50)
COMMON/BCC/PT(20),TT(20),PE(20),IEXTRA,TW(20),ISUPER,
$ UI(20),VI(20),P1(20),ROI(20),IEX
COMMON/AV/CAV,NST,LSS,XMU,XLA,RKMU,XRO,QUIT(50,20),
$ QVT(50,20),QPT(50,20),LRCT(50,20)
COMMON/RV/CMU,CLA,CK,EMU,ELA,EK,CHECK,ITM,TAL
REAL MM3,NXNY,LC
ME=MMAX
X3=XE1
ATERM2=0.0
ATERM3=0.0
DO 180 M=1,ME
IF(IEXTRA.EQ.1) GO TO 10
CALL EOS(1,P(LMAX,M,N1),RO(LMAX,M,N1),T,AS,D2,D3)
A1=SQRT(AS)
IF(IEXTRA.EQ.2) GO TO 20
W=SQRT(U(LMAX,M,N1)*U(LMAX,M,N1)+V(LMAX,M,N1)*
$ V(LMAX,M,N1))
IF(W/A1.LT.1.0) GO TO 20
10 U(LMAX,M,N3)=U(L1,M,N3)+FLOAT(IEX)*(U(L1,M,N3)-
$ U(L2,M,N3))
V(LMAX,M,N3)=V(L1,M,N3)+FLOAT(IEX)*(V(L1,M,N3)-
$ V(L2,M,N3))
P(LMAX,M,N3)=P(L1,M,N3)+FLOAT(IEX)*(P(L1,M,N3)-
$ P(L2,M,N3))
RO(LMAX,M,N3)=RO(L1,M,N3)+FLOAT(IEX)*(RO(L1,M,N3)-
$ RO(L2,M,N3))
GO TO 180
20 CALL MAP(2,LMAX,M,AL,BE,DE,L1,AL1,BE1,DE1)
U1=U(LMAX,M,N1)

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U2=U1
A2=A1
IF(ICHAR.NE.1) GO TO 30
U(LMAX,M,N3)=U1
R0(LMAX,M,N3)=R0(LMAX,M,N1)
A3=A1
C
C      CALCULATE THE PROPERTY INTERPOLATION POLYNOMIAL
C      COEFFICIENTS
C
30 BU=(U(LMAX,M,N1)-U(L1,M,N1))*DXR
BV=(V(LMAX,M,N1)-V(L1,M,N1))*DXR
BP=(P(LMAX,M,N1)-P(L1,M,N1))*DXR
BRO=(R0(LMAX,M,N1)-R0(L1,M,N1))*DXR
BAL=(AL-AL1)*DXR
BBE=(BE-BE1)*DXR
BDE=(DE-DE1)*DXR
CU=U(LMAX,M,N1)-BU*X3
CV=V(LMAX,M,N1)-BV*X3
CP=P(LMAX,M,N1)-BP*X3
CRO=R0(LMAX,M,N1)-BRO*X3
CAL=AL-BAL*X3
CBE=BE-BBE*X3
CDE=DE-BDE*X3
C
C      CALCULATE THE CROSS DERIVATIVE INTERPOLATING POLYNOMIAL
C      COEFFICIENTS
C
IF(M.EQ.1) GO TO 40
DU=(U(LMAX,M,N1)-U(LMAX,M-1,N1))*DYR
DV=(V(LMAX,M,N1)-V(LMAX,M-1,N1))*DYR
DP=(P(LMAX,M,N1)-P(LMAX,M-1,N1))*DYR
DRO=(R0(LMAX,M,N1)-R0(LMAX,M-1,N1))*DYR
DU1=(U(L1,M,N1)-U(L1,M-1,N1))*DYR
DV1=(V(L1,M,N1)-V(L1,M-1,N1))*DYR
DP1=(P(L1,M,N1)-P(L1,M-1,N1))*DYR
DRO1=(R0(L1,M,N1)-R0(L1,M-1,N1))*DYR
GO TO 60
40 DU=0.0
DV=(4.0*V(LMAX,2,N1)-V(LMAX,3,N1))*0.5*DYR
DP=0.0
DRO=0.0
DU1=0.0
DV1=(4.0*V(L1,2,N1)-V(L1,3,N1))*0.5*DYR
DP1=0.0
DRO1=0.0
60 BDU=(DU-DU1)*DXR
BDV=(DV-DV1)*DXR
BDP=(DP-DP1)*DXR
BDR0=(DRO-DRO1)*DXR

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CDU=DU-BDU*X3
CDV=DV-BDV*X3
CDP=DP-BDP*X3
CDR0=DAO-BDRO*X3

C
C      CALCULATE X1 AND X2
C
IF(ICHAR.EQ.1) GO TO 70
CALL EOS(1,P(LMAX,M,N3),R0(LMAX,M,N3),T,AS,D2,D3)
A3=SQRT(AS)
70 DO 80 LL=1,2
X1=X3-(U(LMAX,M,N3)+U1)*0.5*DT
X2=X3-(U(LMAX,M,N3)+A3+U2+A2)*0.5*DT
C
C      INTERPOLATE FOR THE PROPERTIES
C
U1=B0*X1+C0
U2=B1*X2+C1
P2=BP*X2+CP
R02=B0*X2+CRO
CALL EOS(1,P2,R02,T2,AS,D2,D3)
A2=SQRT(AS)
80 CONTINUE
V1=BV*X1+CV
P1=BP*X1+CP
R01=B0*X1+CRO
AL1=BAL*X1+CAL
BE1=BBE*X1+CBE
DE1=BDE*X1+CDE
UV1=U1*AL1+V1*BE1+DE1
CALL EOS(1,P1,R01,T1,AS,D2,D3)
A1=SQRT(AS)
V2=BV*X2+CV
AL2=BAL*X2+CAL
BE2=BBE*X2+CBE
DE2=BDE*X2+CDE
UV2=U2*AL2+V2*BE2+DE2
C
C      INTERPOLATE FOR THE CROSS DERIVATIVES
DV1=BDV*X1+CDV
DP1=BDP*X1+CDP
DR01=BDRO*X1+CDR0
DU2=BDU*X2+CDU
DV2=BDV*X2+CDV
DP2=BDP*X2+CDP
DR02=BDRO*X2+CDEO

C
C      CALCULATE THE PSI TERMS
C
IF(NDIM.EQ.0) GO TO 100

```

```

IF(M.EQ.1) GO TO 90
ATERM2=RO2*V2/(DY*FLOAT(M-1)/BE2)
GO TO 100
90 ATERM2=RO2*BE2*D2
100 PSI31=-UV1*D1-(DP1/A01)*BE1
PSI41=-UV1*DP1+A1*A1*UV1*DRO1
PSI122=-UV2*DRO2-RO2*D2*BE2-ATERM2-RO2*AL2*D02
PSI122=-UV2*D2-AL2*DP2/RO2
PSI42=-UV2*DP2+A2*A2*UV2*DRO2
IF(1CHAR.EQ.1) GO TO 160

C
C      CALCULATE THE CROSS DERIVATIVES AT THE SOLUTION POINT
C

IF(M.EQ.1) GO TO 110
IF(M.EQ.ME) GO TO 120
DU3=(U(LMAX,M+1,N3)-U(LMAX,M,N3))*DYL
DV3=(V(LMAX,M+1,N3)-V(LMAX,M,N3))*DYL
DP3=(P(LMAX,M+1,N3)-P(LMAX,M,N3))*DYL
DRO3=(RO(LMAX,M+1,N3)-RO(LMAX,M,N3))*DYL
GO TO 130
110 DU3=0.0
DV3=(4.0*V(LMAX,2,N3)-V(LMAX,3,N3))*0.5*DYL
DP3=0.0
DRO3=0.0
GO TO 130
120 MB=ME-1
DU3=(U(LMAX,ME,N3)-U(LMAX,MB,N3))*DYL
DV3=(V(LMAX,ME,N3)-V(LMAX,MB,N3))*DYL
DP3=(P(LMAX,ME,N3)-P(LMAX,MB,N3))*DYL
DRO3=(RO(LMAX,ME,N3)-RO(LMAX,MB,N3))*DYL

C
C      CALCULATE THE PSI TERMS AT THE SOLUTION POINT
C

130 IF(BDAM.EQ.0) GO TO 150
IF(M.EQ.1) GO TO 140
ATERM3=RO(LMAX,M,N3)*V(LMAX,M,N3)/(DY*FLOAT(M-1)/BE)
GO TO 150
140 ATERM3=RO(LMAX,1,N3)*BE*D2
150 UV3=V(LMAX,M,N3)*BE+U(LMAX,M,N3)*AL+DE
PSI13=-UV3*DRO3-RO(LMAX,M,N3)*(DV3*BE+AL*D2)-ATERM3
PSI123=-UV3*D2-AL*DP3/RO(LMAX,M,N3)
PSI133=-UV3*D2-(DP3/RO(LMAX,M,N3))*BE
PSI43=-UV3*DP3+A3*A3*UV3*DRO3
PSI131B=(PSI31+PSI13)*0.5+QVT(LMAX,M)
PSI141B=(PSI141+PSI43)*0.5+QPT(LMAX,M)
PSI122B=(PSI122+PSI123)*0.5+QROT(LMAX,M)
PSI142B=(PSI142+PSI43)*0.5+QPT(LMAX,M)
GO TO 170
160 PSI131B=PSI31+QVT(LMAX,M)

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PSI41B=PSI41+QPT(LMAX,M)
PSI12B=PSI12+QROT(LMAX,M)
PSI22B=PSI22+QUT(LMAX,M)
PSI42B=PSI42+QPT(LMAX,M)

C
C      SOLVE THE COMPATIBILITY EQUATIONS FOR U,V,P, AND RO
C

170 P(LMAX,M,N3)=PE(M)
AB=0.5*(A2+A3)
ROB=0.5*(RO2+RO(LMAX,M,N3))
RO(LMAX,M,N3)=RO1+2.0*(P(LMAX,M,N3)-P1-DT*PSI41B)/
$ (A3*A3+A1*A1)
IF(RO(LMAX,M,N3).LE.0.0) RO(LMAX,M,N3)=ROLOW
U(LMAX,M,N3)=U2+((PSI42B+ROB*AB*PSI122B+AB*AB*PSI12B)*DT
$ -(P(LMAX,M,N3)-P2))/(ROB*AB)
V(LMAX,M,N3)=V1+DT*PSI31B

C
C      CHECK FOR INFLOW AND IF SO SET INFLOW BOUNDARY
C      CONDITIONS
C

IF(U(LMAX,M,N3).GE.0.0) GO TO 180
V(LMAX,M,N3)=0.0
HO(LMAX,M,N3)=0.5*(RO(LMAX,1,N1)+RO(LMAX,MMAX,N1))

180 CONTINUE
V(LMAX,MMAX,N3)=-U(LMAX,MMAX,N3)*NXNY(LMAX)
RETURN
END

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SUBROUTINE SHOOTS

THIS SUBROUTINE IS USED TO SMOOTH THE PLOW VARIABLES

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DIMENSION KSI(1000),XP(1000),XB1(50,20),YB1(50,20)
DIMENSION TIME(1502),PRES1(1502),PRES2(1502),PRES3(1502)
COMMON/ONESID/UD(4),VD(4),PU(4),RDP(4)
COMMON/SOLUTN/U(50,20,2),V(50,20,2),P(50,20,2),
      & AO(50,20,2)
COMMON/CNTRIC/LMAX,NMAX,MAXX,TCONV,FDT,GAMMA,RGAS,GAM1,
      & GAM2,GAM3,L1,L2,M1,M2,DX,DX,DT,ICHAR,NID,IEER,IUI,IUO,
      & DUR,DUR,ASTAR,N1,N3,RSTARS,G,PC,TC,LC,PLOW,ROLOW,RG,
      & DIA,XE1
COMMON/SENTEL/XW(50),YW(50),NXN(50),L1,XT,AI,KE,NDIM,
      & AHI(50),YHI(50)
COMMON/BCC/PT(20),TT(20),PE(20),IEXTBA,TW(20),ISUPER,
      & UL(20),VL(20),PL(20),HOI(20),IEX
COMMON/AV/CAV,NST,ISS,XMU,XLA,XRU,XKO,JUT(50,20),
      & JVT(50,20),OPT(50,20),CROT(50,20)
COMMON/RV/CRU,CLACK,EMU,ELA,EK,CHECK,ITM,THL
REAL MN3,WNY,LC,KSI

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SMP=0.45
SMP4=0.25*(1.0-SMP)
KMAX=KMAX
J=1
KI=KMAX-1
MS=J+1
DO 20 I=2,L1
U(L,KMAX,N3)=SMP4*(U(L-1,KMAX,N3)+U(L+1,KMAX,N3)+2.0*
& U(L,K1,N3))+SMP*U(L,KMAX,N3)
V(L,KMAX,N3)=-U(L,KMAX,N3)*NKNY(L)
P(I,KMAX,N3)=SMP4*(P(L-1,KMAX,N3)+P(L+1,KMAX,N3)+2.0*
& P(L,K1,N3))+SMP*2*(L,KMAX,N3)
RO(L,KMAX,N3)=SMP4*(RO(L-1,KMAX,N3)+RO(L+1,KMAX,N3)+2.0*
& *RO(L,K1,N3))+SMP*RO(I,KMAX,N3)
U(L,J,N3)=SMP4*(U(L-1,J,N3)+U(L+1,J,N3)+2.0*U(L,HS,N3))
& +SMP*U(L,J,N3)
V(I,J,N3)=SMP4*(V(L-1,J,N3)+V(L+1,J,N3)+2.0*V(L,MS,N3))
& +SMP*V(L,J,N3)
P(L,J,N3)=SMP4*(P(L-1,J,N3)+P(L+1,J,N3)+2.0*P(L,HS,N3))
& +SMP*P(L,J,N3)
RO(L,J,N3)=SMP4*(RO(L-1,J,N3)+RO(L+1,J,N3)+2.0*
& RO(L,HS,N3))+SMP*RO(L,J,N3)
DO 20 H=2,H1

```

```
      U(L,M,N3)=SMP4*(U(L-1,M,N3)+U(L+1,M,N3)+U(L,M-1,N3)+  
$  U(L,M+1,N3))+SMP*U(L,M,N3)  
      V(L,M,N3)=SMP4*(V(L-1,M,N3)+V(L+1,M,N3)+V(L,M-1,N3)+  
$  V(L,M+1,N3))+SMP*V(L,M,N3)  
      P(L,M,N3)=SMP4*(P(L-1,M,N3)+P(L+1,M,N3)+P(L,M-1,N3)+  
$  P(L,M+1,N3))+SMP*P(L,M,N3)  
      RO(L,M,N3)=SMP4*(RO(L-1,M,N3)+RO(L+1,M,N3)+RO(L,M-1,N3)+  
$  RO(L,M+1,N3))+SMP*RO(L,M,N3)  
20 CONTINUE  
      RETURN  
      END
```



```

      RO(L+1,MMAX,N3)=R0D(3)
      GO TO 120
100 IF (L.NE.LJET-1) GO TO 110
      IF ( ICHAR.EQ.1) UOLD=U(L,MMAX,N1)
      U(L,MMAX,N1)=UD(1)
      V(L,MMAX,N1)=VD(1)
      P(L,MMAX,N1)=PD(1)
      RO(L,MMAX,N1)=R0D(1)
      GO TO 120
110 IF (L.NE.LJET) GO TO 120
      U(L-1,MMAX,N1)=UD(2)
      V(L-1,MMAX,N1)=VD(2)
      P(L-1,MMAX,N1)=PD(2)
      RO(L-1,MMAX,N1)=R0D(2)

C
120 U1=U(L,MMAX,N1)
      V1=V(L,MMAX,N1)
      P1=P(L,MMAX,N1)
      R01=R0(L,MMAX,N1)
      U2=U1
      V2=V1
      CALL EOS(1,P1,R01,T,AS,D2,D3)
      A1=SQRT(AS)
      A2=A1
      IF(ICHAR.NE.1) GO TO 130
      U3=U1
      V3=V1
      P3=P1
      R03=R01
      A3=A1
      GO TO 140
130 U3=U(L,MMAX,N3)
      V3=V(L,MMAX,N3)
      P3=P(L,MMAX,N3)
      R03=R0(L,MMAX,N3)
      CALL EOS(1,P3,R03,T,AS,D2,D3)
      A3=SQRT(AS)

C
C      CALCULATE THE PROPERTY INTERPOLATING POLYNOMIAL
C      COEFFICIENTS
C
140 BU=(U1-U(L,M1,N1))*DYR
      BV=(V1-V(L,M1,N1))*DYR
      BP=(P1-P(L,M1,N1))*DYR
      BR0=(R01-RO(L,M1,N1))*DYR
      CU=U1-BU*Y3
      CV=V1-BV*Y3
      CP=P1-BP*Y3
      CR0=R01-BR0*Y3

```

```

C      CALCULATE THE CROSS DERIVATIVE INTERPOLATING POLYNOMIAL
C      COEFFICIENTS
C
DU=(U1-U(L-1,MMAX,N1))*DXR
DV=(V1-V(L-1,MMAX,N1))*DXR
DP=(P1-P(L-1,MMAX,N1))*DXR
DRO=(RO1-RO(L-1,MMAX,N1))*DXR
DU1=(U(L,M1,N1)-U(L-1,M1,N1))*DXR
DV1=(V(L,M1,N1)-V(L-1,M1,N1))*DXR
DP1=(P(L,M1,N1)-P(L-1,M1,N1))*DXR
DRO1=(RO(L,M1,N1)-RO(L-1,M1,N1))*DXR
BDU=(DU-DU1)*DYR
BDV=(DV-DV1)*DYR
BDP=(DP-DP1)*DYR
BDR0=(DRO-DRO1)*DYR
CDU=DU-BDU*Y3
CDV=DV-BDV*Y3
CDP=DP-BDP*Y3
CDRO=DRO-BDR0*Y3

C      CALCULATE Y2
C
ALS=SQR(T*(AL*AL+BE*BE))
UV3=U3*AL+V3*BE+DE
DO 170 ILL=1,3
UV2=U2*AL+V2*BE+DE
Y2=Y3-(UV2+SIGN*ALS*A2+UV3+SIGN*ALS*A3)*DT*0.5
C      INTERPOLATE FOR PROPERTIES
C
U2=3U*Y2+CU
V2=BV*Y2+CV
P2=BP*Y2+CP
RO2=BRO*Y2+CRO
CALL EOS(1,P2,RO2,T,AD,D2,D3)
IF(AD.GT.0.0) GO TO 160
IERR=1
RETURN
160 A2=SQR(T(AD))
170 CONTINUE

C      INTERPOLATE FOR THE CROSS DERIVATIVES
C
DU1=DU
DV1=DV
DP1=DP
DRO1=DRO
DU2=BDU*Y2+CDU
DV2=BDV*Y2+CDV
DP2=BDP*Y2+CDP

```

```

D402=BD40*Y2+CDR0
ATERM2=R02*V2/(Y2/BE)
PSI21=-U1*DU1-DP1/R01
PSI31=-U1*DVI
PSI41=-U1*DP1+A1*A1*U1*DRC1
PSI12=-U2*DRO2-R02*DU2-ATERM2
PSI22=-U2*DU2-DP2/R02
PSI32=-U2*DVI
PSI42=-U2*DP2+A2*A2*U2*DRO2
IF (ICHAE.EQ.1) GO TO 240
C
C      CALCULATE THE CROSS DERIVATIVES AT THE SOLUTION POINT
C
IF (JFLAG.EQ.0) GO TO 220
IF (L.EQ.2) GO TO 220
IF (L.EQ.LJET-1) GO TO 220
GO TO 230
220 DU3=(U(L+1,MMAX,N3)-U3)*DXR
DV3=(V(L+1,MMAX,N3)-V3)*DXR
DP3=(P(L+1,MMAX,N3)-P3)*DXR
DR03=(R0(L+1,MMAX,N3)-R03)*DXR
GO TO 240
230 DU3=(U3-U(L-1,MMAX,N3))*DXR
DV3=(V3-V(L-1,MMAX,N3))*DXR
DP3=(P3-P(L-1,MMAX,N3))*DXR
DR03=(R03-R0(L-1,MMAX,N3))*DXR
C
C      ENTER THE FREE-JET BOUNDARY ITERATION LOOP
C
240 YWI(L)=YW(L)
DO 390 NJ=1,10
IF (ICHAE.EQ.1) GO TO 340
IF (JFLAG.LE.0) GO TO 300
IF (L.EQ.LJET) GO TO 300.
IF (NJ.EQ.1) GO TO 290
IF (NJ.GT.2) GO TO 270
250 YWOLD=YW(L)
POLD=P(L,MMAX,N3)
IF (P(L,MMAX,N3).LT.PE(MMAX)) GO TO 260
YW(L)=YW(L)+DELY
GO TO 280
260 YW(L)=YW(L)-DELY
GO TO 280
270 IF (P(L,MMAX,N3).EQ.PCLD) GO TO 250
DYDP=(YW(L)-YWOLD)/(P(L,MMAX,N3)-POLD)
YWNEW=YW(L)+DYDP*(PE(MMAX)-P(L,MMAX,N3))
YWOLD=YW(L)
POLD=P(L,MMAX,N3)
YW(L)=YWNEW
280 IF (YW(L).LT.(1.0-DYH)*YWOLD) YW(L)=(1.0-DYH)*YWOLD

```

```

IF (YW(L).GT.(1.0+DYN)*YHOLD) YW(L)=(1.0+DYN)*YHOLD
290 NANY(L)=-{YW(L)-YW(L-1})*DXR
XWI(L)=(YW(L)-YWI(L))/DT
XWID=XWI(L)
CALL MAP (1,L,MMAX,AL,BE,DE,LD1,AL1,BE1,DE1)
ALS=SQRT(AL*AL+BE*BE)

C
C      CALCULATE THE PSI TERMS AT THE SOLUTION POINT
C

300 IF (NDIM.EQ.0) GO TO 330
ATERM3=RO3*V3/(1.0/BE)
330 PSI13=-U3*DRO3-R03*D03-ATERM3
PSI23=-U3*D03-DP3/RO3
PSI33=-U3*D3
PSI43=-U3*DP3+A3*A3*U3*DRO3
340 ABR=NANY(L)
ALB=AL/ALS
BEB=BE/ALS
A1B=(A1+A3)*0.5
A2B=(A2+A3)*0.5
RO2B=(R02+R03)*0.5
IF (ICHAR.EQ.1) GO TO 350
PSI21B=(PSI21+PSI23)*0.5+QUT(L,MMAX)
PSI31B=(PSI31+PSI33)*0.5+QVT(L,MMAX)
PSI41B=(PSI41+PSI43)*0.5+QPT(L,MMAX)
PSI12B=(PSI12+PSI13)*0.5+QR0T(L,MMAX)
PSI22B=(PSI22+PSI23)*0.5+QUT(L,MMAX)
PSI32B=(PSI32+PSI33)*0.5+QVT(L,MMAX)
PSI42B=(PSI42+PSI43)*0.5+QPT(L,MMAX)
GO TO 360
350 PSI21B=PSI21+QUT(L,MMAX)
PSI31B=PSI31+QVT(L,MMAX)
PSI41B=PSI41+QPT(L,MMAX)
PSI12B=PSI12
PSI22B=PSI22
PSI32B=PSI32
PSI42B=PSI42

C
C      SOLVE THE COMPATIBILITY EQUATIONS FOR FREE SLIP WALLS
C

360 XWID=0.0
U(L,MMAX,N3)=(U1-ABR*(V1-XWID)+(PSI21B-ABR*PSI31B)*DT)/
$ (1.+ABR*ABR)
V(L,MMAX,N3)=-U(L,MMAX,N3)*ABR+XWID
P(L,MMAX,N3)=P2-SIGN*RO2B*A2B*(ALB*(U(L,MMAX,N3)-U2)+
$ BEB*(V(L,MMAX,N3)-V2))+(PSI42B+A2E*A2B*PSI12B+SIGN*
$ RO2B*A2B*(ALB*PSI22B+BEB*PSI32B))*DT
IF(P(L,MMAX,N3).LT.0.0) P(L,MMAX,N3)=PL0W*PC
RO(L,MMAX,N3)=E01+(P(L,MMAX,N3)-P1-PSI41B*DT)/(A1B*A1B)
IF(RO(L,MMAX,N3).LE.0.0) RO(L,MMAX,N3)=ROLOW/G

```

```

1P ( JFLAG.EQ.0) GO TO 510
IF ( L.LE.LJET-1) GO TO 510
IF ( L.E. LJET-1) GO TO 400
IF ( ICHAB.EQ.1) GO TO 510
IF ( JFLAG.EQ.-1.AND.L.NE.LJET) GO TO 510
IF ( JFLAG.EQ.-1.AND.L.EQ.LJET) GO TO 500
DPL=ABS(1P(L,MMAX,N3)-PE(MMAX))/PE(MMAX)
IF ( DPL.LE.0.001.AND.L.NE.LJET) GO TO 510
IF ( DPL.LE.0.001.AND.L.EQ.LJET) GO TO 500
390 CONTINUE
IF ( L.E. LJET) GO TO 500
GO TO 510
400 UD(3)=U(L,MMAX,N3)
VD(3)=V(L,MMAX,N3)
PD(3)=P(L,MMAX,N3)
ROD(3)=ROD(L,MMAX,N3)
PU(4)=PE(MMAX)
CALL EOS(5,PD(3),ROD(3),TD,AS,D2,D3)
XN1=SQRT((UD(3)*UD(3)+VD(3)*VD(3))/AS)
CALL EOS(5,PD(3),ROD(3),TD,PTD,TID,XN1)
IF ( PE(MMAX).GT.PD(3).AND.XN1.GE.1.0) GO TO 450
CALL EOS(11,PE(MMAX),ROD(4),TD,PD(3),ROD(3),D3)
GO TO 400
C
450 CALL EOS(12,PE(MMAX),ROD(4),TD,PD(3),ROD(3),D3)
460 CALL EOS(12,PE(MMAX),ROD(4),TE,AS,D2,D3)
CALL EOS(10,PE(MMAX),ROD(4),TE,PTD,TID,XM2)
470 CALL EOS(1,PD(4),ROD(4),T,AS,D2,D3)
VMAJ=XN2*SQRT(AS)
UD(4)=VMAJ/SQRT(1.0+NXNY(LJET)*NXNY(LJET))
VJ(4)=-UD(4)*NXNY(LJET)
IF ( JFLAG.EQ.-1) GO TO 510
IF ( XN1.GE.-1.0) GO TO 510
IF ( XN1.GE.1.0) GO TO 510
XMB=(XN1+XN2)/2.0
IF ( XMB.GE.1.0) GO TO 480
DPL=1.0
DPR=1.0
GC TO 490
480 DPL=XN2-1.0
DPR=1.0-XN1
XMB=1.0
490 DPR=DPR+DPL
CALL EOS(16,P(L,MMAX,N3),RO(L,MMAX,N3),TEMP,PTD,TID,
$ XN3)
CALL EOS(16,P(L,MMAX,N3),RO(L,MMAX,N3),TEMP,PTD,TID,
$ XN3)
Q=XM2*SQRT(AS)
DNXY=(DPR*NXNY(LJET)+DPL*NXNY(LJET))/DPL
U(L,MMAX,N3)=Q/SQRT(1.0+DNXY*DNXY)
V(L,MMAX,N3)=-U(L,MMAX,N3)*DNXY
GO TO 510

```

```
500 UD(1)=UD(3)
    VD(1)=VD(3)
    PD(1)=PD(3)
    ROD(1)=ROD(3)
    UD(2)=UD(4)
    VD(2)=VD(4)
    PD(2)=PD(4)
    ROD(2)=ROD(4)
510 CONTINUE
    DKR=DXX
    IF ( JFLAG.EQ.0) RETURN
    IF ( ICHAN.EQ.1) RETURN
    U(LJET-1,MMAX,N1)=UOLD
    IF ( JFLAG.EQ.-1) RETURN
    YW1(LMAX)=YW(LMAX)
    YW(LMAX)=2.0*YW(L1)-YW(L2)
    NXNY(LMAX)=-(YW(LMAX)-YW(L1))*DKR
    XW1(LMAX)=(YW(LMAX)-YW1(LMAX))/DT
    RETURN
    END
```

```

C SUBROUTINE GRAPH(XP1,YP1,NP)
C ****
C THIS SUBROUTINE PLOTS THE VELOCITY VECTORS
C ****
C
C DIMENSION KSI(1000),XP(1000),XP1(50,20),YP1(50,20)
C DIMENSION TIME(1502),PEES1(1502),PRES2(1502),PRES3(1502)
C COMMON/ONESID/UD(4),VD(4),PD(4),ROD(4)
C COMMON/SOLUTN/U(50,20,2),V(50,20,2),P(50,20,2),
C      S RO(50,20,2)
C COMMON/CNTRLC/LMAX,MMAX,NNMAX,TCONV,FDT,GAMMA,RGAS,GAM1,
C      S GAM2,GAM3,L1,L2,M1,M2,DY,DT,ICHAR,N1D,IERR,IUI,IUO,
C      S DXR,DYR,RSTAR,N,N1,N3,RSTARS,G,PC,TC,LC,PLOW,ROLOW,RG,
C      S DXX,XE1
C COMMON/GEMTRL/XW(50),YW(50),NXNY(50),LT,XT,XI,XE,NDIM,
C      S XWI(50),YWI(50)
C COMMON/BCC/PT(20),TT(20),PE(20),IEXTRA,TW(20),ISUPER,
C      S UI(20),VI(20),PI(20),ROI(20),IEX
C COMMON/AV/CAV,NST,LSS,XMU,XLA,RKMU,XRO,QUT(50,20),
C      S QVT(50,20),QPT(50,20),QRQT(50,20)
C COMMON/RV/CMU,CLA,CK,EMU,ELA,EK,CHECK,ITM,TML
C REAL MN3,NXNY,LC
C XW(LMAX+1)=0.0
C YW(LMAX+1)=0.0
C XW(LMAX+2)=1.0
C YW(LMAX+2)=1.0
C CALL VTHICK(2)
C CALL AXIS(0.0,0.0,'AXIAL DISTANCE',-15,6.0,0.0,
C      S XW(LMAX+1),XW(LMAX+2))
C CALL AXIS(0.0,0.0,'RADIUS',7.3,0,90.0,YW(LMAX+1),
C      S YW(LMAX+2))
C CALL VTHICK(4)
C CALL LINE(XW,YW,LMAX,1,0,0)
C CALL SYMBOL(0.75,-1.25,.14,'VELOCITY DISTRIBUTION IN
C      S THE DISCHARGE CHAMBER',0.0,46)
C SCHAMBER',0.0,46)
C EPSIL=0.000001
C DO 2 L=1,LMAX
C DO 2 M=1,MMAX
C Q=U(L,M,NP)*U(L,M,NP)+V(L,M,NP)*V(L,M,NP)
C Q1=SQRT(Q)
C IF(L.EQ.1.AND.M.EQ.1) QMAX=Q1
C IF(Q1.GT.QMAX) QMAX=Q1
C 2 CONTINUE
C DO 3 L=1,LMAX
C DO 3 M=1,MMAX
C IF(L.EQ.1.AND.M.EQ.1) GO TO 3

```

```
Y1=YP1(L,M)
X1=XP1(L,M)
XT1=U(L,M,MP)*0.25/QMAX
YT1=V(L,M,MP)*0.25/QMAX
XT2=ABS(XT1)
YT2=ABS(YT1)
IF(XT2.LT.EPSIL.AND.YT2.LT.EPSIL) GO TO 3
XT=XT1+X1
YT=YT1+YT1
CALL AROHD(X1,Y1,XT,YT,-1,0.075,14)
3 CONTINUE
CALL VTHICK(1)
CALL EOPLLOT(1)
RETURN
END
```

```

C      SUBROUTINE SHOW(TIME,PRES1,PRES2,PRES3,N)
C ****
C      THIS SUBROUTINE PLOTS A FLOW VARIABLE VERSUS TIME
C AND LOCATION
C ****
C
C      DIMENSION KSI(1000),XP(1000),XP1(50,20),YP1(50,20)
C      DIMENSION TIME(1502),PRES1(1502),PRES2(1502),PRES3(1502)
C      COMMON/ONESID/UD(4),VD(4),PD(4),ROD(4)
C      COMMON/SOLUTN/U(50,20,2),V(50,20,2),P(50,20,2),
C      $ RO(50,20,2)
C      COMMON/GENTRL/XW(50),YW(50),NXNY(50),LT,XT,XI,XE,NDIM,
C      $ XWI(50),YWI(50)
C      COMMON/BCC/PT(20),IT(20),PE(20),IEIXTRA,TW(20),ISUPER,
C      $ UI(20),VI(20),PI(20),BCI(20),IEX
C      COMMON/AV/CAV,NST,LSS,XMU,XLA,BKMU,XHO,QUT(50,20),
C      $ QVT(50,20),QPT(50,20),QRCT(50,20)
C      COMMON/RV/CMU,CLA,CK,EMU,ELA,EK,CHECK,ITM,TML
C      REAL MM3,NXNY,LC
C      CALL VTICK(2)
C      CALL NSCALE (TIME,5.0,N,1,1)
C      CALL NSCALE (PRES1,5.,N,1,1)
C      CALL NSCALE (PRES2,5.,N,1,1)
C      CALL NSCALE (PRES3,5.,N,1,1)
C      CALL GRAD (0.,0.,2.5.,1.,2.5.,1.)
C      CALL AXIS (0.,0.,'TIME IN SECONDS',-15,5.,0.,TIME(N+1),
C      $ TIME(N+2))
C      XMAX=AMAX1(PRES1(N+2),PRES2(N+2),PRES3(N+2))
C      XMIN=AMIN1(PRES1(N+1),PRES2(N+1),PRES3(N+1))
C      CALL AXIS (0.,0.,'DISCHARGE PRESSURE IN PSI',25,5.,90.,
C      $ XMIN,XMAX)
C      CALL VTICK(3)
C      CALL LINE(TIME,PRES1,N,1,10,1)
C      CALL LINE(TIME,PRES2,N,1,10,2)
C      CALL LINE(TIME,PRES3,N,1,10,3)
C      CALL EOPLT(1)
C      RETURN
C      END

```

- APPENDIX E -

C*****

**FORTRAN IV LISTING OF THE PROGRAM FOR
THE MAIN AND BARREL CHAMBER FLOW**

C*****

MAIN PROGRAM

THIS COMPUTER PROGRAM IS FOR THE COMPUTATION OF ONE AND TWO DIMENSIONAL, TIME DEPENDENT FLOW REPRESENTATIVE OF A BARREL AND HAIN CHAMBER FLOW OF A GAS GUN

```

COMMON /CTRL/ ICOR,IE,E1,E2,E3,E4,E0,GL,GN,G1,G2,BG1,BG2
COMMON /D0/ XBRL(70),UBRL(70,2),PBRL(70,2),RBRL(70,2)
COMMON /D1/ X4,U4,P4,X4,X5,U5,P5,E5,X6,U6,P6,E6,X7,U7,P7
S ,R7,PA,SE
COMMON /D2/ E3,U4,X8,U0,P8,R8,X3L,U3L,P3L,R3L,X3R,U3R,
$ P3R,R3E,X4L,U4L,P4L,R4L,X4R,U4R,P4R,XP,AREA,BP
COMMON /D3/ DPBRL(70),DUBBL(70),DRBRL(70),NTI1(8),NTI2(8)
S ,NC(14)
COMMON /DU/ X1L,K1R,U1L,U1R,P1L,F1L,R1L,R2L
COMMON /DS/ XPROJ(1010),UPROJ(1010),PBPROJ(1010),
$ XAPROJ(1010),XSHOCK(1010),USHOCK(1010),TIME(1010),
$ XCAB(1010),XFPROJ(1010)
COMMON /D6/ U(50,20,2),V(50,20,2),P(50,20,2),R0(50,20,2)
COMMON /D7/ LMAX,MAX,RCB,LCB,LBR,N,N1,N3,L1,L2,M1,M2,
$ RCB1,RCB2,LCB1,LCB2,LBR1,LBR2,GAMMA,DY,DY,DT,ICHR,DXR,
$ DYR,ASTAR,RSTARS,G,PC,LC,PLOT,ROLOR,BG
COMMON /DB/ XPI(50,20),YP1(50,20),KSI(2000),ETA(2000),
$ PX(2000),PI(2000),XR(50),XH(20),NNY(20),XE1,
$ NDIS,PDS,AP
COMMON /D9/ CAV,XMU,XLA,BKNU,XRO,QUT(50,20),QVT(50,20),
$ UPT(50,20),QROT(50,20),PE(20)
COMMON /D15/ PRBF1(1000),PRBF2(1000),NCOUNT
REAL LP,LA,LO,AUL,APL,ML,MR,BP,MRB,MRP,AP
REAL MNX,MNY,LC,KSI

SET DEFAULT VALUES

MAIN=2
NPC=-1
XSTEP=0.0
JCOUNT=1
NNAX=1000
N1=1
N3=2
LCOUNT=1
TSHOCK=1.0
TB=0.0
IH=-1
BRLE=2.0
NNAX=5000

```

```

RSTARS=0.0
SMACH=0.0
LC=12.0
MDIM=1
GAMMA=1.4
RGAS=53.35
G=32.174
PC=144.0
RG=RGAS*G
PLOW=0.01*PC
ROLOW=0.0001/G
PE(1)=14.7*PC
T=0.0
FDT=0.5
PCHARG=50. E+5
RCHARG=25.0
C BOUNDARY CONDITIONS
BI=PCHARG
COI=RCHARG
PA=103000.
UB=-1.0
ROA=1.22
BI=BI*PC/6894.8
COA=1.22/(G*16.02)
COI=COI/(G*16.02)
XI=0.0
NPLOT=1
PDS=5.0
AP=0.0254*0.0254*3.1415*0.25
UB=0.0
XPNEW=0.0
C
C GEOMETRY OF THE PHYSICAL DOMAIN
C
XE=7.0
LCB=30
LCB=31
LBR=34
MCB=6
HMAX=16
LMAX=40
LI=LMAX-1
LCB1=LCB-1
LCB2=LCB+1
LBR1=LBR-1
LBR2=LBR+1
N1=HMAX-1
MCB1=MCB-1
MCB2=MCB+1
T2=3.0

```

```

IF ( MPC.EQ.1 ) GO TO 432
C
C      CALL GMETY
C
DXRS=DXR*DXR
DYRS=DYR*DYR
LSS=LCB1
C      INITIAL CONDITIONS
DO 19 L=1,LMAX
DO 19 M=1,MMAX
U(L,M,M1)=0.0
U(L,M,M3)=0.0
V(L,M,M1)=0.0
V(L,M,M3)=0.0
19 CONTINUE
DO 15 L=1,LBR
DO 15 M=MCB,MMAX
U(L,M,M1)=0.0001
U(L,M,M3)=0.0001
V(L,M,M3)=0.0001
V(L,M,M1)=0.0001
P(L,M,M1)=BI
P(L,M,M3)=BI
RO(L,M,M1)=COI
RO(L,M,M3)=COI
QUT(L,M)=0.0
QVT(L,M)=0.0
QPT(L,M)=0.0
QROT(L,M)=0.0
15 CONTINUE
DO 16 L=LCB,LMAX
DO 16 M=1,MCB
U(L,M,M1)=0.0001
U(L,M,M3)=0.0001
V(L,M,M3)=0.0001
V(L,M,M1)=0.0001
P(L,M,M1)=14.7*PC
P(L,M,M3)=14.7*PC
RO(L,M,M1)=COA
RO(L,M,M3)=COA
QUT(L,M)=0.0
QVT(L,M)=0.0
QPT(L,M)=0.0
QROT(L,M)=0.0
16 CONTINUE
C
C      GEOMETRY AND DEFAULT VALUES FOR THE BARREL
C
432 FAR=0.0
G1=1.4

```

```

G2=G1
RG1=287.0
RG2=287.0
GO=1.0
GL=1.0
GM=1.0
ICOR=15
E1=0.0001
E2=0.0001
E3=0.0001
E4=0.0001
DB=4.897*0.0254
TB=0.0
DTB=2.E-5
KMAX=70
MP=1.107
LP=0.2
AREA=(3.1415*DB*DB)/4.
X3L=0.08
X3R=X3L+LP
P3R=1.013E+5
R3R=1.22
U3R=0.0001
U3L=U3R
P3L=P3R
R3L=R3R

C
C      INITIAL CONDITIONS
C

DO 71 K=1,KMAX
UBRL(K,1)=0.0001
UBRL(K,2)=0.0001
PBRL(K,1)=1.013E+5
PBRL(K,2)=1.013E+5
RBRL(K,1)=1.22
RBRL(K,2)=1.22
71 CONTINUE
PBRL(KMAX,1)=1.012E+5
RBRL(KMAX,1)=1.15
DS=2.0/(FLOAT(KMAX-1))
X=0.0-DS
DO 700 K=1,KMAX
X=X+DS
XBRL(K)=X
700 CONTINUE
AZ=340.
XE2DX=BRLE-2.0*DS
IF ( NPC.NE.1 ) GO TO 430
ACC=PCHARG*AREA/MP
TSHOCK=2.*AZ/((G1+1.)*ACC)

```

```

USH=A2
XSH=TSHOCK*A2
XSH=XSH+X3R
XSH=X3R+2.*DS
TSHOCK=0.0
TSHOCK=1.0
XSHOCK(1)=XSH
USHOCK(1)=USH
WRITE(6,10) TSHOCK,XSH,USH
DO 72 K=1,10
PBRL(K,1)=PCHARG
PBRL(K,2)=PCHARG
RBRL(K,1)=RCHARG
RBRL(K,2)=RCHARG
72 CONTINUE
P3L=PCHARG
R3L=RCHARG
U3L=0.0001
U3R=U3L
430 FARID=0.0
NPLOT=1
XPROJ(1)=X3L
UPROJ(1)=U3L
PPBPROJ(1)=P3L
TIME(1)=0.0
LCOUNT=1
X1L=X3L
X1R=X3R
U1L=U3L
U1R=U3R
P1L=P3L
P1R=P3R
R1R=R3R
R1L=R3L
R2L=R1R

C
C      ENTER STEP TIME INTEGRATION
C
DO 660 N=1,NMAX
IF ( MPC.EQ.1 ) GO TO 76
DO 450 L=LCB,LMAX
WX=SQRT(1.+NXY(L)*NXY(L))
DO 450 M=1,MCB
WY=SQRT(1.+NYX(M)*NYX(M))
AS=P(L,N,M1)*GAMMA/RO(L,N,M1)
DXDY=WX*WX*DIXRS+WY*WY*DYRS
UPA=ABS(U(L,N,M1))*WX*DIXR+ABS(V(L,N,M1))*WY*DYR+
8 SQRT(AS*DIXY)
IF (L.EQ.LCB.AND.N.EQ.1) UPAN=UPA
IF (UPA.GT.UPAN) UPAN=UPA

```

```

450 CONTINUE
DO 451 L=1,LBR
  WY=SQRT(1.+WXY(L)*WXY(L))
  DO 451 M=MCB,MMAX
    WY=SQRT(1.+WYX(M)+WYX(M))
    AS=P(L,M,M1)*GAMMA/RO(L,M,M1)
    DXYD=WX*WY*DXYR+HY*HY*DYSR
    UPA=ABS(U(L,M,M1))+WX*DXYR+ABS(V(L,M,M1))+HY*DYSR+
    $ SQRT(AS*DXYD)
    IF (UPA.GT.UPAM) UPAM=UPA
451 CONTINUE
DT=FDT/UPAM
DTT=DT/LC
T=T+DTT
TB=TB+DTT

C
C      FIRST STEP INTEGRATION
C
ICHAR=1
CALL INTER(2,LCB,MCB2,M1)
CALL INTER(LBR,L1,1,MCB1)
CALL INTER(LCB2,LBR1,1,M1)
CALL BDRY(UB)
PE(1)=PBRL(2,M1)*PC/6894.2
PIMB=PE(1)*47.876
CALL EXIT(0)

C
C      SECOND STEP INTEGRATION
C
ICHAR=2
CALL INTER(2,LCB,MCB2,M1)
CALL INTER(LBR,L1,1,MCB1)
CALL INTER(LCB2,LBR1,1,M1)
CALL BDRY(UB)
CALL SMOOTH(2,LCB1,MCB2,MMAX)
CALL SMOOTH(LBR2,L1,2,MCB)
CALL SMOOTH(LCB2,LBR1,2,MMAX)
CALL BDRY(UB)
DO 21 L=1,LCB
  U(L,MCB,M3)=--UB
21 CONTINUE
MCB3=MCB-2
U(LBR,MCB,M3)=0.25*(U(LBR1,MCB,M3)+2.0*U(LBR,MCB1,M3)+.
$ U(LBR2,MCB,M3))
V(LBR,MCB,M3)=0.25*(V(LBR1,MCB,M3)+2.0*V(LBR,MCB1,M3)+.
$ V(LBR2,MCB,M3))
P(LBR,MCB,M3)=0.25*(P(LBR1,MCB,M3)+2.0*P(LBR,MCB1,M3)+.
$ P(LBR2,MCB,M3))
RO(LBR,MCB,M3)=0.25*(RO(LBR1,MCB,M3)+2.0*RO(LBR,MCB1,M3)+.
$ RO(LBR2,MCB,M3))

```

```

CALL EXIT ( 0 )
IF ( XPMEN.GT.0.021 ) GO TO 27
XPOLD=XPNEW
VPOLD=UB*3.208
CALL PISTON ( VPOLD,XPOLD,VPNEW,XPNEW )
UB=VPNEW/3.208
IF ( N.EQ.20 ) CALL GRAPH ( N3,T,VPNEW )
IF ( ABS ( XPNEW-XSTEP).LT.0.0254*(XW(LCB)-XW(LCB1))) )
5 GO TO 28
LCB=LCB1
LCB1=LCB-1
LCB2=LCB+1
DO 452 M=1,NCB
U(LCB,M,N1)=U(LCB2,M,N1)
V(LCB,M,N1)=V(LCB2,M,N1)
P(LCB,M,N1)=P(LCB2,M,N1)
RO(LCB,M,N1)=RO(LCB2,M,N1)
U(LCB,M,N3)=U(LCB2,M,N3)
V(LCB,M,N3)=V(LCB2,M,N3)
P(LCB,M,N3)=P(LCB2,M,N3)
RO(LCB,M,N3)=RO(LCB2,M,N3)
452 CONTINUE
XSTEP=XPNEW
WRITE(6,10) PINB,DTT,XPMEN,VPNEW
CALL GRAPH ( N3,T,VPNEW )
GO TO 28
27 UB=0.0
VPNEW=0.0
28 HAR=0.0
IF ( NPLOT.EQ.70 ) CALL GRAPH ( N3,T,VPNEW )
IF ( MAIN.EQ.1 ) GO TO 24
IF ( N.EQ.1 ) GO TO 79
IF ( TB.LT.DTB ) GO TO 89
TB=0.0
79 HAR=0.0
UAV=(U(LMAX,1,N3)+U(LMAX,MCB,N3))*0.5/3.208
ROAV=(RO(LMAX,1,N3)+RO(LMAX,MCB,N3))*0.5*32.194*16.02
PAV=(P(LMAX,1,N3)+P(LMAX,MCB,N3))*0.5*6894.2/PC
UBRL(1,N3)=UAV
RBRL(1,N3)=ROAV
PBRL(1,N3)=PAV
76 IF ( MPC.EQ.1 ) T=T+DTB
CALL BARREL ( DS,IH,MPC,DTB,NPLOT,N1,N3 )
DO 1 K=1,KMAX
DPBRL(K)=(PBRL(K,N3)-PBRL(K,N1))/DTB
DUBRL(K)=(UBRL(K,N3)-UBRL(K,N1))/DTB
DRBRL(K)=(RBRL(K,N3)-RBRL(K,N1))/DTB
1 CONTINUE
NPLOT=NPLOT+1
TIME(NPLOT)=T

```

```

X5L=XPROJ(NPLOT)
X5R=X5L+LP
IF ( X5L.GE.BRLE ) GO TO 890
IF ( T.GE.TSHOCK ) IH=2
IF ( XSHOCK(NPLOT).GE.XE2DX) IH=-1
IF ( X5R.GT.XE2DX ) IH=1
IF ( JCOUNT.EQ.200 ) GO TO 77
IF ( MPC.EQ.1 ) GO TO 84
GO TO 89
77 JCOUNT=1
LCOUNT=1
WRITE(6,88) DT,T
DO 74 K=1,KMAX
X=XBRL(K)*100.
VEL=UBRL(K,N3)
PRES=PBRL(K,N3)*0.001
PRESS=PRES*1000.
RHO=RBRL(K,N3)
CALL THERMO ( PRESS,RHO,A)
XMACH=VEL/A
WRITE(6,110) X,VEL,PRES,RHO,XMACH
74 CONTINUE
WRITE(6,103)
89 HAR=0.0
IF ( MPC.EQ.1 ) GO TO 84
UAV=(U(LMAX,1,N3)+U(LMAX,MCB,N3))*0.5/3.208
ROAV=(RO(LMAX,1,N3)+RO(LMAX,MCB,N3))*0.5*32.194*16.02
PAV=(P(LMAX,1,N3)+P(LMAX,MCB,N3))*0.5*6894.2/PC
UBRL(1,N3)=UAV
RBRL(1,N3)=ROAV
PBRL(1,N3)=PAV
DO 22 L=2,KMAX
UBRL(L,N3)=UBRL(L,N1)+DUBRL(L)*DTT
PBRL(L,N3)=PBRL(L,N1)+DPBRL(L)*DTT
RBRL(L,N3)=RBRL(L,N1)+DRBRL(L)*DTT
22 CONTINUE
C
24 HAR=0.0
IF ( JCOUNT.EQ.10000 ) GO TO 85
JCOUNT=JCOUNT+1
GO TO 84
85 JCOUNT=1
WRITE(6,88) DTT,T
DO 580 L=1,LBR
DO 580 M=MCB,MMAX
VELMAG=SQRT(U(L,M,N3)**2+V(L,M,N3)**2)
XMACH=VELMAG/SQRT(P(L,M,N3)*GAMMA/RO(L,M,N3))
DRES=P(L,M,N3)/PC
RHO=RO(L,M,N3)*G
UP=U(L,M,N3)

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```

VP=V(L,M,M3)
XP=XP1(L,M)
YP=YP1(L,M)
WRITE(6,920) L,M,XP,YP,UP,VP,DRES,RHO,VELMAG,XMACH
580 CONTINUE
DO 581 L=LCB,LMAX
DO 581 M=1,MCB
VELMAG=SQRT(U(L,M,M3)**2+V(L,M,M3)**2)
XMACH=VELMAG/SQRT(P(L,M,M3)*GAMMA/RO(L,M,M3))
DRES=P(L,M,M3)/PC
RHO=RO(L,M,M3)*G
UP=U(L,M,M3)
VP=V(L,M,M3)
XP=XP1(L,M)
YP=YP1(L,M)
WRITE(6,920) L,M,XP,YP,UP,VP,DRES,RHO,VELMAG,XMACH
581 CONTINUE
C PRINT THE SOLUTION
WRITE(6,103)
WRITE(6,101)
WRITE(6,102)
75 PARID=0.0
84 MNN=N1
N1=N3
N3=MNN
660 CONTINUE
C PRINT THE SOLUTIONS
890 WRITE(6,88) DT,T
DSS=AZ*DTB
XPPROJ(1)=0.28
XCAR(1)=0.28
XL1=0.28
DX5=.20
XJ4=0.0
DO 992 L=2,NPLOT
IF (XL1.GE.2.0) DSS=0.0
XL1=XL1+DSS
XCAR(L)=XL1
IF (XJ4.GE.2.0) GO TO 993
XJ4=XPROJ(L)+DX5
XPPROJ(L)=XJ4
GO TO 992
993 XPPROJ(L)=XPPROJ(L-1)
992 CONTINUE
WRITE(6,10) XCAR(NPLOT),XCAR(1)
DO 994 L=1,NPLOT
PBPROJ(L)=PBPROJ(L)*0.001
994 CONTINUE
CALL IDENT
CALL PLOT(4.0,4.0,-3)

```

```

DO 1112 L=1,NPLOT
PREF1(L)=PREF1(L)*0.001
PREF2(L)=PREF2(L)*0.001
1112 CONTINUE
N=NPLOT
CALL VTHICK(3)
CALL NSCALE(PREF1,5.0,N,1,1)
CALL NSCALE(PREF2,5.0,N,1,1)
CALL NSCALE(TIME,4.0,N,1,1)
TIME(N+1)=0.000
TIME(N+2)=0.003
CALL AXIS(0.0,0.0,'TIME (S)',-8,4.0,0.0,TIME(N+1),
$ TIME(N+2))
XMAX=AMAX1(PREF1(N+2),PREF2(N+2))
XMIN=AMIN1(PREF1(N+1),PREF1(N+1))
CALL AXIS(0.0,0.0,'PRESSURE (KPA)',14,5.0,90.0,XMIN,XMAX)
CALL LINE(TIME,PREF1,N,1,20,2)
CALL LINE(TIME,PREF2,N,1,20,0)
CALL VECTOR(0.0,0.0,0.0,6.0)
CALL VECTOR(0.0,6.0,5.0,6.0)
CALL VECTOR(5.0,6.0,5.0,0.0)
CALL VECTOR(5.0,0.0,0.0,0.0)
CALL EOPLOT(1)
CALL SHOW(NPLOT)
IF (TSHOCK.NE.1) CALL SHOW2(NPLOT)
CALL BOJOB
C
C      FORMAT STATEMENTS
C
88 FORMAT (/,5X,'DT=',F12.9,3X,'T=',F12.9,/)
101 FORMAT (10X,'LOCATION',14X,'VELOCITY',13X,'PRESSURE',
$ 8X,'DENSITY',8X,'MACH NUMBER',/)
102 FORMAT (15X,'(CM)',15X,'(M/S)',15X,'(KPA)',10X,'(KG/M3
$ )',15X,'(NO'),/
103 FORMAT (/,11X,'X4L',11X,'U4L',11X,'P4L',11X,'R4L',11X,'
$ 'X4R',11X,'U4R',13X,'P4R',11X,'R4R',/)
920 FORMAT (1H,7X,2I5,4F12.4,F13.5,F12.6,2F12.4)
10 FORMAT (5X,9E14.5)
110 FORMAT (10X,F8.3,14X,F8.3,13X,3(F8.3,8X))
910 FORMAT (1H,25X,4H(CM),7X,4H(CM),6X,5H(M/S),7X,5H(M/S),7X,
$ 5H(KPA),7X,7H(KG/M3),5X,5H(H/S),10X,2HNO)
100 FORMAT (5X,4E14.5)
1190 FORMAT (1H,10X,'-----'
$ ,-----
$ ,-----')
STOP
END

```

SUBROUTINE BARREL (DS, IH, MPC, DTB, NPLOT, N1, N3)

C
C
C
C
C
C

THIS SUBROUTINE IS USED FOR THE CALLING OF THE DIFFERENT
SUBROUTINES NEEDED TO COMPUTE THE FLOW VARIABLES
THROUGHOUT THE BARREL

COMMON /CTRL/ ICOR,IE,E1,E2,E3,E4,GO,GL,GM,G1,G2,RG1,RG2
 COMMON /D0/ XBRL(70),UBRL(70,2),PBRL(70,2),RBRL(70,2)
 COMMON /D1/ X4,U4,P4,R4,X5,U5,P5,R5,X6,U6,P6,R6,X7,U7,P7
 S ,R7,PA,ME
 COMMON /D2/ W3,W4,X8,U8,P8,R8,X3L,U3L,P3L,R3L,X3R,U3R,
 S P3R,R3R,X4L,U4L,P4L,R4L,X4R,U4R,P4R,XP,AREA,MP
 COMMON /D3/ DPBRL(70),DUBRL(70),DRBRL(70),NT1T1(8),NT1T2(8)
 S ,NC(14)
 COMMON /D4/ X1L,X1R,U1L,U1R,P1L,P1R,R1L,R1R,R2L
 COMMON /D5/ XPROJ(1010),UPROJ(1010),PBPROJ(1010),
 S XBPROJ(1010),XSHOCK(1010),USHOCK(1010),TIME(1010),
 S XCAR(1010),XFPROJ(1010)
 COMMON /D15/ PREF1(1000),PREF2(1000),NCOUNT
 REAL LP,LN,LO,MUL,MPL,MRL,MUR,MPR,MRR,MP
 KMAX=70
 T=FLOAT(NPLOT)*DTB
 USH=USHOCK(NPLOT)
 XSH=XSHOCK(NPLOT)
 X1=X1L-DS
 X3DX=X1R+DS
 XL1=X1L
 XR1=X1R
 XX1=XL1+0.2
 XSHDS=XSH-DS
 JCOUNT=1
 IH2=-1
 IH1=0
 DO 840 K=1,KMAX
 IF (IH1.EQ.-1) GO TO 96
 IF (IH2.EQ.-1) GO TO 99
 IF (K.EQ.3) GO TO 840
 IF (IH.NE.-1) GO TO 94
 IF (K.EQ.KMAX) GO TO 810
 94 FAR=0.0

C
C
C
C

COMPUTE THE PROPERTIES OF THE POINTS LOCATED AT THE LHS
OF THE PROJECTILE

IF (XBRL(K).GT.XBRL(1).AND.XBRL(K).LE.X1) GO TO 730
GO TO 740

730 X5=XBRL(K-1)
 X6=XBRL(K)
 X4=X6
 X7=XBRL(K+1)
 U5=UBRL(K-1,N1)
 U6=UBRL(K,N1)
 U7=UBRL(K+1,N1)
 P5=PBRL(K-1,N1)
 P6=PBRL(K,N1)
 P7=PBRL(K+1,N1)
 R5=RBRL(K-1,N1)
 R6=RBRL(K,N1)
 R7=RBRL(K+1,N1)

C CALL POINT (DTB)
 C

UBRL(K,N3)=U4
 PBRL(K,N3)=P4
 RBRL(K,N3)=R4
 IF (K.NE.2) GO TO 840
 IF (MPC.NE.1) GO TO 840
 UBRLL(1,N3)=UBRL(2,N3)
 PBRL(1,N3)=PBRL(2,N3)
 RBRL(1,N3)=RBRL(2,N3)
 GO TO 840

740 IF (XBRL(K).GT.X1.AND.XBRL(K).LT.XL1) GO TO 750
 IF (IH.NE.-1) GO TO 92
 GO TO 760

750 K1=K
 IF (IH.NE.-1) GO TO 92
 GO TO 840

C POINTS LOCATED BETWEEN THE LHS AND THE RHS OF THE
 C THE PROJECTILE
 C

760 IF (XBRL(K).GT.XL1.AND.XBRL(K).LT.XR1) GO TO 840

C LOCATE THE POINT TO BE USED FOR THE RHS OF THE
 C PROJECTILE
 C

IF (XBRL(K).GT.XR1.AND.XBRL(K).LT.X3RDX) GO TO 780
 GO TO 790

780 K2=K
 X5=XBRL(K1-1)
 U5=UBRL(K1-1,N1)
 P5=PBRL(K1-1,N1)
 R5=RBRL(K1-1,N1)
 X7=XBRL(K2+1)
 U7=UBRL(K2+1,N1)
 P7=PBRL(K2+1,N1)

```

      R7=RBRL(K2+1,N1)
      X3L=X1L
      X3R=X1R
      U3L=U1L
      U3R=U1R
      P3L=P1L
      P3R=P1R
      R3L=R1L
      R3R=R1R
C
      CALL MISSILE ( DTB, 2 )
C
      P4L1=P4L*1.0E-5
      P4R1=P4R*1.0E-5
      X4L=X4L*100.
      X4R=X4R*100.
      WRITE ( 6, 10 ) NPLOT,X4L,U4L,P4L1,R4L,P4R1,R4R,T,IH
10   FORMAT ( 5X,I3,5X,7E14.5,I3 )
      NCOUNT=NPLOT
C
      CALL PSABOT(U4L,P4R,P3R,IH)
C
      WRITE(6,1111) PREF1(NCOUNT),PREF2(NCOUNT)
1111 FORMAT(50I,2E14.5)
      XPROJ(NPLOT+1)=X4L
      UPROJ(NPLOT+1)=U4L
      PBProj(NPLOT+1)=P4L
      X1L=X4L
      X1R=X4R
      U1L=U4L
      U1R=U4R
      P1L=P4L
      P1R=P4R
      R1R=R4R
      R1L=R4L
      GO TO 840
790  IF ( XBRL(K).GT.X3RDX) GO TO 800
      GO TO 840
800  X5=XBRL(K-1)
      X6=XBRL(K)
      X4=X6
      X7=XBRL(K+1)
      U5=UBRL(K-1,N1)
      P5=PBRL(K-1,N1)
      R5=RBRL(K-1,N1)
      U6=UBRL(K,N1)
      P6=PBRL(K,N1)
      R6=RBRL(K,N1)
      U7=UBRL(K+1,N1)
      P7=PBRL(K+1,N1)

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```

R7=RBRL(K+1,N1)
CALL POINT (DTB)
UBRL(K,N3)=U4
PBRL(K,N3)=P4
RBRL(K,N3)=R4
GO TO 840
810 UBRL(K1,N3)=0.5*(U4L+UBRL(K1-1,N3))
PBRL(K1,N3)=0.5*(P4L+PBRL(K1-1,N3))
RBRL(K1,N3)=0.5*(R4L+RBRL(K1-1,N3))
UBRL(K2,N3)=0.5*(U4R+UBRL(K2+1,N3))
PBRL(K2,N3)=0.5*(P4R+PBRL(K2+1,N3))
RBRL(K2,N3)=0.5*(R4R+RBRL(K2+1,N3))
M3=K1+1
M4=K2-1
DO 73 M=M3,M4
UBRL(M,N3)=U4L
PBRL(M,N3)=P4L
RBRL(M,N3)=R4L
73 CONTINUE
X5=XBRL(K-1)
U5=UBRL(K-1,N1)
P5=PBRL(K-1,N1)
R5=RBRL(K-1,N1)
X6=XBRL(K)
U6=UBRL(K,N1)
P6=PBRL(K,N1)
R6=RBRL(K,N1)
CALL OPEN (DTB)
UBRL(K,N3)=U4
PBRL(K,N3)=P4
RBRL(K,N3)=R4
GO TO 840
92 HAR=0.0
IF ( LH.EQ.2 ) GO TO 95
C
C THIS STEP IS EXECUTED IF THE RHS OF THE PROJECTILE IS OUT
C
X3L=X1L
X3R=X1R
U3L=U1L
U3R=U1R
P3L=P1L
P3R=P1R
R3L=R1L
R3R=R1R
X5=XBRL(K1-1)
U5=UBRL(K1-1,N1)
P5=PBRL(K1-1,N1)
R5=RBRL(K1-1,N1)

```

C

```

CALL MISILE ( DTB, 1 )

C
P4L1=P4L*1.0E-5
P4R1=P4R*1.0E-5
X41=X4L*100.
X42=X4R*100.
WRITE ( 6, 10 ) NPLOT,X41,U4L,P4L1,R4L,P4R1,R4R,T,IH
XPROJ(NPLOT+1)=X4L
UPROJ(NPLOT+1)=U4L
PBPROJ(NPLOT+1)=P4L
X1L=X4L
X1R=X4R
U1L=U4L
U1R=U4R
P1L=P4L
P1R=P4R
R1R=R4R
R1L=R4L
IH1=1
96 HAB=0.0
UBRL(K,N3)=U1L
PBRL(K,N3)=P1L
RBRL(K,N3)=R1L
GO TO 840
C THE SHOCK HAS OCCURED INSIDE THE BARREL
95 HAB=0.0
IF (JCOUNI.EQ.2) GO TO 111
X3L=X1L
X3R=X1R
U3L=U1L
U3R=U1R
P3L=P1L
P3R=P1R
R3L=R1L
R3R=R1R
X5=XBRL(K1-1)
U5=UBRL(K1-1,N1)
P5=PBRL(K1-1,N1)
R5=RBRL(K1-1,N1)
C
CALL MISILE ( DTB, 1 )

C
P4L1=P4L*1.0E-5
P4R1=P4R*1.0E-5
X41=X4L*100.
X42=X4R*100.
WRITE ( 6, 10 ) NPLOT,X41,U4L,P4L1,R4L,P4R1,R4R,T,IH
XPROJ(NPLOT+1)=X4L
UPROJ(NPLOT+1)=U4L
PBPROJ(NPLOT+1)=P4L

```

```

X1L=X4L
X1R=X1L+0.2
U1L=U4L
U1R=U1L
P1L=P4L
P1R=P4R
R1L=R4L
R1R=R4R
JCOUNT=2
GO TO 840
111 HAR=0.0
C
C      LOCATE POINTS BETWEEN THE LHS AND THE RHS OF THE
C      PROJECTILE
C
IF ( XBRL(K).GT.XX1 ) GO TO 97
UBRL(K,M3)=U1L
PBRL(K,M3)=P1L
RBRL(K,M3)=R1L
GO TO 840
97 HAR=0.0
IF ( XBRL(K).GT.XSHDS ) GO TO 98
UBRL(K,M3)=U1R
PBRL(K,M3)=P1R
RBRL(K,M3)=R1R
GO TO 840
98 IF ( XBRL(K).GT.XSH ) GO TO 99
KS8=K-1
KS5=K+1
KS6=K+2
KS7=K+3
X5=XBRL(KS5)
X6=XBRL(KS6)
X7=XBRL(KS7)
X8=XBRL(KS8)
P5=PBRL(KS5,M1)
P6=PBRL(KS6,M1)
P7=PBRL(KS7,M1)
P8=PBRL(KS8,M1)
R5=RBRL(KS5,M1)
R6=RBRL(KS6,M1)
R7=RBRL(KS7,M1)
R8=RBRL(KS8,M1)
U5=UBRL(KS5,M1)
U6=UBRL(KS6,M1)
U7=UBRL(KS7,M1)
U8=UBRL(KS8,M1)
X3L=XSH
X3R=X3L
X3=X3L

```

```

W3=USH
U3L=U8
P3L=P8
R3L=R8
U3R=U5
P3R=P5
R3R=R5

C      CALL SHOCK ( DTB,X3 )

C
IH2=1
XSH=X4
USH=W4
U2R=U4R
P2R=P4R
R2R=R4R
U2L=U4L
P2L=P4L
R2L=R4L
UBRL(KS8+1,M3)=U4L
PBRL(KS8+1,M3)=P4L
RBRL(KS8+1,M3)=R4L
XSHOCK(NPLOT+1)=XSH
USHOCK(NPLOT+1)=USH
R4R=R2L
P4R=P2L
WRITE(6,10) NPLOT,XSH,USH ,P2L,P2R,R2L,R2R
GO TO 840
C      POINTS LOCATED IN FRONT OF THE SHOCK
99  HAR=0.0
UBRL(K,M3)=U2R
PBRL(K,M3)=P2R
RBRL(K,M3)=R2R
840  CONTINUE
XSHOCK(NPLOT+1)=XSH
USHOCK(NPLOT+1)=USH
X3DS=2.0-3.0*DS
IF ( XSH.GE.X3DS ) IH=-1
RETURN
END

```

SUBROUTINE PSABOT(U4L,P4R,P3R,1H)

THIS SUBROUTINE CALCULATES THE PRESSURE AHEAD OF THE PROJECTILE

```

COMMON /D15/ PREF1(1000),PREF2(1000),NCOUNT
PA=1.031E+5
XMA=U4L/340.
XMAI=1./XMA
SPART=SQRT(2.4*2.4*0.25*0.25+XMAI*XMAI)
P4RI=PA*(1.+1.4*XMA*XMA*(2.4*0.25+SPART))
P4BT=PA*(1.+0.2*XMA)**7.0
P&EF1(NCOUNT)=P4RI
P&EF2(NCOUNT)=P4BT
IF(IH.EQ.-1) RETURN
P3R=P4BT
P4R=P3R
RETURN
END

```



```

X1S=X1
U1=MUL*X1+BUL
P1=MPL*X1+BPL
R1=MRL*X1+BRL
GO TO 10
30 QP=R*A/GO
TP=GL*P1+QP*U1
C
C      LOCATE POINT 3 AND DETERMINE COEFFICIENTS ALONG LINE 34
C
40 IF (ITER.GT.0) GO TO 50
U4=U3
P4=P3
R4=R3
50 U=0.5*(U3+U4)
P=0.5*(P3+P4)
R=0.5*(R3+R4)
CALL THERMO (P,R,A)
LO=1.0/U
X3=X4-DT*GN/LO
IF (ABS(X3-X3S).LT.0.0001) GO TO 60
X3S=X3
U3=MUL*X3+BUL
P3=MPL*X3+BPL
R3=MRL*X3+BRL
GO TO 40
60 AO=A**2/GO
TO=GL*P3-AO*R3
C
C      CALCULATE THE PROPERTIES AT POINT 4, AND TEST FOR
C      CONVERGENCE
C
IF (IE.EQ.0) GO TO 90
I=1
IF (ITER.EQ.0) P4=P6
70 U4=(TP-GL*P4)/QP
R4=(GL*P4-TO)/AO
CALL THERMO (P4,R4,AO)
H4=U4/A4
FF=FLOAT(H4-HE)
IF (ABS(FF).LT.0.00001) GO TO 100
IF (I.GT.1) GO TO 80
I=I+1
P4I=P4
H4I=H4
P4=1.01*P4
GO TO 70
80 SL=(H4-H4I)/(P4-P4I)
P4I=P4
H4I=H4

```

```
P4=P4+ (ME-M4)/SL
GO TO 70
90 P4=PA
U4=(TP-GL*P4)/QP
R4=(GL*P4-TO)/AO
100 IF (ITER.EQ.1) RETURN
IF (ITER.EQ.0) GO TO 110
IF ((ABS(U4-UD).LT.E1*UD).AND.(ABS(R4-RD).LT.E3*RD))
$ RETURN
110 ITER=ITER+1
UD=U4
RD=R4
GO TO 10
END
```

SUBROUTINE MISSLE (DT,IE)

SUBROUTINE MISSILE CALCULATES PROPERTIES OF A MOVING PROJECTILE

REAL LP,LM,LO,MUL,MPL,MLR,MUR,MUR,MP
COMMON/CTRL/ ICOR,IE,E1,E2,E3,E4,GO,GL,GN,G1,G2,RG1,RG2
COMMON /D1/ X4,U4,P4,B4,X5,U5,P5,B5,X6,U6,P6,B6,X7,U7,P7
\$,R7,PA,ME
COMMON /D2/ W3,W4,X8,U8,P8,B8,X3L,U3L,P3L,R3L,X3R,U3R,
\$ P3R,R3R,X4L,U4L,P4L,R4L,X4R,U4R,P4R,XP,AREA,RP

DEFINE INITIAL PROPERTIES AND DETERMINE INTERPOLATING POLYNOMIALS

四百一十五

P4L = P3L

$$BOL = (U5 - U3L) / D_{XJ}$$

BPL=D3L-HPL*EXJL

BR=BJL-BELK3L
IP (IH.EQ.1) GO TO 11

四

ДИКИЙ

$$BUR = 0.7 - MUR * X_7$$

$$R_{\text{RR}} = (R_{32} - R_{17}) / D_{\text{KK}}$$

CONTENTS

CALCULATE ACCELERATION, VELOCITY, AND LOCATION OF THE PROJECTILE

11 IP (IN.R) = 1) CALL PSABOT(04L,P4R,P3R,INR)

```

10 AP=0.5*(P3L+P4L-P3R-P4R)*AREA*GO/HP
UMISLE=U3L+AP*DT
U4L=UMISLE
U=0.5*(U3L+U4L)
LO=1.0/U
DX=DT*GN/LO
X4L=X3L+DX
IF (IH.EQ.1) GO TO 20
X4R=X3R+DX

C
C      LOCATE POINT 1 AND DETERMINE COEFFICIENTS ALONG LINE 14
C

20 G=G1
IF (ITER.GT.0) GO TO 30
U4L=U1
P4L=P1
R4L=R1
30 U=0.5*(U1+U4L)
P=0.5*(P1+P4L)
R=0.5*(R1+R4L)
CALL THERMO (P,R,A)
LP=1.0/(U+A)
X1=X4L-DT*GN/LP
IF (ABS(X1-X1S).LT.0.0001) GO TO 40
X1S=X1
U1=MUL*X1+BUL
P1=MPL*X1+BPL
R1=ML*X1+BRL
GO TO 20
40 QP=R*A/GO
TP=GL*P1+QP*U1
IF (IH.EQ.1) GO TO 70

C
C      LOCATE POINT 2 AND DETERMINE COEFFICIENTS ALONG LINE 24
C

50 G=G2
IF (ITER.GT.0) GO TO 60
U4R=U2
P4R=P2
R4R=R2
60 U=0.5*(U2+U4R)
P=0.5*(P2+P4R)
R=0.5*(R2+R4R)
CALL THERMO (P,R,A)
LM=1.0/(U-A)
X2=X4R-DT*GN/LM
IF (ABS(X2-X2S).LT.0.0001) GO TO 70
X2S=X2
U2=MUR*X2+BUR
P2=MPR*X2+BPR

```

```

R2=AER*X2+BRR
GO TO 50
70 QH=R*A/GO
TH=GL*P2-QH*U2

C
C      DETERMINE COEFFICIENTS ALONG LINE 34
C
IF ( ITER.GT.0) GO TO 80
P4L=P3L
R4L=R3L
IF ( IH.EQ.1 ) GO TO 80
P4R=P3R
R4R=R3R
80 PL=0.5*(P3L+P4L)
RL=0.5*(R3L+R4L)
G=G1
CALL THERMO (PL,RL,AL)
AOL=AL**2/GO
TOL=GL*P3L-AOL*R3L
PR=0.5*(P3R+P4R)
RR=0.5*(R3R+R4R)
IF ( IH.EQ.1 ) GO TO 95
G=G2
CALL THERMO (PR,RR,AR)
AOR=AR**2/GO
TOR=GL*P3R-AOR*R3R

C
C      CALCULATE THE PROPERTIES AT POINT 4 AND TEST FOR
C      CONVERGENCE
C
U4R=UMISLE
P4R=(TH+QH*U4R)/GL
R4R=(GL*P4R-TOR)/AOR
95 FAR=0.0
U4L=UMISLE
P4L=(TP-QP*U4L)/GL
R4L=(GL*P4L-TOL)/AOL
IF ( ITER.EQ.1) RETURN
IF ( ITER.EQ.0) GO TO 90
IF ((ABS(X4L-XD).GT.E4).OR.(ABS(U4L-UD).GT.E1*UD))
5 GO TO 90
IF ((ABS(P4L-PDL).GT.E2*PDL).OR.(ABS(R4L-RDL).GT.E3*RDL))
5 GO TO 90
IF ( IH.EQ.1 ) GO TO 90
IF ((ABS(R4R-RDR).LT.E2*RDR).AND.(ABS(R4L-RDL).LT.E3*RDL
$ )) RETURN
90 ITER=ITER+1
XD=X4L
UD=U4L
PDL=P4L

```

```
RDL=R4L  
IF ( IH.EQ.1 ) GO TO 10  
PDR=P4R  
RDR=R4R  
GO TO 10  
END
```

SUBROUTINE SHCK

```

C
C
C THIS SUBROUTINE CALCULATES THE PROPERTIES ACROSS
C A SHOCK WAVE
C
C
C DEFINE QUASI-STEADY SHOCK WAVE PROPERTIES
REAL LP,LB,LO,MUL,MEL,MUR,MPR,MRR,MP
COMMON/CONTRL/ ICOR,IE,E1,E2,E3,E4,GO,GL,GN,G1,G2,RG1,RG2
COMMON /D0/ XBR1(70),UBRL(70,2),PBRL(70,2),RBRL(70,2)
COMMON /D1/ X4,U4,P4,R4,X5,U5,P5,R5,X6,U6,P6,R6,X7,U7,P7
$ ,R7,PA,ME
COMMON /D2/ U3,W4,X8,U8,P8,R8,X3L,U3L,P3L,R3L,X3R,U3R,
$ P3R,R3R,X4L,U4L,P4L,R4L,X4R,U4R,P4R,XP,AREA,MP
G=G1
V1=H4
CALL THERMO ( P4R,R4R,A1 )
R1=V1/A1
CALCULATE PROPERTIES RATIOS ACROSS SHOCK WAVE FOR A
PERFECT GAS
V2V1=(2.0+(G-1.0)*R1**2)/((G+1.0)*R1**2)
R2R1 =1.0/V2V1
P2P1=2.0*G*R1**2/(G+1.0)-(G-1.0)/(G+1.0)
CALCULATES PROPERTIES BEHIND QUASI -STEADY SHOCK WAVE
V2=V2V1*V1
P4L=P2P1*P4R
R4L=R2R1*E4R
U4L=W4+U4R-V2
RETURN
END

```

SUBROUTINE SHOCK (DT, X3)

C
C
C
C
C
C

THIS SUBROUTINE CALCULATES THE FLOW PROPERTIES AT THE
LEFT AND THE RIGHT SIDES OF A SHOCK WAVE

C
C

```
REAL LP,LM,LO,MUL,MPL,MUR,MPR,MRR,MP
COMMON/CONTRL/ ICOR,IE,E1,E2,E3,E4,GO,GL,GN,G1,G2,RG1,RG2
COMMON /D0/ XBRL(70),UBRL(70,2),PBRL(70,2),RBRL(70,2)
COMMON /D1/ X4,U4,P4,R4,X5,U5,P5,R5,X6,U6,P6,R6,X7,U7,P7
S ,R7,PA,ME
COMMON /D2/ W3,W4,X8,U8,P8,R8,X3L,U3L,P3L,R3L,X3R,U3R,
S P3R,R3R,X4L,U4L,P4L,R4L,X4R,U4R,P4R,XP,AREA,MP
DEFINE INITIAL PROPERTIES AND DETERMINE INTERPOLATING
POLYNOMIALS
ITER=0.0
X1S=X8
U1=U8
P1=P8
R1=R8
DX=X8-X3
MUL=(U8-U3L)/DX
BUL=U3L-MUL*X3
MPL=(P8-P3L)/DX
BPL=P3L-MPL*X3
MRL=(R8-R3L)/DX
BRL=R3L-MRL*X3
LOCATE POINT 4
10 IF ( ITER.EQ.0 ) W4=W3
W=0.5*(W3+W4)
LO=1./W
X4=X3+DT*3.W/LO
CALL INTER AND SHCK TO DETERMINE THE SOLUTION AT POINTS
4L AND 4R
CALL POINT ( DT )
U4R=U4
P4R=P4
R4R=R4
CALL SHCK
LOCATE POINT 1 AND DETERMINE THE COEFFICIENTS ALONG
LINE 14
30 U=0.5*(U4L+U1)
P=0.5*(P4L+P1)
R=0.5*(R4L+R1)
CALL THERMO ( P,R,A )
LP=1.0/(U+A)
X1=X4-DT*GN/LP
```

```
IF ( ABS(X1-X1S) .LT. 0.0001 ) GO TO 40
X1S=X1
U1=MUL*X1+BUL
P1=MPL*X1+BPL
R1=MRL*X1+BRL
GO TO 30
40 QP=R*A/GO
TP =GL*P1+QP*U1
C CALCULATE S4RRO AND TEST FOR CONVERGENCE
P4RRO= (TP-QP*U4L)/GL
DP=P4L-P4RRO
IF ( ABS (DP) .LT. E2*P4L) RETURN
ITER=ITER+1
IF ( ITER.GT.ICOR ) RETURN
IF ( ITER.GT.1 ) GO TO 50
W4I=W4
DPI=DP
W4=1.01*W4
GO TO 10
50 SL=(DP-DP1)/(W4-W4I)
W4I=W4
DPI=DP
W4=W4-DP/SL
GO TO 10
END
```

SUBROUTINE THERMO (P,R,A)

C
C
C
C
C
C

SUBROUTINE THERMO CALCULATES THE SPEED OF SOUND FOR
A PERFECT GAS

COMMON/COMTRL/ ICOR,IE,E1,E2,E3,E4,G0,GL,GN,G1,G2,RG1,RG2
A=SQRT(1.4*P/R)
RETURN
END

SUBROUTINE POINT (DT)

C
C
C
C
C
C

SUBROUTINE INTER CALCULATES THE SOLUTION AT AN INTERIOR
POINT

C
C

```
REAL LP, LM, LO, MUL, MPL, MRL, MUR, MPR, MRR, MP
COMMON/CTRL/ ICOR, IE, E1, E2, E3, E4, GO, GL, GN, G1, G2, RG1, RG2
COMMON /D1/ X4, U4, P4, R4, X5, U5, P5, R5, X6, U6, P6, R6, X7, U7, P7
S ,R7,PA,ME
      DEFINE INITIAL PROPERTIES AND DETERMINE INTERPOLATING
      POLYNOMIALS
ITER=0
X1S=X5
X2S=X7
X3S=X6
U1=U5
P1=P5
R1=R5
U2=U7
P2=P7
R2=R7
U3=U6
P3=P6
R3=R6
DX=X5-X6
MUL=(U5-U6)/DX
BUL=U6-MUL*X6
MPL=(P5-P6)/DX
BPL=P6-MPL*X6
MRL=(R5-R6)/DX
BRL=R6-MRL*X6
DX=X6-X7
MUR=(U6-U7)/DX
BUR=U7-MUR*X7
MPR=(P6-P7)/DX
BPR=P7-MPR*X7
MRR=(R6-R7)/DX
BRB=R7-MRR*X7
```

C
C
C

LOCATE POINT 1 AND DETERMINE COEFFICIENTS ALONG LINE 14

```
10 IF ( ITER.GT.0) GO TO 20
U4=U1
P4=P1
R4=R1
```

```

20 U=0.5*(U1+U4)
P=0.5*(P1+P4)
R=0.5*(R1+R4)
CALL THERMO (P,R,A)
LP=1.0/(U+A)
X1=X4-DT*GM/LP
IF (ABS(X1-X1S).LT.0.0001) GO TO 40
X1S=X1
IF (X1.GT.X6) GO TO 30
U1=MUL*X1+BUL
P1=MPL*X1+BPL
R1=MRL*X1+BRL
GO TO 10
30 U1=MUR*X1+BUR
P1=MPR*X1+BPR
R1=MRR*X1+BRR
GO TO 10
40 QP=R*A/GO
TP=GL*P1+QP*U1
C
C      LOCATE POINT 2 AND DETERMINE COEFFICIENTS ALONG 24
C
50 IF (ITER.GT.0) GO TO 60
U4=U2
P4=P2
R4=R2
60 U=0.5*(U2+U4)
P=0.5*(P2+P4)
R=0.5*(R2+R4)
CALL THERMO (P,R,A)
LM=1.0/(U-A)
X2=X4-DT*GM/LM
IF (ABS(X2-X2S).LT.0.0001) GO TO 80
X2S=X2
IF (X2.GT.X6) GO TO 70
U2=MUL*X2+BUL
P2=MPL*X2+BPL
R2=MRL*X2+BRL
GO TO 50
70 U2=MUR*X2+BUR
P2=MPR*X2+BPR
R2=MRR*X2+BRR
GO TO 50
80 QM=R*A/GO
TM=GL*P2-QM*U2
C
C      LOCATE POINT 3 AND DETERMINE COEFFICIENTS ALONG LINE 34
C
90 IF (ITER.GT.0) GO TO 100
U4=U3

```

```

P4=P3
R4=R3
100 U=0.5*(U3+U4)
P=0.5*(P3+P4)
R=0.5*(R3+R4)
CALL THERMO (P,R,A)
IF (ABS(U).GT.1.0E-6) GO TO 110
LO=0.0
X3=X4
GO TO 120
110 LO=1.0/U
X3=X4-DT*GM/LO
120 IF (ABS (X3-X3S).LT.0.0001) GO TO 140
X3S=X3
IF (X3.GT.X6) GO TO 130
U3=MUL*X3+BUL
P3=MPL*X3+BPL
R3=MRL*X3+BRL
GO TO 90
130 U3=MUR*X3+BUR
P3=MPH*X3+BPR
R3=MRB*X3+BRR
GO TO 90
140 AO=A**2/G0
TO=GL*P3-AO*R3
C
C      CALCULATE THE PROPERTIES AT POINT 4, AND TEST FOR
C      CONVERGENCE
C
U4=(TP-TM)/(QP+QB)
P4=(TP-QP*U4)/GL
R4=(GL*P4-TO)/AO
IF (ITER.EQ.1C0R) RETURN
IF (ITER.EQ.0) GO TO 150
IF ((ABS(U4-UD).GT.E1*UD).OR.(ABS(P4-PD).GT.E2*PD)) 3
GO TO 150
IF (ABS(R4-RD).LT.E3*RD) RETURN
150 ITER=ITER+1
UD=U4
PD=P4
RD=R4
GO TO 10
END

```

SUBROUTINE SH02 (N)

```
C
C
C
C
```

THIS SUBROUTINE PLOTS THE PROJECTILE CHARACTERISTICS

```
C
C
```

```
COMMON /D5/ XPROJ(1010), UPROJ(1010), PBPROJ(1010),
$ XBPROJ(1010), XSHOCK(1010), USHOCK(1010), TIME(1010),
$ XCAB(1010), XPPROJ(1010)
CALL VTHICK(3)
CALL VECTOR(0.0,0.0,0.0,6.0)
CALL VECTOR(-0.5,0.0,-0.5,6.0)
CALL VECTOR(-1.0,0.0,-1.0,6.0)
CALL VECTOR(-1.0,6.0,4.5,6.0)
CALL VECTOR(4.5,6.0,4.5,0.0)
CALL VECTOR(4.5,0.0,-1.0,0.0)
CALL NSCALE(TIME,4.0,N,1,1)
CALL NSCALE(XSHOCK,5.0,N,1,1)
TIME(N+1)=0.000
TIME(N+2)=0.001
CALL AXIS(0.0,0.0,'TIME (S)',-8,4.0,0.0,TIME(N+1),
$ TIME(N+2))
CALL AXIS(0.0,0.0,'DISPLACEMENT (M)',16,5.0,90.0,
$ XSHOCK(N+1),XSHOCK(N+2))
CALL LINE(TIME,XSHOCK,N,1,0,0)
CALL VTHICK(0)
CALL NSCALE(TIME,4.0,N,1,1)
CALL NSCALE(USHOCK,5.0,N,1,1)
TIME(N+1)=0.000
TIME(N+2)=0.001
CALL AXIS(0.0,0.0,'TIME (S)',-8,4.0,0.0,TIME(N+1),
$ TIME(N+2))
CALL VTHICK(3)
CALL AXIS(-0.5,0.0,'VELOCITY (M/S)',14,5.0,90.0,
$ USHOCK(N+1),USHOCK(N+2))
CALL LINE(TIME,USHOCK,N,1,0,0)
CALL EOPLOT(1)
RETURN
END
```

SUBROUTINE SHOW (N)

C
C
C
C
C

THIS SUBROUTINE PLOTS A PROJECTILE VARIABLE VESUS TIME

```

COMMON /DS/ XPROJ(1010), UPROJ(1010), PBPROJ(1010),
$ XBPProj(1010), XSHOCK(1010), USHOCK(1010), TIME(1010),
$ XCAR(1010), XFPProj(1010)
CALL VTHICK(3)
CALL VECTOR(0.0,0.0,0.0,6.0)
CALL VECTOR(-0.5,0.0,-0.5,6.0)
CALL VECTOR(-1.0,0.0,-1.0,6.0)
CALL VECTOR(-1.0,6.0,4.5,6.0)
CALL VECTOR(4.5,6.0,4.5,0.0)
CALL VECTOR(4.5,0.0,-1.0,0.0)
CALL NSCALE(TIME,4.0,N,1,1)
CALL NSCALE(XPROJ,5.0,N,1,1)
TIME(N+1)=0.000
TIME(N+2)=0.001
TIME(N+2)=0.003
CALL AXIS(0.0,0.0,'TIME (S)',-8,4.0,0.0,TIME(N+1),
$ TIME(N+2))
CALL AXIS(0.0,0.0,'DISPLACEMENT (M)',16,5.0,90.0,
$ XPROJ(N+1),XPROJ(N+2))
CALL LINE(TIME,XPROJ,N,1,0,0)
CALL VTHICK(0)
CALL NSCALE(TIME,4.0,N,1,1)
CALL NSCALE(UPROJ,5.0,N,1,1)
TIME(N+1)=0.000
TIME(N+2)=0.001
TIME(N+2)=0.003
CALL AXIS(0.0,0.0,'TIME (S)',-8,4.0,0.0,TIME(N+1),
$ TIME(N+2))
CALL VTHICK(3)
CALL AXIS(-0.5,0.0,'VELOCITY (M/S)',14,5.0,90.0,
$ UPROJ(N+1),UPROJ(N+2))
CALL LINE(TIME,UPROJ,N,1,0,0)
CALL VTHICK(0)
CALL NSCALE(TIME,4.0,N,1,1)
CALL NSCALE(PBProj,5.0,N,1,1)
TIME(N+1)=0.000
TIME(N+2)=0.001
TIME(N+2)=0.003
CALL AXIS(0.0,0.0,'TIME (S)',-8,4.0,0.0,TIME(N+1),
$ TIME(N+2))
CALL VTHICK(3)
CALL AXIS(-1.0,0.0,'PRESSURE (KPA)',14,5.0,90.0,
$ XCAR(N+1),XCAR(N+2))

```

```
S PBPProj (N+1),PBPProj (N+2))
S PBPProj (N+2))
CALL LINE(TIME,PBPProj,N,1,0,0)
CALL EOPLOT(1)
RETURN
END
```

```

SUBROUTINE INTER(LS,LF,MS,MF)
C
C
C THIS SUBROUTINE CALCULATES THE INTERIOR MESH POINTS
C
C
COMMON/D6/ U(50,20,2),V(50,20,2),P(50,20,2),R0(50,20,2)
COMMON/D7/ LMAX,MMAX,MCB,LCB,LBR,N,N1,N3,L1,L2,N1,N2,
$ MCB1,MCB2,LCB1,LCB2,LBR1,LBR2,GAMMA,DX,DY,DT,ICHAR,DXR,
$ DYR,RSTAR,RSTARS,G,PC,LC,PLOW,BOLOW,RG
COMMON/D8/ XP1(50,20),YP1(50,20),KSI(2000),ETA(2000),
$ PI(2000),PY(2000),XW(50),YW(20),NXNY(20),XE1,
$ NDIM,PDS,AP
COMMON/D9/CAV,XHU,XLA,RKHU,XRO,QUT(50,20),QVT(50,20),
$ QPT(50,20),QRQT(50,20),PE(20)
REAL NXNY,NYNX,LC,KSI
AL=0.0
BE=1.0
DE=0.0
DP=0.0
ATERM=0.
IF(ICHAR.NE.1) GO TO 40
C
C COMPUTE THE TENTATIVE SOLUTION AT T+DT
C
DO 30 L=LS,LF
M=SQRT(1.+NXNY(L)*NXNY(L))
DO 30 M=MS,MF
W=SQRT(1.+NYNX(M)+NYNX(M))
UB=U(L,M,N1)
VB=V(L,M,N1)
PB=P(L,M,N1)
ROB=R0(L,M,N1)
ASB=P(L,M,N3)*GAMMA/R0(L,M,N3)
UB=UB+DP
IF(M.NE.1) GO TO 10
C
C COMPUTATION OF THE MIDPLANE MESH POINTS
C
C COMPUTATION OF THE DERIVATIVES WITH RESPECT TO X
C
DUDX=(UB-U(L-1,M,N1))*DXR
DPDX=(PB-P(L-1,M,N1))*DXR
DRDX=(ROB-R0(L-1,M,N1))*DXR
C

```

```

C COMPUTATION OF THE DERIVATIVES WITH RESPECT TO X
C
DUDY= (4.0*V(L,2,N1)-V(L,3,N1))*-S*DYR
V(L,M,N3)=0.0
URHS=-U1B*DUDX-M*DPDX/ROB+QUT(L,M)
RORHS=-U1B*DRODX-ROB*M*DUDX-FLOAT(1+NDIM)*ROB*DUDY
S *BE+QROT(L,M)
PRHS=-U1B*DPDX+ASB*(RORHS+U1B*DRODX)+QPT(L,M)
GO TO 20
10 IF (NDIM.EQ.1) ATERM=ROB*VB/YW(M)

C COMPUTATION OF THE DERIVATIVES WITH RESPECT TO X
C
UVB=UB*AL+VB*WY+DE
DUDX=(UB-U(L-1,M,N1))*DXR
DUDY=(VB-V(L-1,M,N1))*DXR
DPDX=(PB-P(L-1,M,N1))*DXR
DRODX=(ROB-RO(L-1,M,N1))*DXR
DUDY=(UB-U(L,M-1,N1))*DYR
DUDY=(VB-V(L,M-1,N1))*DYR
DPDY=(PB-P(L,M-1,N1))*DYR
DRODY=(ROB-RO(L,M-1,N1))*DYR
URHS=-U1B*DUDX-UVB*DUDY-(M*DPDX+AL*DPDY)/ROB+QUT(L,M)
VRHS=-U1B*DUDX-UVB*DUDY-WY*DPDY/ROB+QVT(L,M)
RORHS=-U1B*DRODX-UVB*DRODY-ROB*(M*DUDX+AL*DUDY+WY*DUDY)
S -ATERM+QROT(L,M)
PRHS=-U1B*DPDX-UVB*DPDY+ASB*(RORHS+U1B*DRODX+UVB*DRODY)
S +QPT(L,M)
V(L,M,N3)=V(L,M,N1)+VRHS*DT
20 U(L,M,N3)=U(L,M,N1)+URHS*DT
P(L,M,N3)=P(L,M,N1)+PRHS*DT
RO(L,M,N3)=RO(L,M,N1)+RORHS*DT
IF (P(L,M,N3).LE.0.0) P(L,M,N3)=PLOW
IF (RO(L,M,N3).LE.0.0) RO(L,M,N3)=ROLOW
30 CONTINUE
RETURN

C COMPUTE THE FINAL SOLUTION AT T+DT
C
40 DO 70 L=LS,LF
H=SQRT(1.+MXNY(L)*MYNX(L))
DO 70 M=MS,MF
HY=SQRT(1.+MYNX(M)*MYNX(M))
UB=U(L,M,N3)
VB=V(L,M,N3)
PB=P(L,M,N3)
ROB=RO(L,M,N3)
ASB=P(L,M,N3)*GAMMA/RO(L,M,N3)

```

```

U1B=U*UB+DP
IF (M.NE.1) GO TO 50
DUDX=(U(L+1,N,N3)-UB)*DXR
DVDR=(V(L+1,N,N3)-VB)*DXR
DPDX=(P(L+1,N,N3)-PB)*DXR
DRODX=(RO(L+1,N,N3)-ROB)*DXR
DVDR=(4.0*V(L,2,N3)-V(L,3,N3))*5*DXR
V(L,N,N3)=0.0
URHS=-U1B*DUDX-U*DPDX/ROB+QVT(L,N)
ROHS=-U1B*DRODX-ROB*V*DUDX-FLOAT(1+NDIM)*ROB*DVDR
$ *BE+QROT(L,N)
PANS=-U1B*DPDX+ASB*(ROHS+U1B*DRODX)+QPT(L,N)
GO TO 60

C
50 IF (NDIM.EQ.1) ATERR=ROB*VB/VB/VN(N)
UVB=UB*AL+VB*AN+DE
DUDX=(U(L+1,N,N3)-UB)*DXR
DPDX=(P(L+1,N,N3)-PB)*DXR
DVDR=(V(L+1,N,N3)-VB)*DXR
DRODX=(RO(L+1,N,N3)-ROB)*DXR
DVDR=(U(L,N+1,N3)-UB)*DVDR
DPDX=(P(L,N+1,N3)-PB)*DVDR
DRODX=(RO(L,N+1,N3)-ROB)*DVDR

C
URHS=-U1B*DUDX-UVB*DUDY-(W*DPDX+AL*DPDY)/ROB+QVT(L,N)
VRHS=-U1B*DVDR-UVB*DUDY-W*DPDY/ROB+QVT(L,N)
ROHS=-U1B*DRODX-UVB*DRODY-ROB*(W*DUDX+AL*DUDY+W*DUDY)
$ -ATERB+QROT(L,N)
PRHS=-U1B*DPDX-UVB*DPDY+ASB*(ROHS+U1B*DUDX+UVB*DRODY)
$ +OPT(L,N)
V(L,N,N3)=(V(L,N,N1)+V(L,N,N3)+VRHS*DT)*.5
U(L,N,N3)=(U(L,N,N1)+U(L,N,N3)+URHS*DT)*.5
P(L,N,N3)=(P(L,N,N1)+P(L,N,N3)+PRHS*DT)*.5
RO(L,N,N3)=(RO(L,N,N1)+RO(L,N,N3)+ROHS*DT)*.5
IF(P(L,N,N3).LE.0.0) P(L,N,N3)=PLOW
IF(RO(L,N,N3).LE.0.0) RO(L,N,N3)=ROLOW
70 CONTINUE
RETURN
END

```

SUBROUTINE BBY (OB)

THIS SUBROUTINE CALCULATES THE BOUNDARY ABSI POINTS

```

COMMON/D7/ LMAX,NMAX,ACB1,ACB2,LCB1,LCB2,LBR1,LBR2,GAMMA,DY,DZ,DT,
      DVB,RSTAR,RSTAR5,G,PC,LC,PLOW,ROLOW,RG
COMMON/D8/ XPI(50,20),YPI(50,20),KSI(2000),ETA(2000),
      PX(2000),PY(2000),XH(50),YH(20),NMN(20),XE,
      NDIN,PDS,AP
COMMON/D9/ CAV,XNU,XLA,RKNU,IRO,QUT(50,20),QVZ(50,20),
      QPT(50,20),QROT(50,20),PE(20)
REAL NMV,NVN,LC,KSI
DO 22 L=LCB2,LBR1
      U(L,ACB,NJ)=0.0
22 CONTINUE
DO 1 M=NCB,B1
      U(1,N,NJ)=0.0
      V(1,N,NJ)=V(2,N,NJ)
      P(1,N,NJ)=P(2,N,NJ)
      RO(1,N,NJ)=RO(2,N,NJ)
1 CONTINUE
DO 3 N=1,BCB
      U(LCB,N,NJ)=UB
      V(LCB,N,NJ)=V(LCB2,N,NJ)
      P(LCB,N,NJ)=P(LCB2,N,NJ)
      RO(LCB,N,NJ)=RO(LCB2,N,NJ)
3 CONTINUE
DO 4 N=NCB2,NMAX
      U(LBR,N,NJ)=0.0
      V(LBR,N,NJ)=V(LBR1,N,NJ)
      P(LBR,N,NJ)=P(LBR1,N,NJ)
      RO(LBR,N,NJ)=RO(LBR1,N,NJ)
4 CONTINUE
DO 5 L=2,LCB1
      U(L,ACB,NJ)=UB
      V(L,ACB,NJ)=0.0
      P(L,ACB,NJ)=P(L,NCB2,NJ)
      RO(L,ACB,NJ)=RO(L,NCB2,NJ)
5 CONTINUE
DO 6 L=1,LBR
      U(L,NMAX,NJ)=U(L,N1,NJ)
      V(L,NMAX,NJ)=0.0
      P(L,NMAX,NJ)=P(L,N1,NJ)
      RO(L,NMAX,NJ)=RO(L,N1,NJ)

```

```
6 CONTINUE
DO 7 L=LBR,L1
U(L,MCB,N3)=U(L,MCB1,N3)
P(L,MCB,N3)=P(L,MCB1,N3)
R0(L,MCB,N3)=R0(L,MCB1,N3)
V(L,MCB,N3)=0.0
7 CONTINUE
RETURN
END
```

SUBROUTINE SMOOTH (LS,LF,MS,KMAX)

C
C
C
C
C

THIS SUBROUTINE IS USED TO SMOOTH THE FLOW VARIABLES

```
COMMON/D6/ U(50,20,2),V(50,20,2),P(50,20,2),RO(50,20,2)
COMMON/D7/ LMAX,MMAX,MCB,LCB,LBR,N,N1,N3,L1,L2,N1,N2,
$ MCB1,MCB2,LCB1,LCB2,LBR1,LBR2,GAMMA,DX,DY,DT,ICHAR,DXR,
$ DYL,RSTAR,RSTARS,G,PC,LC,PLOW,ROLOW,RG
COMMON/D8/ XP1(50,20),YP1(50,20),KSI(2000),ETA(2000),
$ PX(2000),PY(2000),XW(50),YW(20),NXNY(20),XE1,
$ NDIM,PDS,AP
COMMON/D9/CAV,XMU,XLA,RKMU,XRO,QUT(50,20),QVT(50,20),
$ QPT(50,20),QRROT(50,20),PE(20)
REAL NXNY,NYMX,LC,KSI
```

C

```
K1=KMAX-1
J=MS-1
SMP=0.50
SMP=0.15
SMP=0.95
SMP=0.25
SMP4=0.25*(1.0-SMP)
DO 20 L=LS,LF
  U(L,KMAX,N3)=SMP4*(U(L-1,KMAX,N3)+U(L+1,KMAX,N3)
$ +2.0*U(L,K1,N3))+SMP*U(L,KMAX,N3)
  V(L,KMAX,N3)=-U(L,KMAX,N3)*NXNY(L)
  P(L,KMAX,N3)=SMP4*(P(L-1,KMAX,N3)+P(L+1,KMAX,N3)
$ +2.0*P(L,K1,N3))+SMP*P(L,KMAX,N3)
  RO(L,KMAX,N3)=SMP4*(RO(L-1,KMAX,N3)+RO(L+1,KMAX,N3)
$ +2.0*RO(L,K1,N3))+SMP*RO(L,KMAX,N3)
  U(L,J,N3)=SMP4*(U(L-1,J,N3)+U(L+1,J,N3)+2.0*U(L,MS,N3))
$ +SMP*U(L,J,N3)
  V(L,J,N3)=SMP4*(V(L-1,J,N3)+V(L+1,J,N3)+2.0*V(L,MS,N3))
$ +SMP*V(L,J,N3)
  P(L,J,N3)=SMP4*(P(L-1,J,N3)+P(L+1,J,N3)+2.0*P(L,MS,N3))
$ +SMP*P(L,J,N3)
  RO(L,J,N3)=SMP4*(RO(L-1,J,N3)+RO(L+1,J,N3)+2.0
$ *RO(L,MS,N3))+SMP*RO(L,J,N3)
  DO 20 N=MS,K1
    U(L,N,N3)=SMP4*(U(L-1,N,N3)+U(L+1,N,N3)+U(L,N-1,N3)
$ +U(L,N+1,N3))+SMP*U(L,N,N3)
    V(L,N,N3)=SMP4*(V(L-1,N,N3)+V(L+1,N,N3)+V(L,N-1,N3)
$ +V(L,N+1,N3))+SMP*V(L,N,N3)
    P(L,N,N3)=SMP4*(P(L-1,N,N3)+P(L+1,N,N3)+P(L,N-1,N3)
$ +P(L,N+1,N3))+SMP*P(L,N,N3)
    RO(L,N,N3)=SMP4*(RO(L-1,N,N3)+RO(L+1,N,N3)+RO(L,N-1,N3)
$ +RO(L,N+1,N3))
```

```
      S RO(L,M+1,M3) + SMP*RO(L,M,M3)
20  CONTINUE
     RETURN
     END
```

SUBROUTINE PISTON (VPOLD,XPOLD,VPNEW,IPNEW)

C
C
C
C
C

THIS SUBROUTINE CALCULATES THE PISTON VARIABLES

```
COMMON/D6/ U(50,20,2),V(50,20,2),P(50,20,2),R0(50,20,2)
COMMON/D7/ LMAX,MMAX,MCB,LCB,LBR,N,N1,N3,L1,L2,N1,N2,
$ MCB1,MCB2,LCB1,LCB2,LBR1,LBR2,GAMMA,DX,DY,DT,ICHAR,DXR,
$ DYR,RSTAR,RSTARS,G,PC,LC,PLOW,ROLOW,RG
COMMON/D8/ XP1(50,20),YP1(50,20),KSI(2000),ETA(2000),
$ PX(2000),PY(2000),XW(50),YW(20),MXNY(20),XE1,
$ MDIM,PDS,AP
COMMON/D9/CAV,XHU,XLA,RKMU,XRO,QUT(50,20),QVT(50,20),
$ QPT(50,20),QROT(50,20),PE(20)
REAL MXNY,MYNX,LC,KSI
PA=1.031E+5
PAV=P(LCB,1,N3)
DO 1 N=2,MCB
PAV=PAV+P(LCB,N,N3)
1 CONTINUE
PAV=PAV*6894.2/FLOAT(MCB)
DELTAV=AP*(PAV-PA)*DT/(PDS*LC)
DETERMINE THE PISTON VELOCITY
VPNEW=VPOLD+DELTAV
DETERMINE THE PISTON DISPLACEMENT
XPNEW=XPOLD+VPOLD*DT/LC+DELTAV*DT/(2.0*LC)
RETURN
END
```

C
C

SUBROUTINE GRAPH (MP,T,VPNEH)

```

C
C
C THIS SUBROUTINE PLOTS THE VELOCITY VECTORS
C AND PLOTS THE PHYSICAL GRID
C
C
COMMON/D6/ U(50,20,2), V(50,20,2), P(50,20,2), R0(50,20,2)
COMMON/D7/ LMAX,MMAX,MCB,LCB,LBR,N,N1,N3,L1,L2,M1,M2,
$ MCB1,MCB2,LCB1,LCB2,LBR1,LBR2,GAMMA,DX,DY,DT,ICHAR,DXR,
$ DYR,RSTAR,RSTARS,G,PC,LC,PLOW,ROLOW,RG
COMMON/D8/ XP1(50,20), YP1(50,20), KSI(2000), ETA(2000),
$ PI(2000), PY(2000), XW(50), YW(20), MXNY(20), XE1,
$ NDIM,PDS,AP
COMMON/D9/CAV,XMU,XLA,RKMU,XRO,QUT(50,20), QVT(50,20),
$ QPT(50,20), QROT(50,20), PE(20)
REAL MXNY, NYNX, LC, KSI
CALL RECT(-2.45,-2.55,8.5,11.0,0.0,3)
DO 220 L=1,LMAX
IF ( L.LT.LCB) GO TO 6
IF ( L.LE.LBR ) GO TO 8
J1=1
JMAX=MCB
GO TO 7
8 HAB=0.0
J1=1
JMAX=MMAX
GO TO 7
6 HAB=0.0
J1=MCB
JMAX=MMAX
7 CALL VECTOR ( XP1(L,J1), YP1(L,J1), XP1(L,JMAX),
$ YP1(L,JMAX))
220 CONTINUE
DO 230 L=1,LBR1
DO 230 M=MCB,MMAX
CALL VECTOR ( XP1(L,M), YP1(L,M), XP1(L+1,M), YP1(L+1,M))
230 CONTINUE
DO 240 L=LCB,L1
DO 240 M=1,MCB
CALL VECTOR ( XP1(L,M), YP1(L,M), XP1(L+1,M), YP1(L+1,M))
240 CONTINUE
CALL RECT(0.0,0.0,YW(MCB),XW(LCB),0.0,3)
CALL PATTERN(0.0,0.0,YW(MCB),XW(LCB),0.03,3)
PPN=FLOAT(N)
PPN1=T*1000.
YW(LMAX+1)=0.0
YW(MMAX+1)=0.0
YW(LMAX+2)=2.50

```

```

YB(MMAX+2)=2.50
CALL VTHICK(2)
CALL AXIS(0.0,0.0,'AXIAL DISTANCE (CM.)',-20,7.0,0.0,
S XW(LMAX+1),XW(LMAX+2))
CALL AXIS(0.0,0.0,'RADIUS (CM.)',12,3.0,90.0,
S YW(MMAX+1),YW(MMAX+2))
CALL SYMBOL(0.00,-1.40,.14,' FIGURE VELOCITY
S DISTRIBUTION IN THE MAIN CHAMBER',0.0,51)
CALL SYMBOL(3.25,-.875,.14,'TIME STEP',0.0,9)
CALL NUMBER(5.00,-.875,.14,PPN,0.0,-1)
CALL SYMBOL(0.1,4.5,.14,'TIME',0.0,5)
CALL SYMBOL(2.0,4.5,.14,'(MS)',0.0,4)
CALL NUMBER(1.1,4.5,.14,PPN1,0.0,2)
CALL SYMBOL(0.1,5.0,.14,'PISTON VELOCITY=',0.0,16)
CALL NUMBER(2.5,5.0,.14,VPNEW,0.0,2)
CALL SYMBOL(3.25,5.0,.14,'(M/S)',0.0,5)
CALL SYMBOL(0.1,4.0,.14,'CHARGE PRESSURE=5,000.KPA',
S 0.0,25)
EPSIL=0.000001
DO 2 L=1,LMAX
DO 2 M=1,MMAX
Q=U(L,M,MP)*U(L,M,MP)+V(L,M,MP)*V(L,M,MP)
Q1=SQRT(Q)
IF(L.EQ.1.AND.M.EQ.1) QMAX=Q1
IF(Q1.GT.QMAX) QMAX=Q1
2 CONTINUE
DO 3 L=1,LMAX
DO 3 M=1,MMAX
IF(L.EQ.1.AND.M.EQ.1) GO TO 3
X1=YP1(L,M)
X1=XP1(L,M)
XT1=U(L,M,MP)*0.25/QMAX
YT1=V(L,M,MP)*0.25/QMAX
XT2=ABS(XT1)
YT2=ABS(YT1)
IF(XT2.LT.EPSIL.AND.YT2.LT.EPSIL) GO TO 3
XT=XT1+X1
YT=YT1+Y1
CALL VTHICK(3)
CALL AROHD(X1,Y1,XT,YT,-1,0.075,14)
3 CONTINUE
CALL VTHICK(1)
CALL EOPLOT(1)
RETURN
END

```

```

C SUBROUTINE EXIT ( IEXTRA )
C -----
C THIS SUBROUTINE CALCULATES THE EXIT MESH POINTS
C -----
C
COMMON/D6/ U(50,20,2),V(50,20,2),P(50,20,2),RO(50,20,2)
COMMON/D7/ LMAX,NMAX,MCB,LCB,LBR,N,N1,N3,L1,L2,M1,M2,
S MCB1,MCB2,LCB1,LCB2,LBR1,LBR2,GAMMA,DX,DY,DT,ICHAR,DXR,
S DYL,RSTAR,RSTARS,G,PC,LC,PLOW,ROLOH,RG
COMMON/D8/ XP1(50,20),YP1(50,20),KSI(2000),ETA(2000),
S PX(2000),PY(2000),XW(50),YW(20),NINI(20),IE1,
S NDIM,PDS,AP
COMMON/D9/CAV,XMU,XLA,RKMU,KRO,QUT(50,20),QVT(50,20),
S QPT(50,20),QROT(50,20),PE(20)
REAL MXNY,NINX,LC,KSI
ME=MCB
AL=0.0
BE=1.0
DE=0.0
AL1=0.0
BE1=1.0
DE1=0.0
IE1=0.0
X3=XE1
DO 1 M=1,ME
PE(M)=PE(1)
1 CONTINUE
ATERM2=0.0
ATERM3=0.0
DO 180 N=1,ME
IF(IEXTRA.EQ.1) GO TO 10
AS=P(LMAX,N,N1)*GAMMA/RO(LMAX,N,N1)
A1=SQRT(AS)
IF (IEXTRA.EQ.2) GO TO 20
Q=SQRT (U(LMAX,N,N1)*U(LMAX,N,N1)+V(LMAX,N,N1)*
S V(LMAX,N,N1))
IF(Q/A1.LT.1.0) GO TO 20
10 U(LMAX,N,N3)=U(L1,N,N3)+FLOAT(IE1)*(U(L1,N,N3)-
S U(L2,N,N3))
V(LMAX,N,N3)=V(L1,N,N3)+FLOAT(IE1)*(V(L1,N,N3)-
S V(L2,N,N3))
P(LMAX,N,N3)=P(L1,N,N3)+FLOAT(IE1)*(P(L1,N,N3)-
S P(L2,N,N3))
RO(LMAX,N,N3)=RO(L1,N,N3)+FLOAT(IE1)*(RO(L1,N,N3)-
S RO(L2,N,N3))
GO TO 180
20 HAR=0.0

```

```

U1=U(LMAX,M,N1)
U2=U1
A2=A1
IF (ICHAR.NE.1) GO TO 30
U(LMAX,M,N3)=U1
RO(LMAX,M,N3)=RO(LMAX,M,N1)
A3=A1
C
C   CALCULATE THE PROPERTY INTERPOLATION POLYNOMIAL
C   COEFFICIENTS
C
30 BU=(U(LMAX,M,N1)-U(L1,M,N1))*DXR
BV=(V(LMAX,M,N1)-V(L1,M,N1))*DXR
BP=(P(LMAX,M,N1)-P(L1,M,N1))*DXR
BRO=(RO(LMAX,M,N1)-RO(L1,M,N1))*DXR
BAL=(AL-AL1)*DXR
BBE=(BE-BE1)*DXR
BDE=(DE-DE1)*DXR
CU=U(LMAX,M,N1)-BU*X3
CV=V(LMAX,M,N1)-BV*X3
CP=P(LMAX,M,N1)-BP*X3
CR0=RO(LMAX,M,N1)-BRO*X3
CAL=AL-BAL*X3
CBE=BE-BBE*X3
CDE=DE-BDE*X3
C
C   CALCULATE THE CROSS DERIVATIVE INTERPOLATING POLYNOMIAL
C   COEFFICIENTS
C
IF (M.EQ.3) GO TO 40
DU=(U(LMAX,M,N1)-U(LMAX,M-1,N1))*DYR
DV=(V(LMAX,M,N1)-V(LMAX,M-1,N1))*DYR
DP=(P(LMAX,M,N1)-P(LMAX,M-1,N1))*DYR
DRO=(RO(LMAX,M,N1)-RO(LMAX,M-1,N1))*DYR
DU1=(U(L1,M,N1)-U(L1,M-1,N1))*DYR
DV1=(V(L1,M,N1)-V(L1,M-1,N1))*DYR
DP1=(P(L1,M,N1)-P(L1,M-1,N1))*DYR
DRO1=(RO(L1,M,N1)-RO(L1,M-1,N1))*DYR
GO TO 60
40 DU=0.0
DV=(4.0*V(LMAX,2,N1)-V(LMAX,3,N1))*0.5*DYR
DP=0.0
DRO=0.0
DU1=0.0
DV1=(4.0*V(L1,2,N1)-V(L1,3,N1))*0.5*DYR
DP1=0.0
DRO1=0.0
60 BDU=(DU-DU1)*DXR
BDV=(DV-DV1)*DXR

```

```

BDRP= (DP- DP 1) *DXR
BDRO= (DRO- DR01) *DXR
CDU= DU- BDU*X3
CDV= DV- BDV*X3
CDP= DP- BDP*X3
CDRO= DRO- BDRO*X3

C
C      CALCULATE X1 AND X2
C

IF(ICHAR.EQ.1) GO TO 70
AS=P(LMAX,M,N3)*GAMMA/RO(LMAX,M,N3)
A3=SQRT(AS)
70 DO 80 IL=1,2
X1=X3- (U(LMAX,M,N3)+U1)*0.5*DT
X2=X3- (U(LMAX,M,N3)+A3+U2+A2)*0.5*DT

C
C      INTERPOLATE FOR THE PROPERTIES
C

U1=BU*X1+CU
U2=BU*X2+CU
P1=BP*X1+CP
P2=BP*X2+CP
RO2=BR0*X2+CRO
AS=P2*GAMMA/RO2
A2=SQRT(AS)
80 CONTINUE
V1=BV*X1+CV
P1=BP*X1+CP
RO1=BR0*X1+CRO
AL1=BAL*X1+CAL
BE1=BBE*X1+CBE
DE1=BDE*X1+CDE
UV1=U1*AL1+V1*BE1+DE1
AS=P1*GAMMA/RO1
A1=SQRT(AS)
V2=BV*X2+CV
AL2=BAL*X2+CAL
BE2=BBE*X2+CBE
DE2=BDE*X2+CDE
UV2=U2*AL2+V2*BE2+DE2

C
C      INTERPOLATE FOR THE CROSS DERIVATIVES
C
DV1=BDV*X1+CDV
DP1=BDP*X1+CDP
DR01=BDRO*X1+CDRO
DU2=BDU*X2+CDU
DV2=BDV*X2+CDV
DP2=BDP*X2+CDP
DR02=BDR0*X2+CDRO

C
C      CALCULATE THE PSI TERMS
C

```

C

```

IF(NDIM.EQ.0) GO TO 100
IF(M.EQ.1) GO TO 90
ATERM2=RO2*V2/(DY*FLOAT(N-1)/BE2)
GO TO 100
90 ATERM2=RO2*BE2*DV2
100 PSI11=-UV1*DV1-(DP1/RO1)*BE1
PSI41=-UV1*DP1+A1*A1*UV1*DRO1
PSI12=-UV2*DRO2-RO2*DV2*BE2-ATERM2-RO2*AL2*DU2
PSI122=-UV2*DU2-AL2*DP2/RO2
PSI42=-UV2*DP2+A2*A2*UV2*DRO2
IF(ICHAR.EQ.1) GO TO 160

```

C
C
C

CALCULATE THE CROSS DERIVATIVES AT THE SOLUTION POINT

```

IF(M.EQ.1) GO TO 110
IF(M.EQ.NE) GO TO 120
DU3=(U(LMAX,N+1,N3)-U(LMAX,N,N3))*DYR
DV3=(V(LMAX,N+1,N3)-V(LMAX,N,N3))*DYR
DP3=(P(LMAX,N+1,N3)-P(LMAX,N,N3))*DYR
DRO3=(RO(LMAX,N+1,N3)-RO(LMAX,N,N3))*DYR
GO TO 130
110 DU3=0.0
DV3=(4.0*V(LMAX,2,N3)-V(LMAX,3,N3))*0.5*DYR
DP3=0.0
DRO3=0.0
GO TO 130
120 MB=ME-1
DU3=(U(LMAX,ME,N3)-U(LMAX,MB,N3))*DYR
DV3=(V(LMAX,ME,N3)-V(LMAX,MB,N3))*DYR
DP3=(P(LMAX,ME,N3)-P(LMAX,MB,N3))*DYR
DRO3=(RO(LMAX,ME,N3)-RO(LMAX,MB,N3))*DYR

```

C
C
C

CALCULATE THE PSI TERMS AT THE SOLUTION POINT

```

130 IF(NDIM.EQ.0) GO TO 150
IF(M.EQ.1) GO TO 140
ATERM3=RO(LMAX,N,N3)*V(LMAX,N,N3)/(DY*FLOAT(N-1)/BE)
GO TO 150
140 ATERM3=RO(LMAX,1,N3)*BE*DV3
150 UV3=V(LMAX,N,N3)*BE+U(LMAX,N,N3)*AL+DE
PSI13=-UV3*DRO3-RO(LMAX,N,N3)*(DV3*BE+AL*DU3)-ATERM3
PSI123=-UV3*DU3-AL*DP3/RO(LMAX,N,N3)
PSI33=-UV3*DV3-(DP3/RO(LMAX,N,N3))*BE
PSI43=-UV3*DP3+A3*A3*UV3*DRO3
PSI31B=(PSI13+PSI33)*0.5+QVT(LMAX,N)
PSI41B=(PSI141+PSI43)*0.5+QPT(LMAX,N)
PSI12B=(PSI12+PSI13)*0.5+QROT(LMAX,N)
PSI22B=(PSI22+PSI23)*0.5+QUT(LMAX,N)
PSI42B=(PSI142+PSI43)*0.5+QPT(LMAX,N)

```

```

GO TO 170
160 PSI31B=PSI31+QVT(LMAX,B)
      PSI41B=PSI41+QPT(LMAX,B)
      PSI12B=PSI12+QBOT(LMAX,B)
      PSI22B=PSI22+QUT(LMAX,B)
      PSI42B=PSI42+QPT(LMAX,B)

C
C   SOLVE THE COMPATIBILITY EQUATIONS FOR U,V,P, AND RO
C
170 P(LMAX,N,N3)=PE(N)
      AB=0.5*(A2+A3)
      ROB=0.5*(RO2+RO(LMAX,N,N3))
      RO(LMAX,N,N3)=RO1+2.0*(P(LMAX,N,N3)-P1-DT*PSI41B)/
      S*(A3*A3+A1*A1)
      IF(RO(LMAX,N,N3).LE.0.0) RO(LMAX,N,N3)=ROLOW
      U(LMAX,N,N3)=U2+((PSI42B+ROB*AB*PSI22B+AB*AB*PSI12B)*DT
      S-(P(LMAX,N,N3)-P2))/(ROB*AB)
      V(LMAX,N,N3)=V1+DT*PSI31B
      V(LMAX,N,N3)=0.0

C
C   CHECK FOR INFLOW AND IF SO SET INFLOW BOUNDARY
C   CONDITIONS
C
      IF(U(LMAX,N,N3).GE.0.0) GO TO 180
      V(LMAX,N,N3)=0.0
      RO(LMAX,N,N3)=0.5*(RO(LMAX,1,N1)+RO(LMAX,MCB,N1))

180 CONTINUE
      V(LMAX,MCB,N3)=-U(LMAX,MCB,N3)*XXNY(LMAX)
      V(LMAX,MCB,N3)=0.0
      RETURN
      END

```

SUBROUTINE GMETY

```
C
C
C
C
C
```

THIS SUBROUTINE CALCULATES AND PLOTS THE PHYSICAL GRID

```
COMMON/D6/ U(50,20,2), V(50,20,2), P(50,20,2), RO(50,20,2)
COMMON/D7/ LMAX,NMAX,MCB,LCB,LBR,M,M3,L1,L2,M1,M2,
S MCB1,MCB2,LCB1,LCB2,LBR1,LBR2,GAMMA,DX,DY,DT,ICHAR,DXR,
S DXR,RSTAR,ASTARS,G,PC,LC,PLOW,BOLOS,RG
COMMON/D8/ XP1(50,20), YP1(50,20), KSI(2000), ETA(2000),
S PX(2000), PY(2000), XH(50), YH(20), MXNY(20), XE1,
S MDIM,PDS,AP
COMMON/D9/CAV,XMU,XLA,BKNU,XRO,QUT(50,20), QVT(50,20),
S QPT(50,20), QRROT(50,20), PE(20)
REAL MXNY,NYNY,LC,KSI
RX=1.0
RX=2.
XX=5.5
XE=7.0
KMAX=2000
K1=KMAX-1
YE=3.0
RY=RX
YY=1.0
GX=6.0
GY=6.0
GX=2.0
GY=2.0
GX=1.0
GY=1.0
DSY=YE/FLOAT(K1)
PY(1)=0.0
DS=XE/FLOAT(K1)
PX(1)=0.0
KSI(1)=0.0
ETA(1)=0.0
SLPY1=0.0
SI=0.0
I=0.0
SLOPE1=0.0
S=0.0
X=0.0
DO 2 L=2,KMAX
Y=Y+DSI
X=X+DS
SLOPE2=RX*GX/(COSH(GX*(X-XX))*COSH(GX*(X-XX)))
SLPY2=RY*GY/(COSH(GY*(Y-YY))*COSH(GY*(Y-YY)))
2
```

```

SI1=SQRT (1.+SLPY1*SLPY1)
SI2=SQRT (1.+SLPY2*SLPY2)
S1=SQRT (1.+SLOPE1*SLOPE1)
S2=SQRT (1.+SLOPE2*SLOPE2)
SY=SY+DSI*0.5*(SI1+SI2)
S=S+DS*0.5*(S1+S2)
ETA(L)=SY
PI(L)=Y
KSI(L)=S
PX(L)=X
SLPY1=SLPY2
2 CONTINUE
SLOPE1=SLOPE2
XE1=ETA(KMAX)
DETA=XE1/FLOAT(N1)
JY=1
SSY=-DETA
XE1=KSI(KMAX)
DKSI=XE1/FLOAT(L1)
J=1
SS=-DKSI
WRITE(6,100) KSI(KMAX),ETA(KMAX)
DO 3 L=1,LMAX
SS=SS+DKSI
DO 4 K=J,KMAX
IF ( ABS ( SS-KSI(K) ) .LT. 0.05) GO TO 5
4 CONTINUE
5 J=K
XW(L)=PX(K)
X=XW(L)
NXY(L)=RI*GX/(COSH(GX*(X-XX))*COSH(GX*(X-XX)))
WRITE(6,100) XW(L),SS,NXY(L)
3 CONTINUE
100 FORMAT (5X,4P14.5)
DO 33 M=1,MMAX
SSY=SSY+DETA
DO 44 KY=JY,KMAX
IF ( ABS ( SSY-ETA(KY) ) .LT. 0.05) GO TO 55
44 CONTINUE
55 JY=KY
XW(M)=PY(KY)
X=XW(M)
NYNX(M)=RY*GY/(COSH(GY*(Y-YY))*COSH(GY*(Y-YY)))
WRITE(6,100) XW(M),SSY,NYNX(M)
33 CONTINUE
DX=XE1/FLOAT(L1)
DY=YE1/FLOAT(N1)
DXR=1./DX
DYS=1./DY
DO 210 L=1,LMAX

```

```
DO 210 M=1,MMAX
XP1(L,M)=XW(L)
YP1(L,M)=YW(M)
210 CONTINUE
RETURN
END
```

VITA

Ameziane Harhad was born July 6, 1952 in Algiers, Algeria. He received his secondary education at Technical High School in Algiers, Algeria, and was graduated in June, 1971. He received his diploma of Engineering in June, 1976 from the University of Algiers (Algeria). He came to the United States and enrolled for graduate study in Mechanical Engineering at Louisiana State University in August, 1977. He completed the Master of science degree in December, 1978. He has also earned a master of science degree in Mathematics in August, 1981. Now he is a candidate for the degree of Doctor of Philosophy in the Department of Mechanical Engineering to be awarded at the convocation to be held on August 7, 1982.

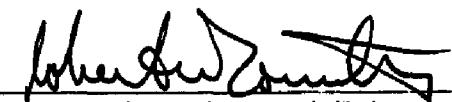
EXAMINATION AND THESIS REPORT

Candidate: Ameziane Harhad

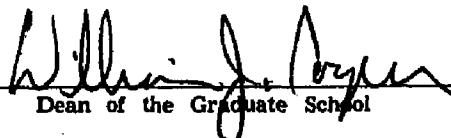
Major Field: Mechanical Engineering

Title of Thesis: Numerical Modeling of the Internal Ballistics of a Gas Gun

Approved:



Major Professor and Chairman



Dean of the Graduate School

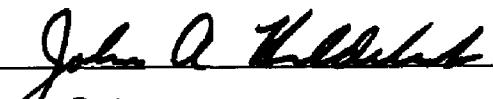
EXAMINING COMMITTEE:



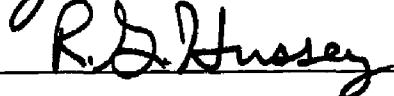
Dr. Miller



Dr. Ames



John A. Kellie



R.D. Hussey

Date of Examination:

May 28, 1982